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THESIS

**FREQUENCY DOMAIN STRUCTURAL
IDENTIFICATION**

by

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June 1996

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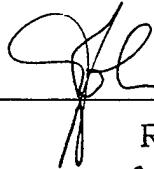
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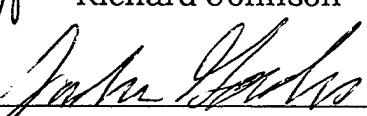
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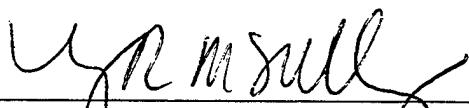


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ABSTRACT

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I. INTRODUCTION

The Finite Element (FE) method is a proven tool for modeling structural dynamic systems. As the complexity of the system increases the FE model may not accurately reflect the dynamic behavior of the system. To determine the extent to which the FE model accurately describes the physical system, a comparison of the dynamic response or modal parameters of the system as predicted by the FE model and the response or modal parameters of the physical system as determined by measurements of the dynamic response of the system can be made. Such a comparison can easily point out the differences in the dynamic behavior of the FE model and the physical system but fail to provide the necessary corrections to the FE model that will provide a more accurate FE model prediction of the dynamic behavior of the physical system.

Structural system identification refers to procedures used to identify finite element modeling errors using dynamic test data. Localization is the process of identifying those degrees of freedom (DOF) of the FE model whose impedance differs from that of the physical system. We refer to this set of DOF as the error set. Identification is the process of finding matrices ΔK , ΔM , and ΔC that are corrections to the FE model stiffness, mass, and damping matrices. When corrections are installed, the frequency response of the corrected FE model better predicts the frequency response of the experimental system, and hence the modal parameters of the corrected FE model better predicts those of the experimental system.

Structural system identification can be conducted using either modal or frequency domain methods. This thesis investigates a frequency domain method based on the structural synthesis transformation (SST) as outlined in Reference (1). When measured dynamic response data is available for each DOF of each coordinate of the FE model the identification is termed *spatially complete* and the SST can be used to exactly determine which elements of the FE model are in error. Additionally, the SST can be used to compute correction matrices ΔM , ΔK , and ΔC that can be used to correct the mass, stiffness, and damping matrices of the FE model. The dynamic response of the corrected

FE model exactly matches that of the physical system. In the more frequent case that dynamic response data is not available at every DOF of the FE model, the identification is termed *spatially incomplete*. In this case the SST provides a frequency dependent solution.

SST-based structural system identification uses the frequency response function (FRF) of the structure under consideration. The FRF is a quantitative description of the dynamic behavior of the structure. To obtain the FRF, known harmonic excitation forces are applied and the resulting harmonic response of the structure measured. The ratio of excitation forces to response at a coordinate evaluated at each frequency in a specified bandwidth defines the FRF.

II. THEORY

A. IMPEDANCE DESCRIPTION

The impedance model of a given physical system can be defined by the relationship of structural displacement with respect to an applied force,

$$\begin{Bmatrix} f_i \\ f_c \end{Bmatrix} = \begin{bmatrix} Z_{ii}^a & Z_{ic}^a \\ Z_{ic}^a & Z_{cc}^a \end{bmatrix} \begin{Bmatrix} x_i \\ x_c \end{Bmatrix} \quad (2.1a)$$

The harmonic force and response vectors are denoted by "f" and "x" respectively. These vectors and the impedance matrix, Z, are in general complex-valued and frequency dependent. Subscripts "i" and "c" denote non-error and error DOF respectively. The superscript "a" denotes quantities calculated from a FE (analytic) model. If the values were obtained from experimental test data, the superscript would be "x". Thus, for the experimental model the impedance relationship would be given by:

$$\begin{Bmatrix} f_i \\ f_c \end{Bmatrix} = \begin{bmatrix} Z_{ii}^x & Z_{ic}^x \\ Z_{ic}^x & Z_{cc}^x \end{bmatrix} \begin{Bmatrix} x_i \\ x_c \end{Bmatrix} \quad (2.1b)$$

In practice, the elements of the experimental impedance matrix of Equation (2.1b) are unmeasurable quantities. To see this we expand row j of Equation (2.1b) to obtain

$$f_j = z_{j1}^x x_1 + z_{j2}^x x_2 + \dots + z_{jn}^x x_n \quad (2.2)$$

To measure an element z_{j1}^x of Z^x , requires that we impose a unit displacement at coordinate x_i while physically holding all other other coordinates at zero displacements. This is not physically possible. Assuming, for purpose of definition, the availability of the experimental impedance matrix, the quantitative difference between the analytical and experimental systems, as a function of frequency, is described by the impedance error

matrix. It is defined by the difference between the analytical and experimental impedance matrices. The error impedance matrix relationship is defined as:

$$\begin{bmatrix} 0 & 0 \\ 0 & \Delta Z(\Omega) \end{bmatrix} = \begin{bmatrix} Z_{ii}^a & Z_{ic}^a \\ Z_{ci}^a & Z_{cc}^a \end{bmatrix} - \begin{bmatrix} Z_{ii}^x & Z_{ic}^x \\ Z_{ci}^x & Z_{cc}^x \end{bmatrix} \quad (2.3)$$

B. STRUCTURAL SYNTHESIS TRANSFORMATION

Since the experimental impedance matrix, Z^x , is in general unavailable, frequency domain structural synthesis is used to identify the impedance error matrix using FRF data exclusively. A structural synthesis transformation is constructed from ΔZ of Equation (2.3) which encompasses the FE model errors. This transformation is applied to the finite element model to produce an experimental system FRF.

The FRF relates structural response to applied excitation. Given a FE model with impedance matrix, Z^a , the FRF, H^a , is the matrix inverse of the impedance matrix Z^a of equation (2.1a). For a static system ($\Omega=0$) the FRF is simply the flexibility matrix (inverse stiffness matrix). Using the notation of Equation (2.1) we may partition H^a as follows:

$$\begin{Bmatrix} x_i \\ x_c \end{Bmatrix} = \begin{bmatrix} H_{ii}^a & H_{ic}^a \\ H_{ic}^a & H_{cc}^a \end{bmatrix} \begin{Bmatrix} f_i \\ f_c \end{Bmatrix} \quad (2.4)$$

Generally, the "c" response coordinates experience applied forces due to both error impedances and externally applied forces, whereas "i" response coordinates experience only externally applied forces, such that,

$$f_c = f_c^{ext} + f_c^{\Delta Z} \quad (2.5a)$$

and

$$f_i = f_i^{ext} \quad (2.5b)$$

Expanding Equation (2.4) and substituting the relations of Equation (2.5) yeilds the following relationships:

$$x_i = H_{ii}^a f_i^{ext} + H_{ic}^a f_c^{ext} + H_{ic}^a f_c^{\Delta Z} \quad (2.6a)$$

$$x_c = H_{ci}^a f_i^{ext} + H_{cc}^a f_c^{ext} + H_{cc}^a f_c^{\Delta Z} \quad (2.6b)$$

If we include a copy of Equation (2.6b), in expanded matrix notation, Equation (2.6) reflects the three harmonic excitation terms to be considered, i.e.,

$$\begin{Bmatrix} x_i \\ x_c \\ x_c \end{Bmatrix} = \begin{bmatrix} H_{ii}^a & H_{ic}^a & H_{ic}^a \\ H_{ci}^a & H_{cc}^a & H_{cc}^a \\ H_{ci}^a & H_{cc}^a & H_{cc}^a \end{bmatrix} \begin{Bmatrix} f_i^{ext} \\ f_c^{ext} \\ f_c^{\Delta Z} \end{Bmatrix} \quad (2.7)$$

Response coordinates "c" and "i", which are due to external forces, will hereafter be referred to as "e" coordinates, denoting their dependence on external force excitation. Consequently, the three excitation forces are condensed into two under the identity:

$$\{f_e\} = [[f_i^{ext}] \ [f_c^{ext}]]^T \quad (2.8a)$$

$$\{f_c\} = \{f_c^{\Delta Z}\} \quad (2.8b)$$

and Equation (2.7) reduces to

$$\begin{Bmatrix} x_e \\ x_c \end{Bmatrix} = \begin{bmatrix} H_{ee}^a & H_{ec}^a \\ H_{ce}^a & H_{cc}^a \end{bmatrix} \begin{Bmatrix} f_e \\ f_c \end{Bmatrix} \quad (2.9)$$

Equation (2.3) shows that the impedance error is defined as the difference between the analytic and experimental impedance models. Hence, a transformation is required which uses the FRF relationship of Equation (2.9) to generate a similar relationship for the

experimental system. The impedance error ΔZ provides the basis by which this transformation is developed.

The impedance error matrix, $\Delta Z(\Omega)$, is a function of frequency and satisfies:

$$\{f_c\} = -[\Delta Z(\Omega)]\{x_c\} \quad (2.10)$$

where

$$[\Delta Z(\Omega)] = [[\Delta K] + j\Omega[\Delta C] - \Omega^2[\Delta M]] \quad (2.11)$$

for Ω the forcing frequency, $j = \sqrt{-1}$ and ΔK , ΔC , and ΔM stiffness, damping and mass error matrices comparable to those of the finite element formulation. The minus sign in Equation (2.10) reflects that the reaction forces imposed by impedance errors on the baseline model are being considered. Substituting the relationship

$$\begin{Bmatrix} f_e \\ f_c \end{Bmatrix} = \begin{bmatrix} I & 0 \\ 0 & -\Delta Z_c \end{bmatrix} \begin{Bmatrix} f_e \\ x_c \end{Bmatrix} \quad (2.12)$$

into Equation (2.9) yields:

$$\begin{Bmatrix} x_e \\ x_c \end{Bmatrix}^* = \begin{bmatrix} H_{ee}^a & H_{ec}^a \\ H_{ce}^a & H_{cc}^a \end{bmatrix} \begin{bmatrix} I & 0 \\ 0 & -\Delta Z \end{bmatrix} \begin{Bmatrix} f_e \\ x_c \end{Bmatrix}^* \quad (2.13)$$

simplifying we get:

$$\begin{Bmatrix} x_e \\ x_c \end{Bmatrix}^* = \begin{bmatrix} H_{ee}^a & -H_{ec}^a \Delta Z \\ H_{ce}^a & -H_{cc}^a \Delta Z \end{bmatrix} \begin{Bmatrix} f_e \\ x_c \end{Bmatrix}^* \quad (2.14)$$

Expanding Equation (2.14) into two equations and using "*" to denote a synthesized modified response results in

$$x_c^* = H_{ce}^a f_e - H_{cc}^a \Delta Z x_c^* \quad (2.15a)$$

$$x_e^* = H_{ee}^a f_e - H_{ec}^a \Delta Z x_c^* \quad (2.15b)$$

Rearranging Equation (2.15b) produces

$$[I + H_{cc}^a \Delta Z] x_c^* = H_{ce}^a f_e \quad (2.16a)$$

Using the property of the frequency response function,

$$x_c = H_{ce}^a f_e \quad (2.16b)$$

$$[I + H_{cc}^a \Delta Z] x_c^* = x_c \quad (2.16c)$$

$$x_c^* = [I + H_{cc}^a \Delta Z]^{-1} x_c \quad (2.16d)$$

Introducing Equation (2.16) into Equation (2.15b) results in

$$x_c^* = H_{ee}^a f_e - H_{ec}^a \Delta Z [I + H_{cc}^a \Delta Z]^{-1} x_c \quad (2.17a)$$

$$x_c^* = H_{ee}^a f_e - H_{ec}^a \Delta Z [I + H_{cc}^a \Delta Z]^{-1} H_{ce}^a f_e \quad (2.17b)$$

Once again we recall the property of a FRF,

$$x_e^* = H_{ee}^a f_e \quad (2.18a)$$

Combining Equations (2.17b) and (2.18a) yields

$$H_{ee}^* = H_{ee}^a - H_{ec}^a \Delta Z [I + H_{cc}^a \Delta Z]^{-1} H_{ce}^a \quad (2.18b)$$

Noting that

$$[I + H_{cc}^a \Delta Z]^{-1} = [(\Delta Z^{-1} + H_{cc}^a) \Delta Z]^{-1} \quad (2.19a)$$

and applying the matrix property

$$([a][b])^{-1} = [b]^{-1}[a]^{-1} \quad (2.19b)$$

we get that

$$H_{ee}^* = H_{ee}^a - H_{ec}^a [\Delta Z^{-1} + H_{cc}^a]^{-1} H_{ce}^a \quad (2.20a)$$

Replacing the superscript "*", which denotes the structures's synthesized coupled response, with the superscript "x" to indicate the test system response we arrive at

$$H_{ee}^x = H_{ee}^a - H_{ec}^a [\Delta Z^{-1} + H_{cc}^a]^{-1} H_{ce}^a \quad (2.20b)$$

In full matrix notation we have

$$\begin{bmatrix} H_{ii}^x & H_{ic}^x \\ H_{ci}^x & H_{cc}^x \end{bmatrix} = \begin{bmatrix} H_{ii}^a & H_{ic}^a \\ H_{ci}^a & H_{cc}^a \end{bmatrix} - \begin{bmatrix} H_{ic}^a \\ H_{cc}^a \end{bmatrix} [\Delta Z^{-1} + H_{cc}^a]^{-1} \begin{bmatrix} H_{ic}^a \\ H_{cc}^a \end{bmatrix}^T \quad (2.20c)$$

Equation (2.20b) is the structural synthesis transformation equation. When the experimental system FRF, H^x , is available the SST can be used to identify a frequency dependent impedance error matrix $[\Delta Z(\Omega)]$. Additionally, using Equation (2.9), $[\Delta Z(\Omega)]$ can be decomposed into constituent stiffness, mass, and damping errors.

C. FREQUENCY DOMAIN LOCALIZATION

We may rewrite Equation (2.20b) as

$$\Delta H_{ee} = H_{ec}^a D^{-1} H_{ce}^a \quad (2.21)$$

where

$$\Delta H_{ee} = H_{ee}^a - H_{ee}^x \quad (2.22a)$$

and

$$D = [\Delta Z^{-1} + H_{ce}^a] \quad (2.22b)$$

We define the localization matrix L as

$$L = Z_{ee}^a \cdot \Delta H_{ee} \cdot Z_{ee}^a \quad (2.23)$$

using the expression of Equation (2.22a) in Equation (2.23) we can rewrite L as

$$L = Z_{ee}^a \cdot H_{ec}^a D^{-1} H_{ce}^a \cdot Z_{ee}^a \quad (2.24a)$$

Expanding the "e" coordinate set into error and non error coordinates we get

$$L = \begin{bmatrix} Z_{ii}^a & Z_{ic}^a \\ Z_{ci}^a & Z_{cc}^a \end{bmatrix} \begin{bmatrix} H_{ii}^a & H_{ic}^a \\ H_{ci}^a & H_{cc}^a \end{bmatrix} D^{-1} \begin{bmatrix} H_{ic}^a & H_{cc}^a \end{bmatrix} \begin{bmatrix} Z_{ii}^a & Z_{ic}^a \\ Z_{ci}^a & Z_{cc}^a \end{bmatrix} \quad (2.24b)$$

Noting the the frequency response matrix is the inverse of the impedance matrix, i.e.,

$$\begin{bmatrix} Z_{ii}^a & Z_{ic}^a \\ Z_{ci}^a & Z_{cc}^a \end{bmatrix} \begin{bmatrix} H_{ii}^a & H_{ic}^a \\ H_{ci}^a & H_{cc}^a \end{bmatrix} = \begin{bmatrix} I & 0 \\ 0 & I \end{bmatrix} \quad (2.25a)$$

all mixed product coordinates of Equation (2.24b) must be zero and L simplifies to

$$L = \begin{bmatrix} 0 & 0 \\ 0 & D^{-1} \end{bmatrix} @ \Omega = \Omega_i \quad (2.25b)$$

D. ERROR IMPEDANCE

We can solve Equation (2.20b) for the impedance error $[\Delta Z]$. From Equation (2.25b) terms of Equation (2.20b) associated with non error coordinates may be assumed to be zero. We get the following form of the impedance error matrix

$$[\Delta Z] = \left(\left[\tilde{H}_{cc}^x \right] - \left[H_{cc}^a \right] \right)^{-1} @ \Omega = \Omega_i \quad (2.26a)$$

where

$$\left[\tilde{H}_{cc}^x \right] = \left(\left[H_{cc}^a \right]^{-1} [\Delta H] \left[H_{cc}^a \right]^{-1} \right)^{-1} \quad (2.26b)$$

Let $\Xi = \{\Omega_1, \Omega_2, \dots, \Omega_n\}$ be a set of frequencies where $\Omega_1 < \Omega_2 < \dots < \Omega_{n-1} < \Omega_n$. If for each $i = 2, 3, \dots, n-1$, we apply Equation (2.10) at each of the frequencies Ω_{i-1}, Ω_i , and Ω_{i+1} and assemble the resulting three equations into a system of three equations in three unknowns, we get the matrix equation

$$\begin{bmatrix} \Delta Z_c(\Omega_{i-1}) \\ \Delta Z_c(\Omega_i) \\ \Delta Z_c(\Omega_{i+1}) \end{bmatrix} = \begin{bmatrix} I & -\Omega_{i-1}^2 I & j\Omega_{i-1} I \\ I & -\Omega_i^2 I & j\Omega_i I \\ I & -\Omega_{i+1}^2 I & j\Omega_{i+1} I \end{bmatrix} \begin{bmatrix} \Delta K_c \\ \Delta M_c \\ \Delta C_c \end{bmatrix} \quad (2.27)$$

Equation (2.27) can be used to decompose the frequency dependent impedance error into constituent stiffness, mass, and damping error matrices, ΔK , ΔM , and ΔC at the frequencies Ω_i , $i=2, 3, \dots, n-1$. These constituents matrices are in general frequency

dependent. Denoting the solution of Equation (2.27) by $\begin{bmatrix} \Delta K_c(\Omega_i) \\ \Delta M_c(\Omega_i) \\ \Delta C_c(\Omega_i) \end{bmatrix}$ for $i=2,3,\dots,n-1$ we

obtain for each Ω_i $i=2,3,\dots,n-1$, error stiffness, mass, and damping matrices $\Delta K_c(\Omega_i)$, $\Delta M_c(\Omega_i)$, and $\Delta C_c(\Omega_i)$ such that

$$\Delta Z_c(\Omega_i) = \Delta K_c(\Omega_i) - \Omega_i^2 \Delta M_c(\Omega_i) + j\Omega_i \Delta C_c(\Omega_i) \quad (2.28)$$

The matrices $\Delta K_c(\Omega_i)$, $\Delta M_c(\Omega_i)$, and $\Delta C_c(\Omega_i)$ can be used to numerically correct the stiffness, mass, and damping matrices of the FE model at the frequencies Ω_i $i=2,3,\dots,n-1$ in that the FRF of the corrected FE model at the frequencies Ω_i approximates the experimental system FRF at Ω_i . To express this symbolically, if K^a , M^a , and C^a are the stiffness, mass, and damping matrices of the FE model and H^x is the FRF matrix of the experimental system at Ω_i

$$H^x(\Omega_i) \approx \left(K^a + \Delta K_c(\Omega_i) - \Omega_i^2 (M^a + \Delta M_c(\Omega_i)) + j\Omega_i (C^a + \Delta C_c(\Omega_i)) \right)^{-1} \quad (2.29)$$

where the "c" subscripted matrices are added at the corresponding error set coordinates of the "a" superscripted matrices.

For a general set of frequencies, $\Xi = \{\Omega_1, \Omega_2, \dots, \Omega_n\}$, we can form the system of n equations in three unknowns given by:

$$\begin{bmatrix} \Delta Z_c(\Omega_1) \\ \vdots \\ \Delta Z_c(\Omega_i) \\ \vdots \\ \Delta Z_c(\Omega_n) \end{bmatrix} = \begin{bmatrix} I & -\Omega_1^2 I & j\Omega_1 I \\ \vdots & \vdots & \vdots \\ I & -\Omega_i^2 I & j\Omega_i I \\ \vdots & \vdots & \vdots \\ I & -\Omega_n^2 I & j\Omega_n I \end{bmatrix} \begin{bmatrix} \Delta K_c \\ \Delta M_c \\ \Delta C_c \end{bmatrix} \quad (2.30)$$

The solution $\begin{bmatrix} \Delta K_c \\ \Delta M_c \\ \Delta C_c \end{bmatrix} = \begin{bmatrix} \Delta K_c(\Xi) \\ \Delta M_c(\Xi) \\ \Delta C_c(\Xi) \end{bmatrix}$ of Equation (2.30) represents error stiffness,

mass, and damping matrices which best corrects the FE model in a least squares sense. Equations (2.27) and (2.30) are fundamental to all that follows.

III. SPATIALLY COMPLETE STRUCTURAL IDENTIFICATION

To illustrate the principles of SST based frequency domain structural identification we will use the FE model of a free-free beam. To simulate the experimental system we impose a 25% addition to the mass and stiffness of elements 3 and 4 of a 10 element FE model. Figure 3-1 shows the finite element model of the beam and the spatially complete experimental system that results from imposing the mass and stiffness additions at element 3 and 4 of the FE model. Table 3-1 shows the system frequencies of the analytic and experimental systems.

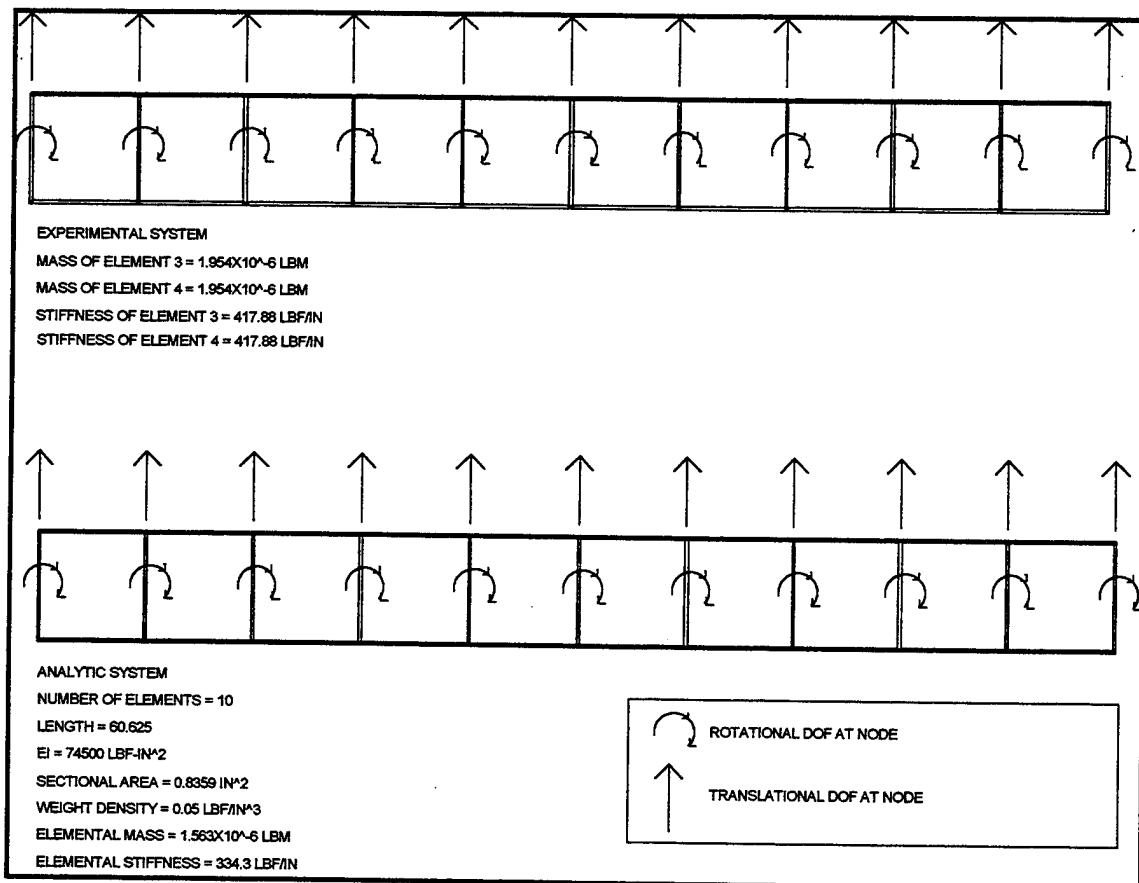


Figure 3- 1 Spatially complete Analytic and Experimental Systems.

MODE	ANALYTIC (Hz)	EXPERIMENTAL (Hz)
1	25.41	24.42
2	70.06	68.41
3	137.4	130.2
4	227.5	216.3
5	340.8	329.4
6	478.0	456.5
7	639.9	612.0
8	826.5	802.6
9	1026	969
10	1363	1323
11	1641	1573
12	1981	1920
13	2381	2283
14	2850	2754
15	3397	3269
16	4028	3912
17	4720	4605
18	5377	5156
19	6795	6791
20	6808	6801

Table 3-1 System Frequencies of spatially complete Analytic and Experimental systems.

We will denote by M^a , K^a , and C^a the mass stiffness and damping matrices of the FE model and by M^x , K^x , and C^x the mass stiffness and damping matrices of the experimental system. The impedance matrices of the analytic and simulated experimental systems are given by:

$$Z^a(\Omega) = K^a + j\Omega C^a - \Omega^2 M^a \quad (3.1a)$$

$$Z^x(\Omega) = K^x + j\Omega C^x - \Omega^2 M^x \quad (3.1b)$$

The FRF matrices of the analytic and simulated experimental systems are given by:

$$H^a(\Omega) = Z^a(\Omega)^{-1} \quad (3.2a)$$

$$H^x(\Omega) = Z^x(\Omega)^{-1} \quad (3.2b)$$

Figure 3-2 shows a comparison of the driving point FRF at DOF 1 of the analytic and experimental system.

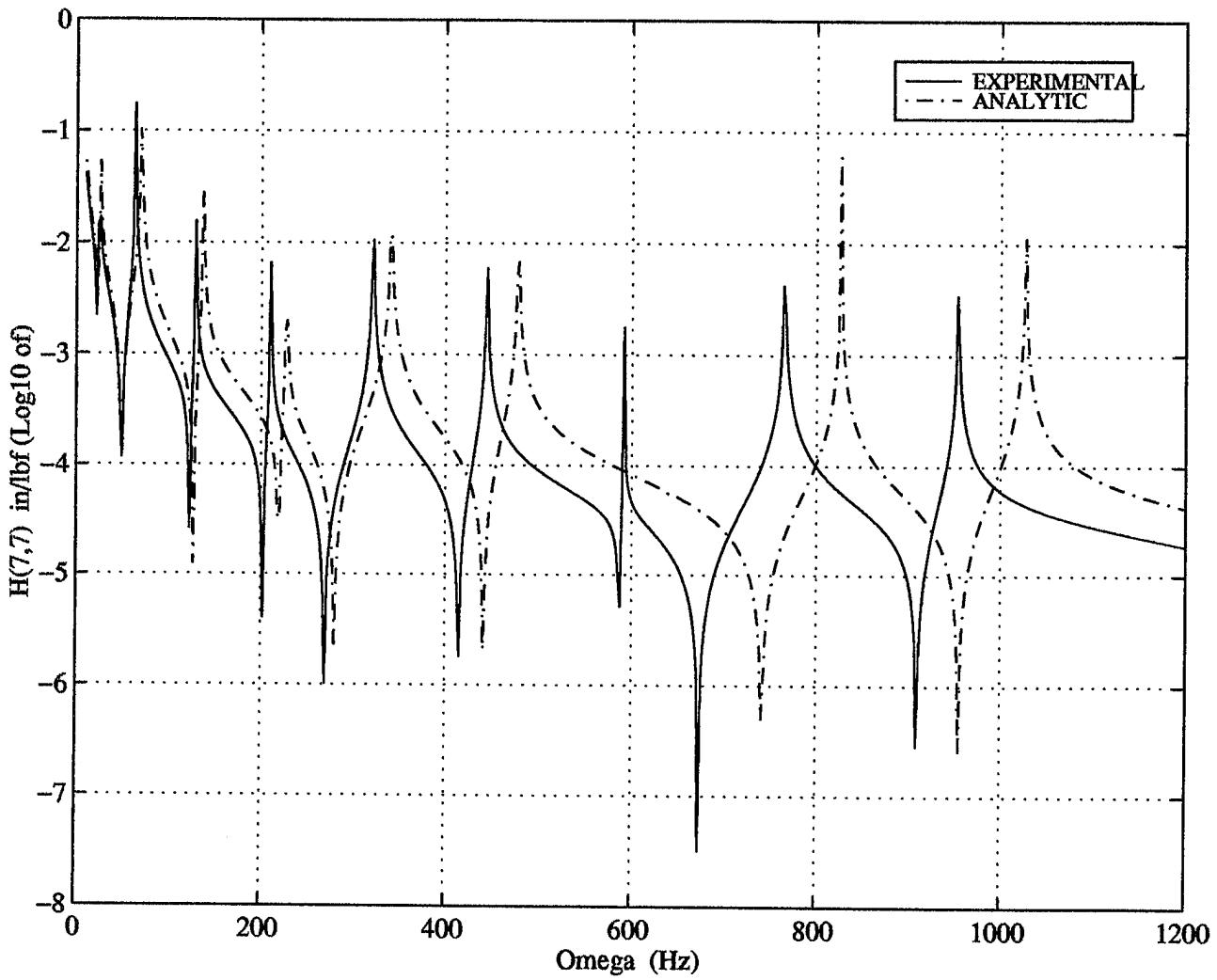


Figure 3- 2 Analytic FRF vs simulated Experimental FRF.

For a given frequency Ω_0 , using Equation (2.24a), we can form the localization matrix L. Plotting the diagonal elements of L versus the associated DOF we obtain the plot shown in Figure 3.3. For each frequency, Ω , in a given frequency range, we can compute the diagonal of L at Ω . Assembling the diagonals over the frequency range of our system into a rectangular matrix and performing a MATLAB mesh plot of the resulting rectangular matrix we obtain the surface plot shown in figure 3.4. Figure 3.4 shows the frequency dependency of the localization matrix diagonals. As our system is spatially complete Equation (2.25) forces all non error coordinates to be zero. From Figure 3-3 we can determine that the locations of the nonzero diagonal values are DOF 9, 10, 11, and 12. We denote by C_{err} the set of DOF for which $L(i,i)$ is non zero. For our beam system $C_{err} = \{9,10,11,12\}$. Figure 3.5 shows the frequency dependence of typical error and non error set diagonal elements of the localization matrix L.

Using the set, C_{err} , which results from the localization, we can perform a partitioning of the FRF matrix as described by Equation(2.4). We can now apply Equation (2.26) to compute ΔZ as a matrix function of frequency over the frequency range of our system. We then use Equation (2.27) to decompose ΔZ into its constituent components ΔK , ΔM , and ΔC . We get exact solutions of error stiffness, mass, and damping as shown in figures 3-7, 3-8 and 3-9. The MATLAB Routine SST.M of Appendix A can be used to accomplish the above steps.

For each Ω in the frequency range of our system we can form the sum:

$$Z_{corr}(\Omega) = Z_{cc}^a(\Omega) + \Delta Z(\Omega) \quad (3.3)$$

We refer to

$$H_{corr} = (Z_{corr})^{-1} \quad (3.4)$$

as the corrected FRF of the analytic model. H_{corr} is the FRF of the model that results from installing the corrections as identified by the Equation (2.26a). Figure 3-10 shows a comparison plot of the FRF of the corrected model and the FRF of the experimental system. Figure 3-10 clearly shows the exactness of the SST solution in the case of a

spatially complete system. The experimental and corrected model FRFs are identical to within plot resolution.

At each frequency, Ω , in the frequency range of our system we can form the sum

$$Z_{corr}^{const}(\Omega) = Z_{cc}^a(\Omega) + \Delta K(\Omega) + j\Omega\Delta C(\Omega) - \Omega^2\Delta M(\Omega) \quad (3.5)$$

We refer to:

$$H_{corr}^{const} = (Z_{corr}^{const})^{-1} \quad (3.6)$$

as the constituent corrected FRF. Figure 3-11 is a comparison plot of the constituent corrected FRF and experimental for our system. The offset between the two plots is exactly equal to the sampling frequency $\Delta\Omega$.

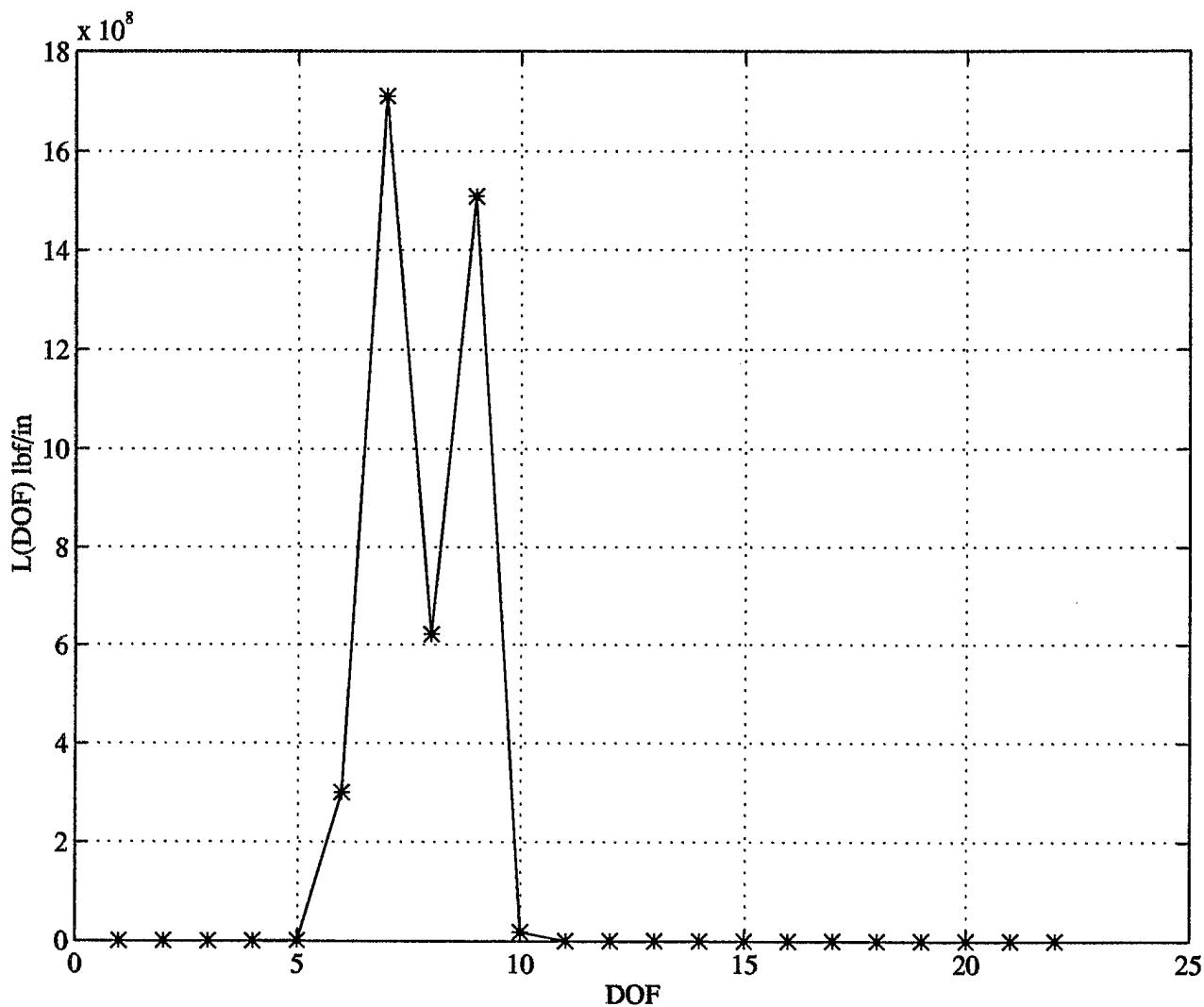


Figure 3- 3 Spatially complete localization matrix diagonal at $\Omega=352$ Hz.

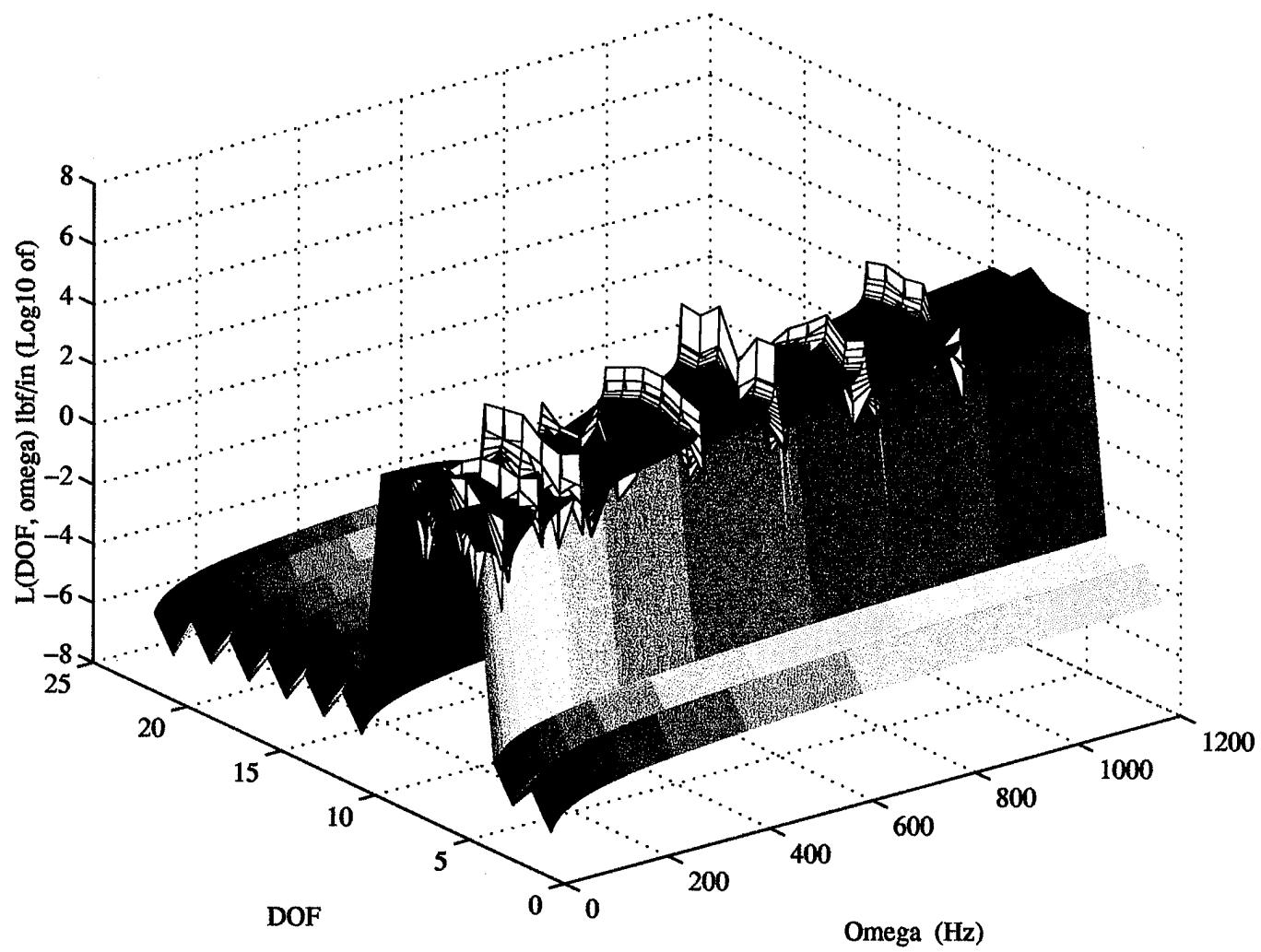


Figure 3- 4 Frequency dependence of spatially complete localization matrix diagonals.

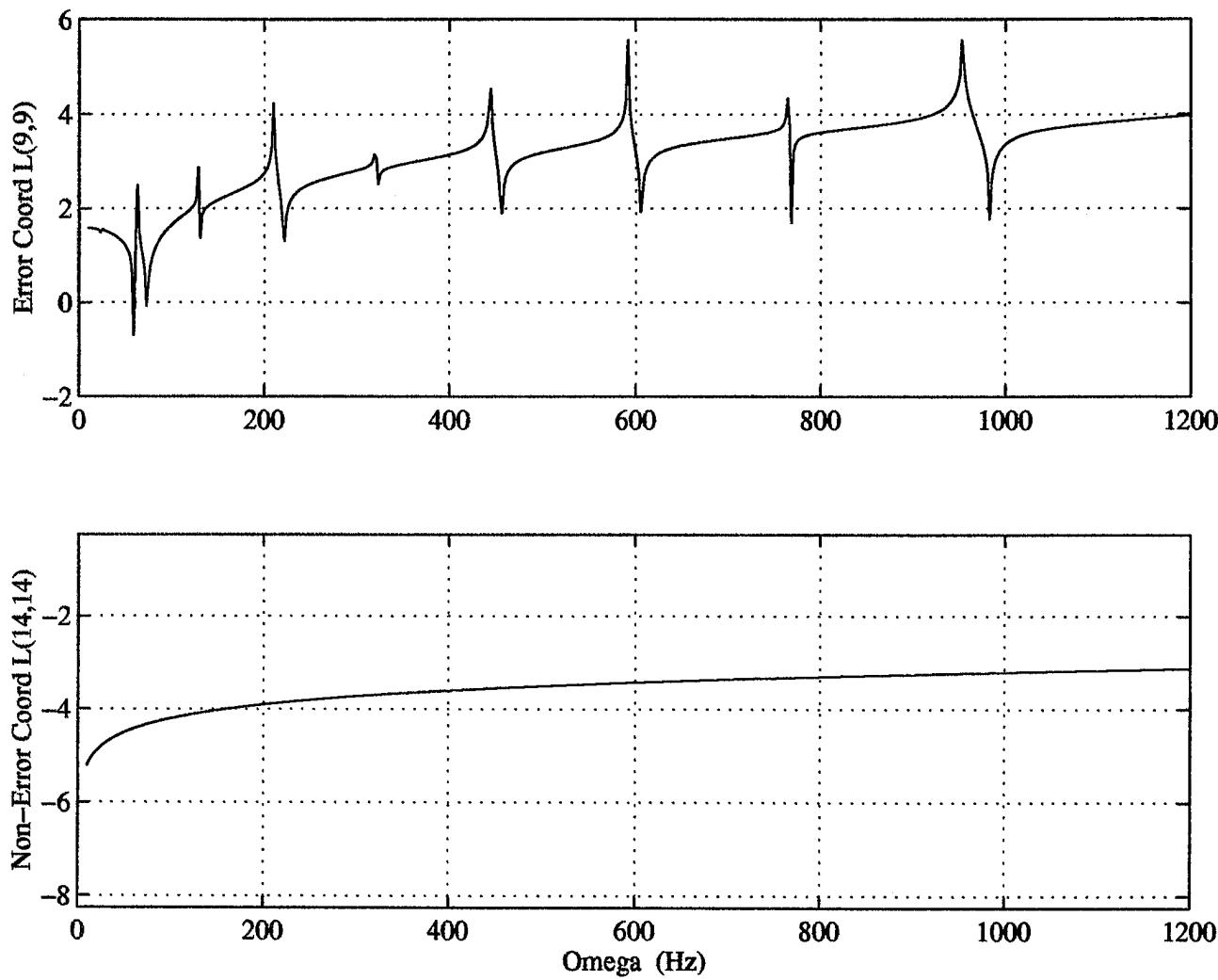


Figure 3- 5 Frequency dependence of spatially complete localization matrix error set DOF. Units are lbf/in (Log10 of).

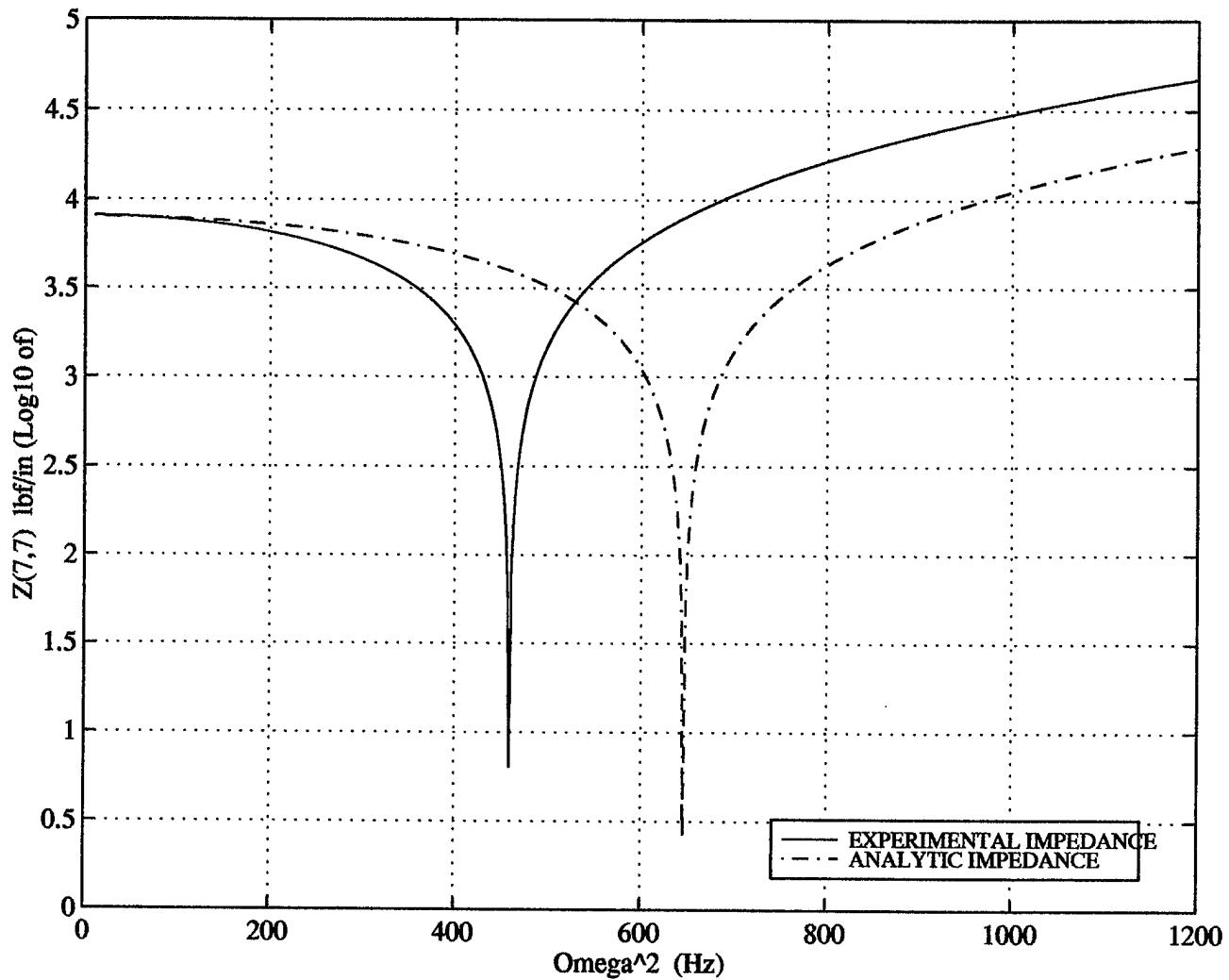


Figure 3- 6 Spatially complete Analytic Impedance vs Experimental impedance.

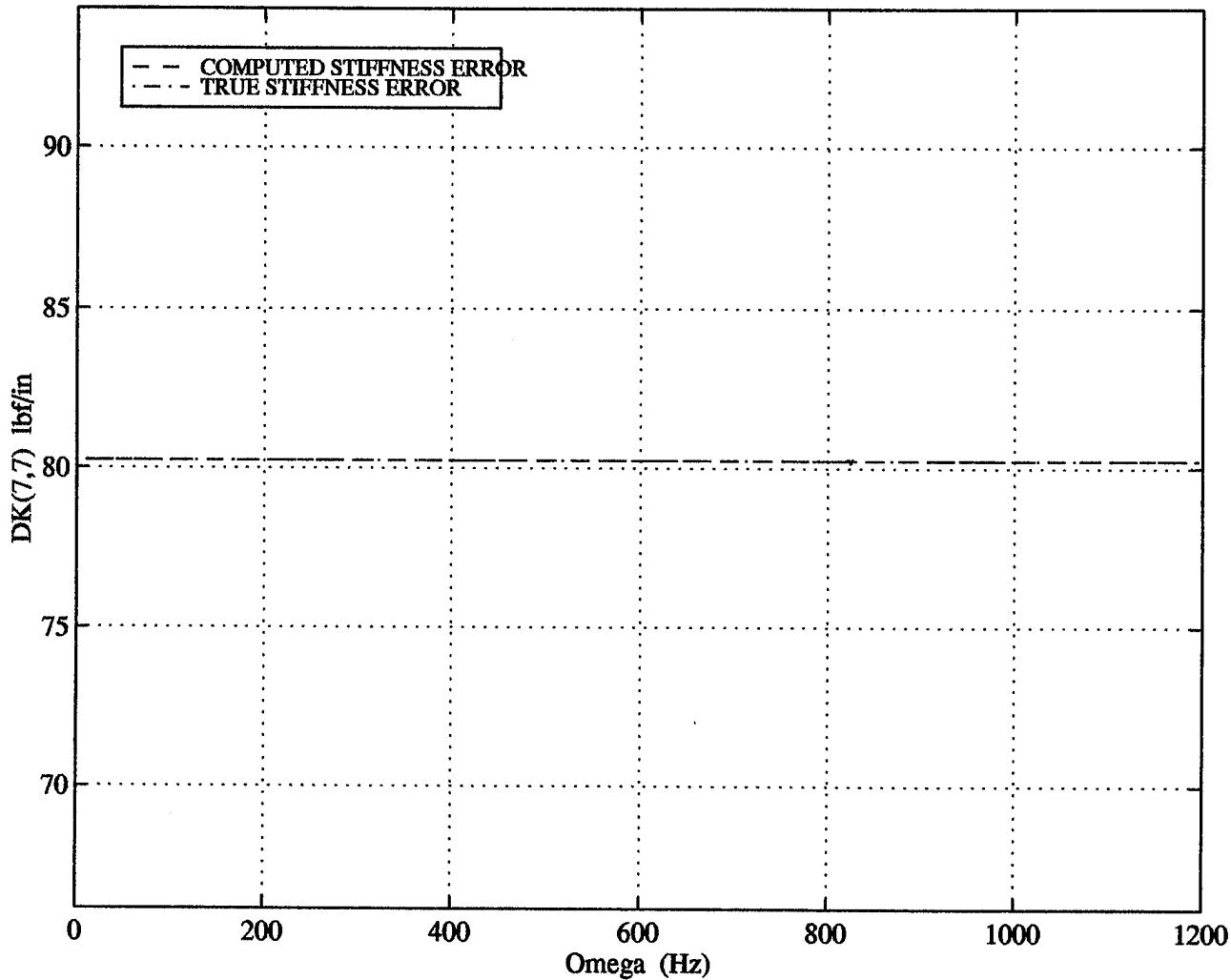


Figure 3- 7 Computed stiffness vs true stiffness for spatially complete beam. Plots are identical within plot resolution.

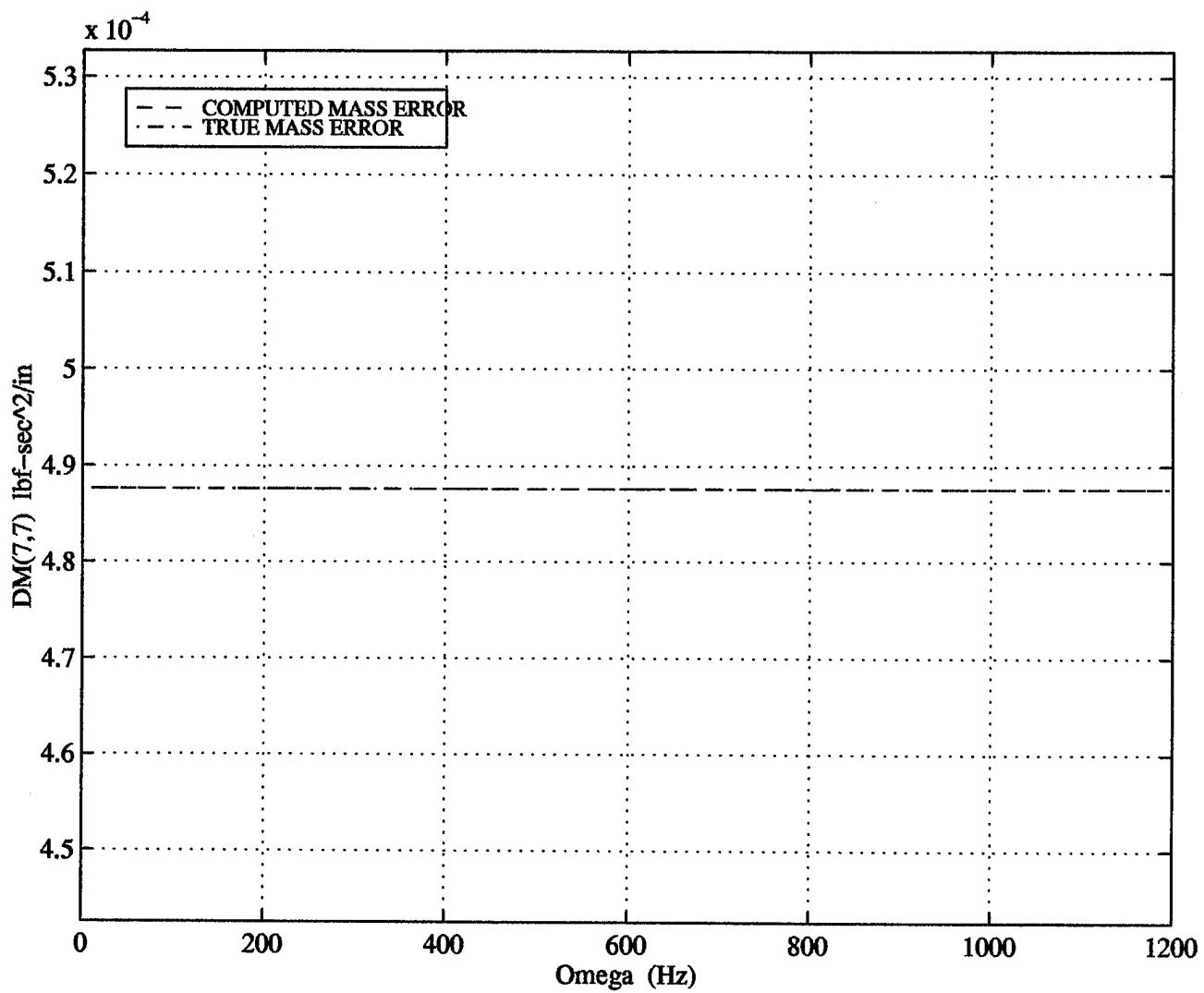


Figure 3- 8 Computed mass vs true mass for spatially complete beam. Plots are identical within plot resolution.

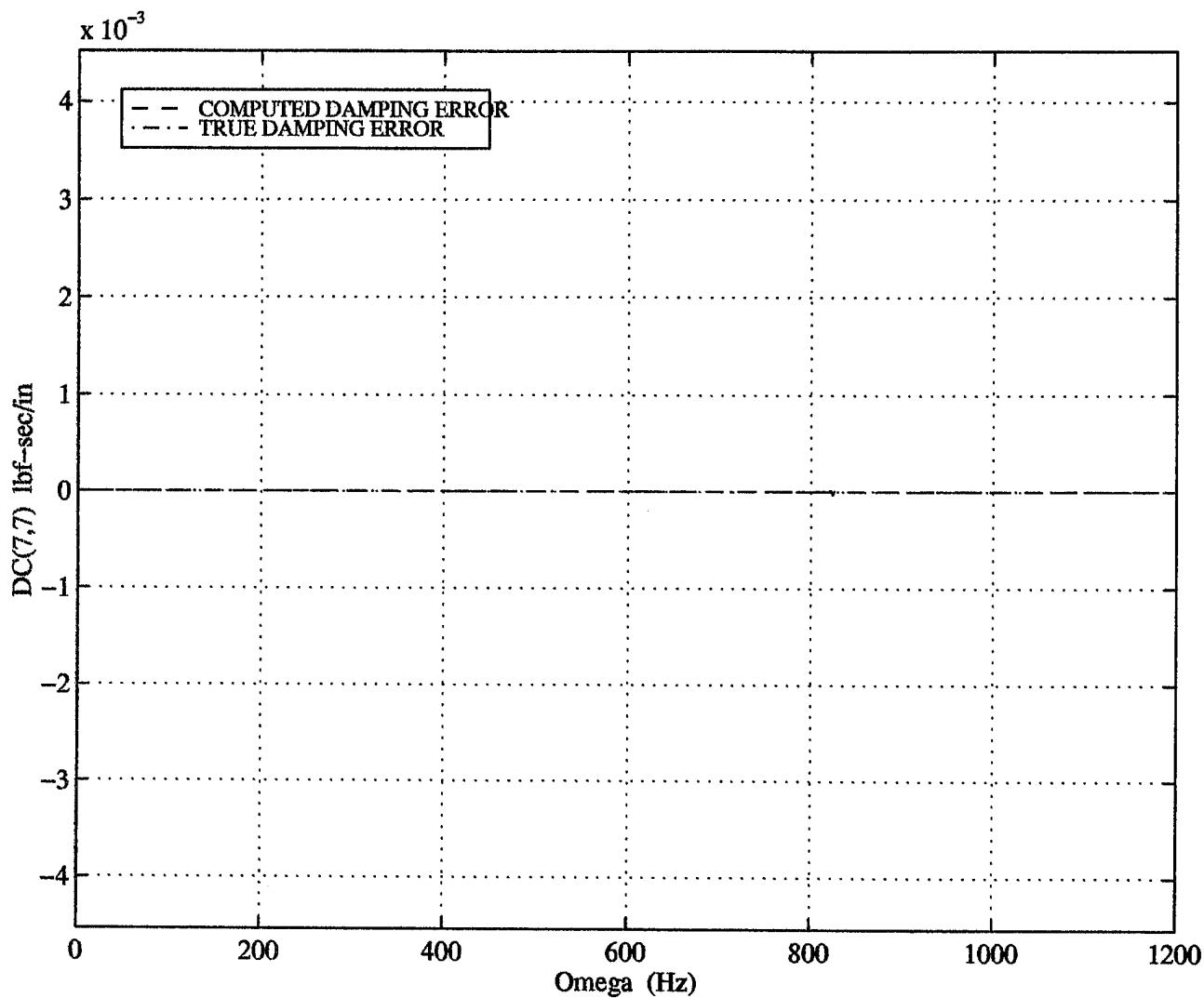


Figure 3- 9 Computed damping vs true damping for spatially complete beam. Plots are identical within plot resolution.

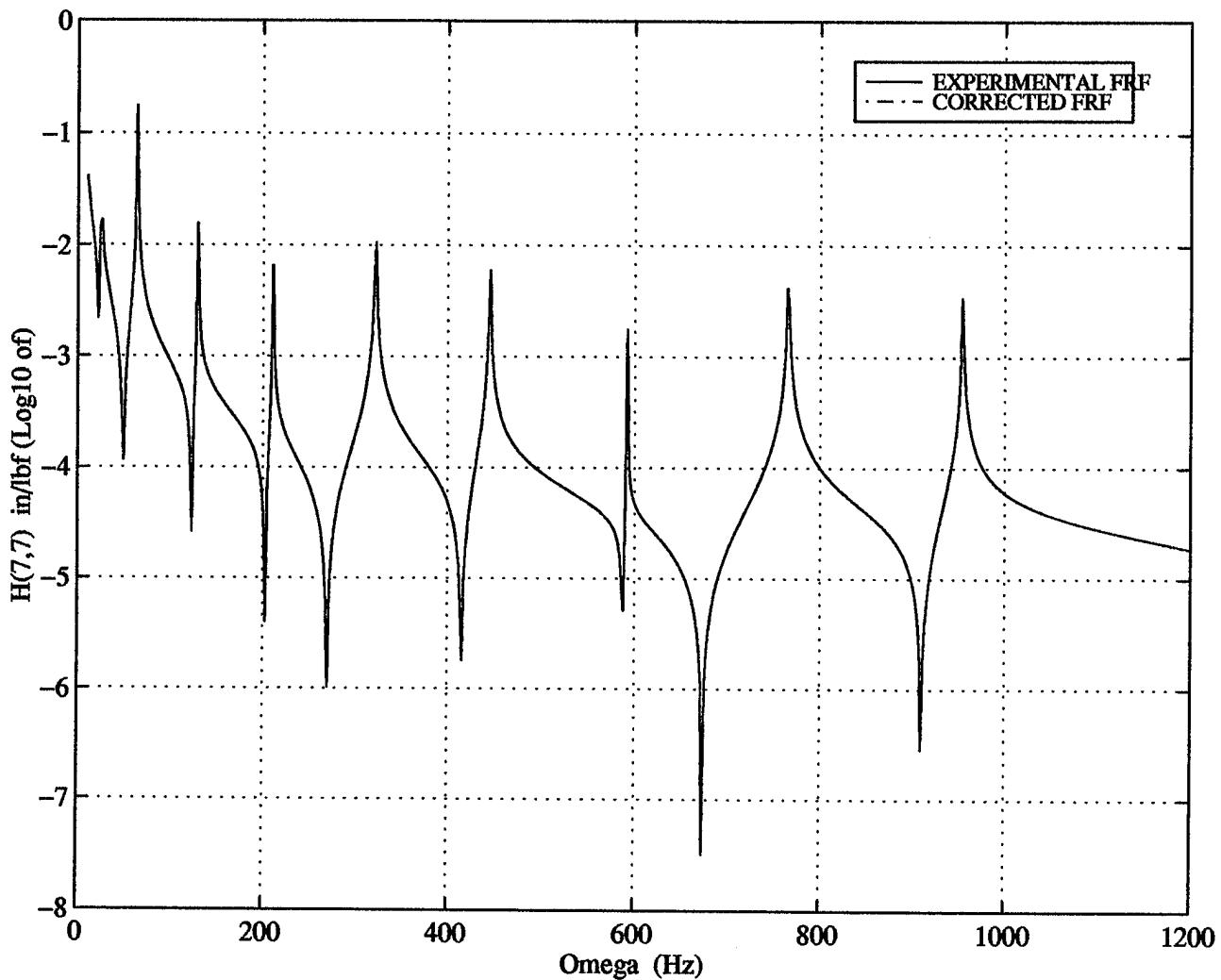


Figure 3- 10 Spatially complete Experimental FRF vs ΔZ corrected FRF. Plots are identical within plot resolution.

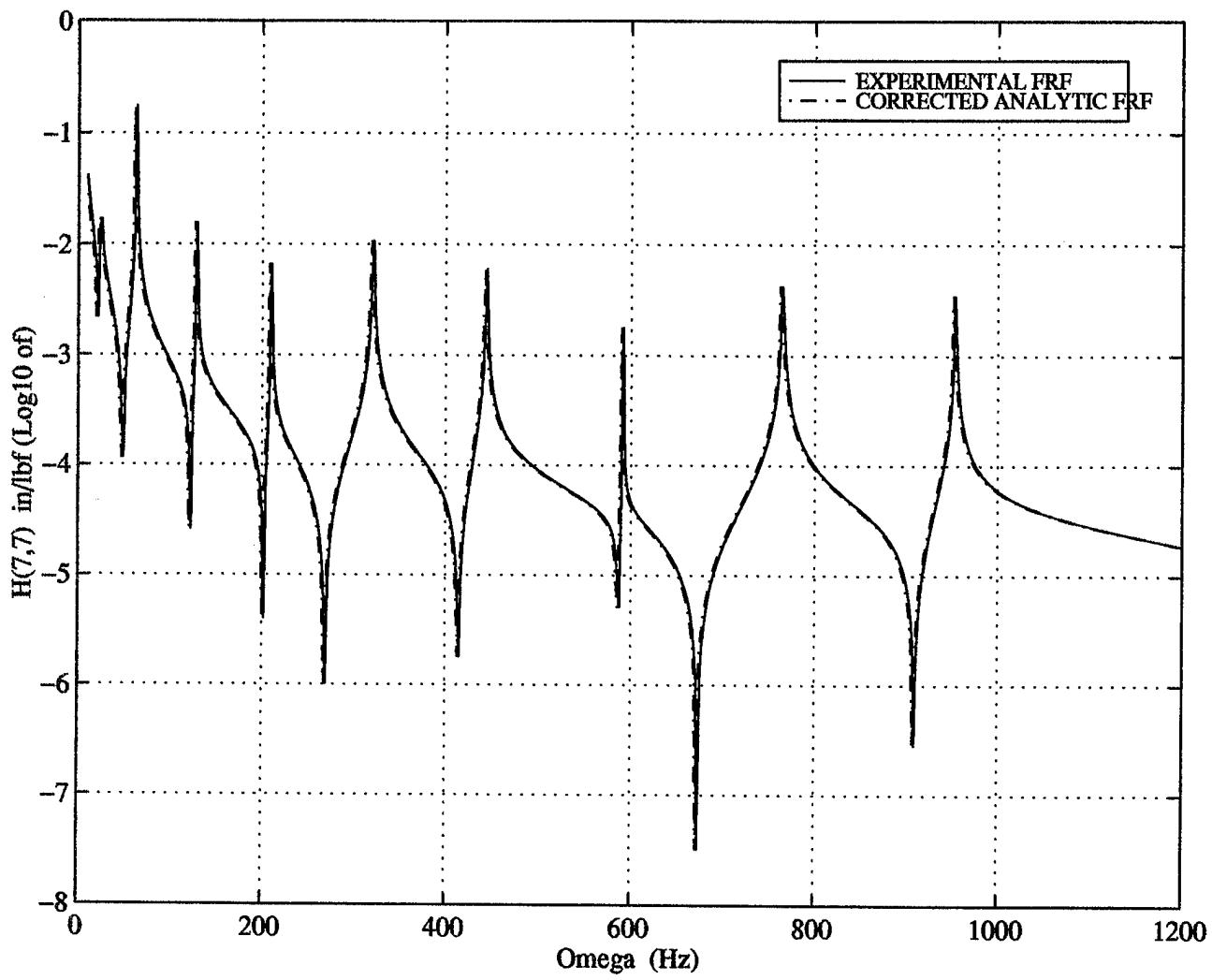


Figure 3- 11 Spatially complete experimental FRF vs ΔK , ΔM , and ΔC corrected FRF.

IV. SPATIALLY INCOMPLETE STRUCTURAL IDENTIFICATION

A. GENERAL DESCRIPTION

To illustrate SST based frequency domain structural identification when the physical system under consideration is spatially incomplete we will again use the FE model of a free-free beam. We simulate the experimental system by imposing a 25% addition to the mass and stiffness of elements 3 and 4 of a 10 element FE model. Figure 4-1 shows the FE modeled beam and the spatially incomplete system that results if FRF data is available only at the displacement DOF of the FE system.

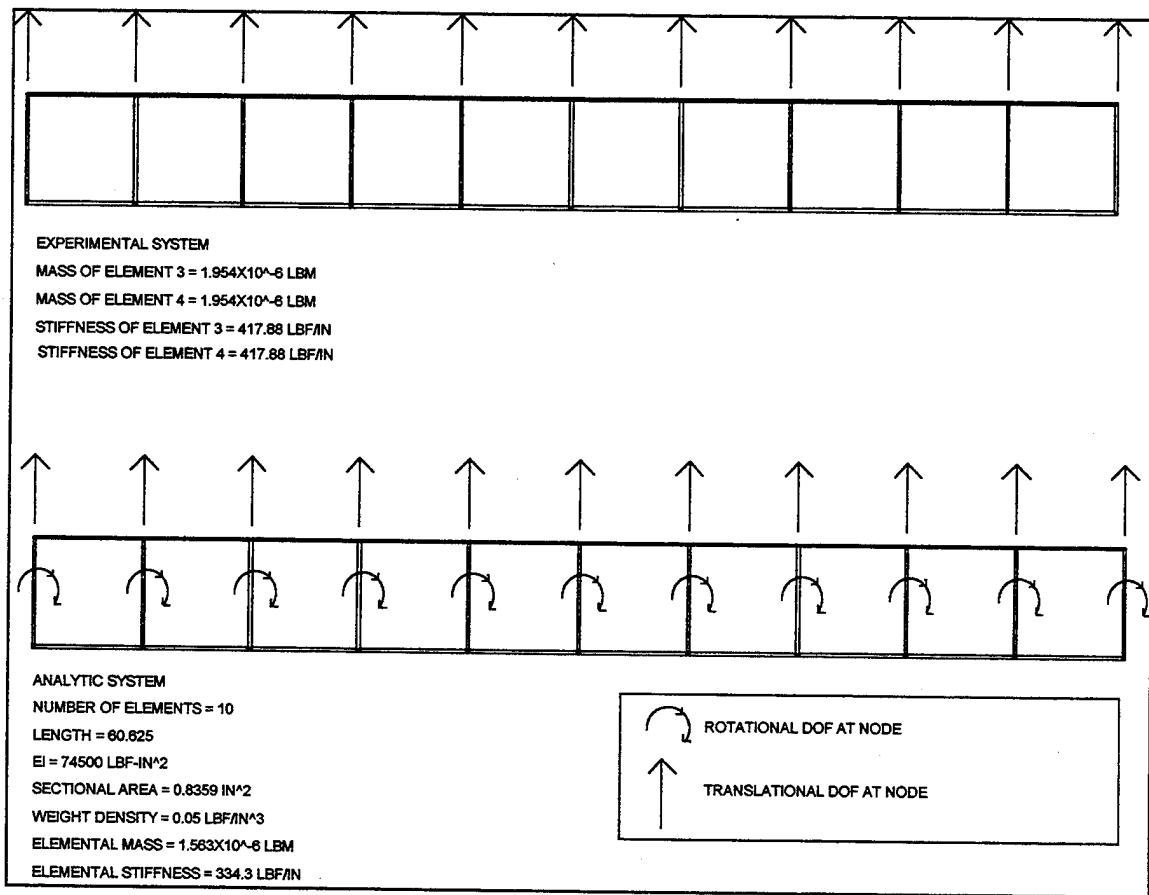


Figure 4- 1 Analytic and experimental spatially incomplete systems.

To obtain a simulated FRF for our spatially incomplete beam we denote by M^a , K^a , and C^a the mass stiffness and damping matrices of the FE model and by M^x , K^x , and C^x the mass stiffness and damping matrices of the experimental system. M^x , K^x , and C^x are obtained by imposing 15% mass and stiffness and a 15% mass additions to the elemental matrices of element 5 and 6 respectively of the FE model. The impedance matrix of the simulated spatially complete beam is given by

$$Z^x(\Omega) = K^x + j\Omega C^x - \Omega^2 M^x \quad (4-1)$$

The FRF matrix of the simulated spatially complete system is given by

$$H^x(\Omega) = Z^x(\Omega)^{-1}. \quad (4-2)$$

We introduce the terminology Analysis set and Omitted set where the Analysis set (A-set) is that set of DOF for which experimental FRF data is available and the Omitted set (O-set) is that set of DOF of the experimental system for which experimental FRF data is unavailable. For our simulated experimental system the A-set consist of the odd numbered translational DOF and the O-set consists of the even numbered rotational DOF, i.e.,

$$A-set = \{1,3,5,\dots,21\} \quad (4.3a)$$

$$O-set = \{2,4,6,\dots,22\} \quad (4.3b)$$

We obtain the simulated FRF of our spatially incomplete beam by physically extracting the rows and columns of the simulated spatially complete matrix, H^x , for which FRF data would be available. In our system these are the rotational DOF and all even numbered rows and columns are omitted from the simulated spatially complete FRF to obtain a spatially incomplete FRF which we will denote by $\overline{H^x}$.

For a fixed Ω_0 , $\overline{H^x}(\Omega_0)$ is a square matrix of size $(\text{length}(A\text{-set}))$ by $(\text{length}(A\text{-set}))$. For our FE model as currently defined, H^a , is of size (number of DOF) by (number of DOF) and is, as is true of most real world cases, of larger size than $\overline{H^x}$. In order to employ the structural synthesis transformation we need to reduce the size of H^a to that of

$\overline{H^a}$. To this end we will consider two reduction methods, FRF matrix extraction [Ref. 1], and the Improved Reduced System as given in [Ref. 2].

B. EXTRACTION REDUCTION METHOD

In order to reduce H^a by the extraction method we simply extract from the full order H those rows and column which correspond to A-set coordinates. Partitioning the impedance and full FRF matrices of our analytical system

$$Z = \begin{bmatrix} Z_{aa}^a & Z_{ao}^a \\ Z_{oa}^a & Z_{oo}^a \end{bmatrix} \quad (4.4a)$$

$$H = \begin{bmatrix} H_{aa}^a & H_{ao}^a \\ H_{oa}^a & H_{oo}^a \end{bmatrix} \quad (4.4b)$$

and using the identity

$$ZH = \begin{bmatrix} Z_{aa}^a & Z_{ao}^a \\ Z_{oa}^a & Z_{oo}^a \end{bmatrix} \begin{bmatrix} H_{aa}^a & H_{ao}^a \\ H_{oa}^a & H_{oo}^a \end{bmatrix} = \begin{bmatrix} I & 0 \\ 0 & I \end{bmatrix} \quad (4.5)$$

we obtain the relationship

$$H_{aa}^a = (Z_{aa} - Z_{ao} Z_{oo}^{-1} Z_{oa})^{-1} \quad (4.6)$$

We denote the reduced Analytic FRF of Equation (4.6) by $\overline{H^a}$.

C. O-SET SYSTEM

Equation (4.6) relates the extracted reduced order FRF of the analytic model to the impedance of the full order order model. Taking advantage of the identity

$$[Z_{oo}]^{-1} = (\det[Z_{oo}])^{-1} \text{adj}[Z_{oo}] \quad (4.7)$$

and replacing Z_{oo} by $K_{oo} + j\Omega C_{oo} - j\Omega^2 M_{oo}$ (where $\det(\bullet)$ and $\text{adj}(\bullet)$ represent the determinant and adjoint respectively), we see that an element $\overline{H_{ij}^a}(\Omega)$ is large for those frequencies Ω_o where Ω_o is an eigenvalue of the O-set system, the O-set system being that FE model having stiffness, mass and damping matrices K_{oo} , M_{oo} , and C_{oo} respectively.

D. IMPROVED REDUCTION SYSTEM

To use the improved reduction method (IRS) we first partition Z^a using the A and O sets, then adjust Equation (2.1) to reflect this new coordinate system obtaining

$$\begin{Bmatrix} f_a \\ f_o \end{Bmatrix} = \begin{bmatrix} Z_{aa}^a & Z_{ao}^a \\ Z_{oa}^a & Z_{oo}^a \end{bmatrix} \begin{Bmatrix} x_a \\ x_o \end{Bmatrix} \quad (4.8)$$

Expanding Equation (4.8) into two equations yields

$$f_a = Z_{aa}^a x_a + Z_{ao}^a x_o \quad (4.9a)$$

$$f_o = Z_{oa}^a x_a + Z_{oo}^a x_o \quad (4.9b)$$

O-set coordinates are not associated with FRF data measurement locations on the physical structure, therefore the forcing function at O-set coordinates can be set to zero. Making these substitution into Equation (4.9b) and solving for the generalized structural response coordinates leads to:

$$x_o = -Z_{oo}^{-1} Z_{oa}^a x_a \quad (4.10a)$$

$$\begin{Bmatrix} x_a \\ x_o \end{Bmatrix} = \begin{bmatrix} I \\ -Z_{oo}^{-1} Z_{oa}^a \end{bmatrix} \begin{Bmatrix} x_a \\ x_o \end{Bmatrix} \quad (4.10b)$$

Substituting these results into Equation (4.8) yields,

$$\begin{Bmatrix} f_a \\ 0 \end{Bmatrix} = \begin{bmatrix} Z_{aa}^{aa} & Z_{aa}^o \\ Z_{oa}^a & Z_{oo}^a \end{bmatrix} \begin{bmatrix} I \\ -Z_{oo}^{-1} Z_{oa}^a \end{bmatrix} \begin{Bmatrix} x_a \\ x_o \end{Bmatrix} \quad (4.11)$$

hence

$$\{f_a\} = [Z_{aa} - Z_{ao} Z_{oo}^{-1} Z_{oa}] \{x_a\} \quad (4.12)$$

When $\Omega=0$ the Equation (4.10a) yields the static reduction relationship between omitted and retained coordinates and is given by

$$\{x_o\} = [-K_{oo}^{-1} K_{oa}] \{x_a\} \quad (4.13)$$

The IRS relationship is given by

$$\{x_o\} = [-K_{oo}^{-1}K_{oa} + TM_{stat}^{-1}K_{stat}] \{x_a\} \quad (4.14)$$

where

$$T = K_{oo}^{-1}M_{oa} - K_{oo}^{-1}M_{oo}K_{oo}^{-1}K_{oa} \quad (4.15)$$

and K_{stat} and M_{stat} are the statically reduced [Ref. 2, 3] stiffness and mass matrices.

Unlike the spatially complete case where we only had to consider two system (the analytic and the experimental systems), in the case of a spatially incomplete system there are actually five systems with which we must concern ourselves [Ref 4]: the analytic system, the experimental system, the reduced analytic system that results from conducting dynamic reduction on the mass and stiffness matrices of the analytic system, and the omitted systems of both the analytic and the experimental system. Table 4-1 shows the frequencies of each the first four of these systems.

Figure 4-2 shows a comparision of the analytic and experimental FRF of our spatially incomplete beam. We see that by using IRS reduction of the analytic system those modes above approximately 1000 Hz i.e., those modes associated with the reduced out rotations, are not present. Figure 4-3 shows a similar comparision where we have used extraction reduction and the higher modes are present. In all that follows we shall use IRS reduction of our analytical systems. Figure 4-4 shows the localization matrix diagonal at $\Omega=196.1$ Hz while Figure 4-5 shows the frequency dependency of the localization diagonals over the frequency range our our spatially incomplete beam. For a spatially incomplete system the determination of the locations of the error coordinates is not a clearly defined task. We shall not discuss the problem of actually determining the exact error set in the spatially incomplete case and will use our knowledge of the true location of the error coordinates to aid in our localization. As we shall make use of the concept of the size of the error set again, we will simply note that using a reduction method like IRS causes errors to be 'smeared' in the reduced analytic model. Table 4-2 shows the A-set and O-set of our simulated beam along with the locations of the true errors as well as the

computed C-set at $\Omega=196$ Hz. The computed C-set is the set of all DOF having a diagonal entry whose absolute value exceeds a given tolerance.

Mode	Anal (Hz)	Exp (Hz)	Reduced (Hz)	O set (Hz)
1	25.41	25.83	25.41	1244.28
2	70.07	70.85	70.07	1287.94
3	137.45	137.93	137.45	1417.14
4	227.55	227.37	227.55	1629.59
5	340.84	341.14	340.84	1928.6
6	478.0	477.97	478.0	2327.83
7	639.9	639.92	640.3	2853.66
8	826.5	826.24	829.7	3541
9	1026.8	1025.99	1029.74	4399.32
10	1363.62	1364.51		5280.86
11	1640.62	1640.71		5701.99
12	1981.19	1984.61		
13	2381.4	2380.32		
14	2850.01	2847.38		
15	3397.39	3397.74		
26	4027.82	4026.28		
17	4720.15	4738.64		
18	5376.83	5380.09		
19	6794.91	6787.09		
20	6807.52	6804.12		

Table 4-1 Analytic, Experimental, Reduced, and Omitted System Frequencies of Spatially Incomplete Beam.

A Set	1	3	5	7	9	11	13	15	17	19	21
O Set	2	4	6	8	10	12	14	16	18	20	22
True C Set	5	6	7	8	9	10					
Computed C Set	1	3	5	7	9	11	13				

Table 4-2 Analytic set, Omitted set, Computed C set, and True C set DOF for spatially incomplete beam.

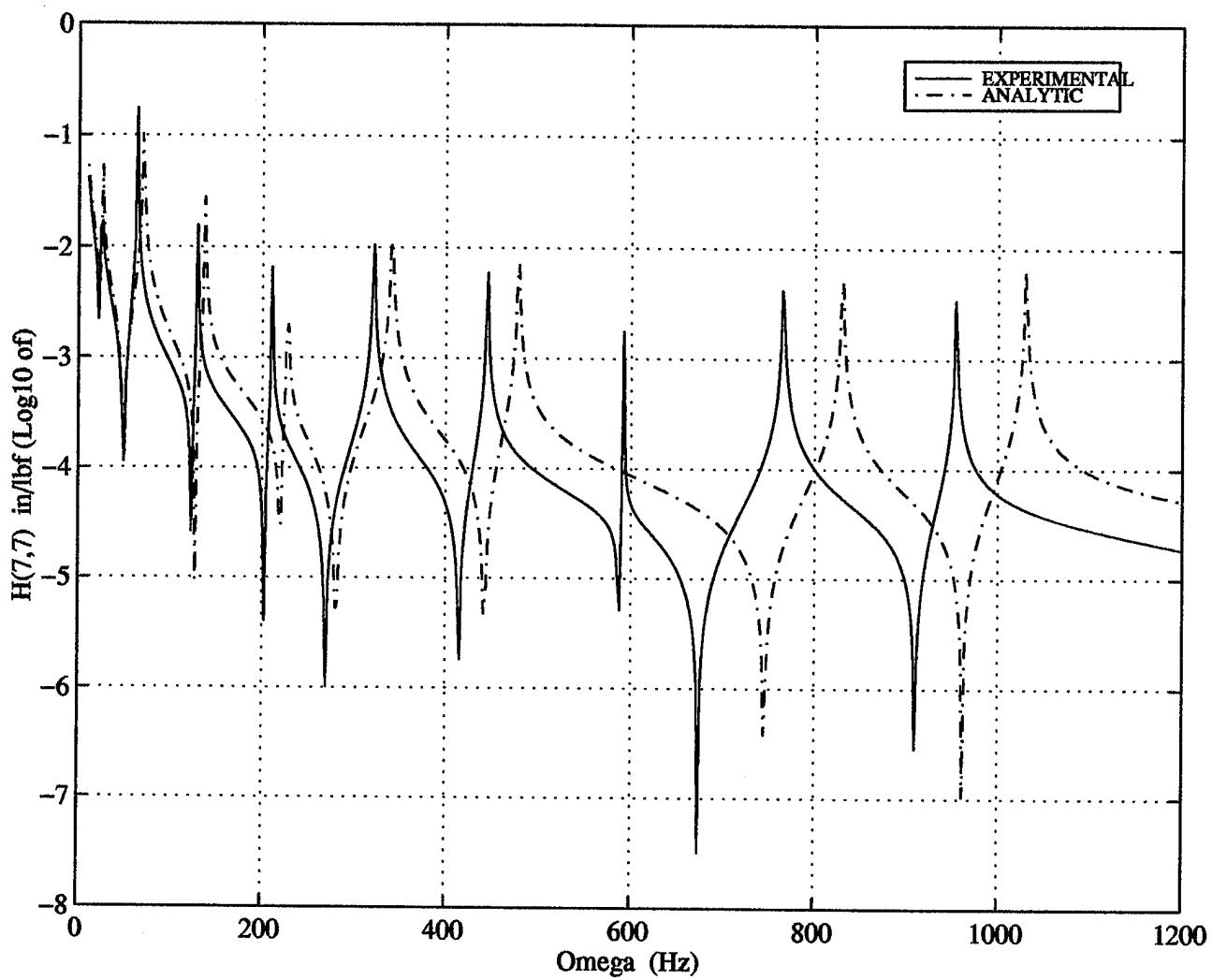


Figure 4- 2 Analytic FRF vs experimental FRF for spatially incomplete beam using IRS reduction.

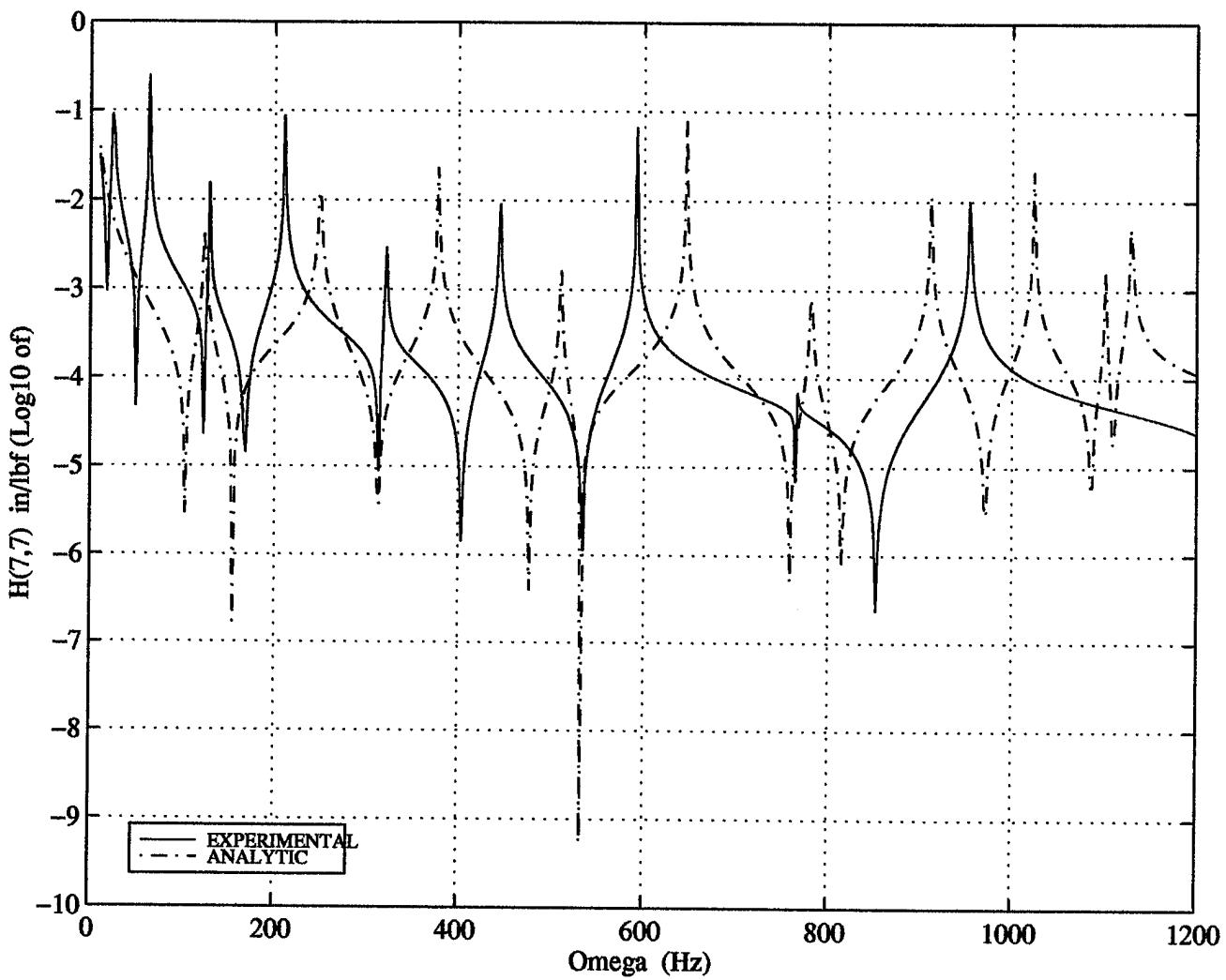


Figure 4- 3 Analytic FRF vs experimental FRF for spatially incomplete beam using extraction reduction.

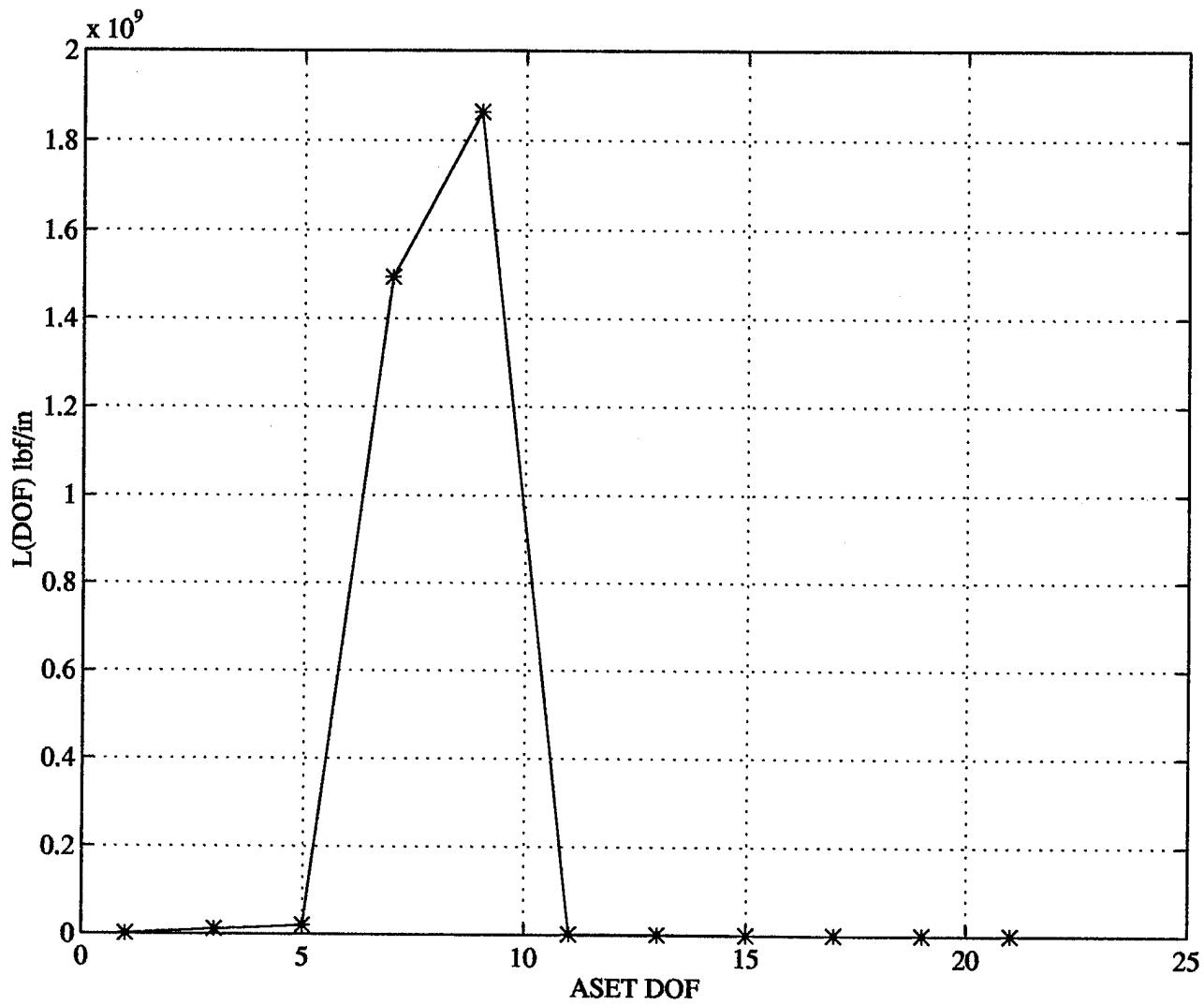


Figure 4- 4 Localization matrix diagonal at $\Omega=196.1$ Hz for spatially incomplete beam.

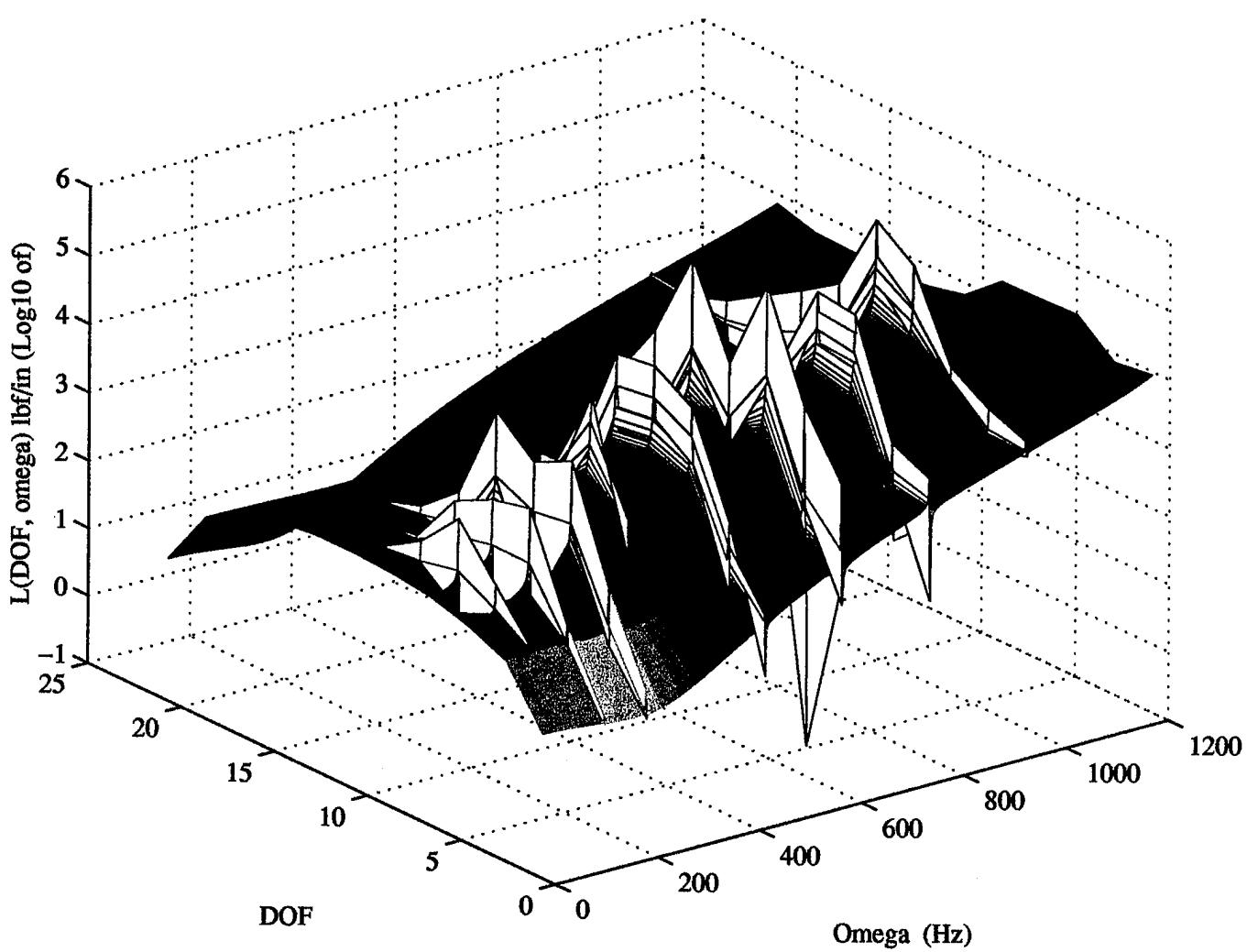


Figure 4- 5 Frequency dependence of localization matrix diagonals for spatially incomplete beam.

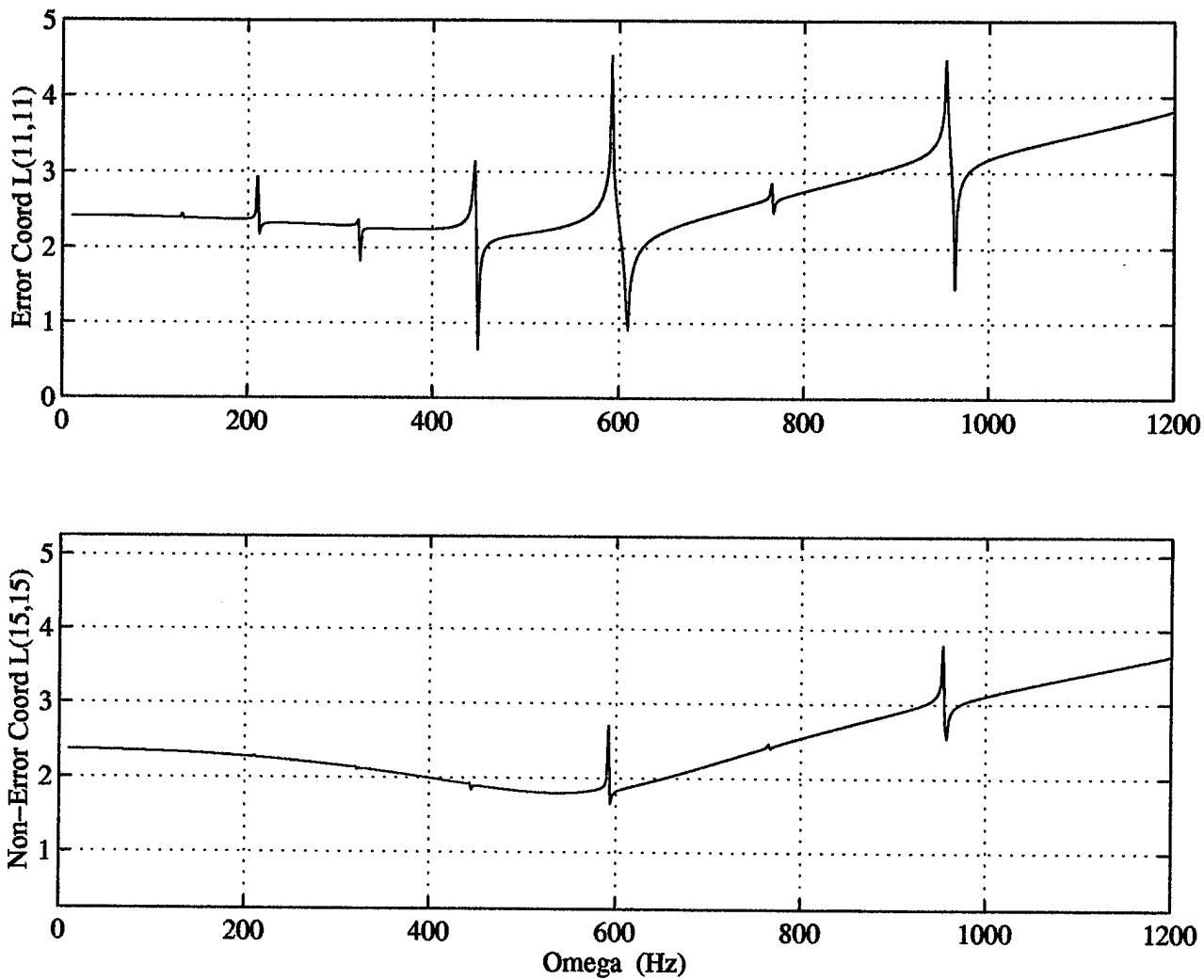


Figure 4- 6 Frequency dependence of error and non error set localization matrix diagonals for spatially incomplete beam. Units are lbf/in (Log10 of).

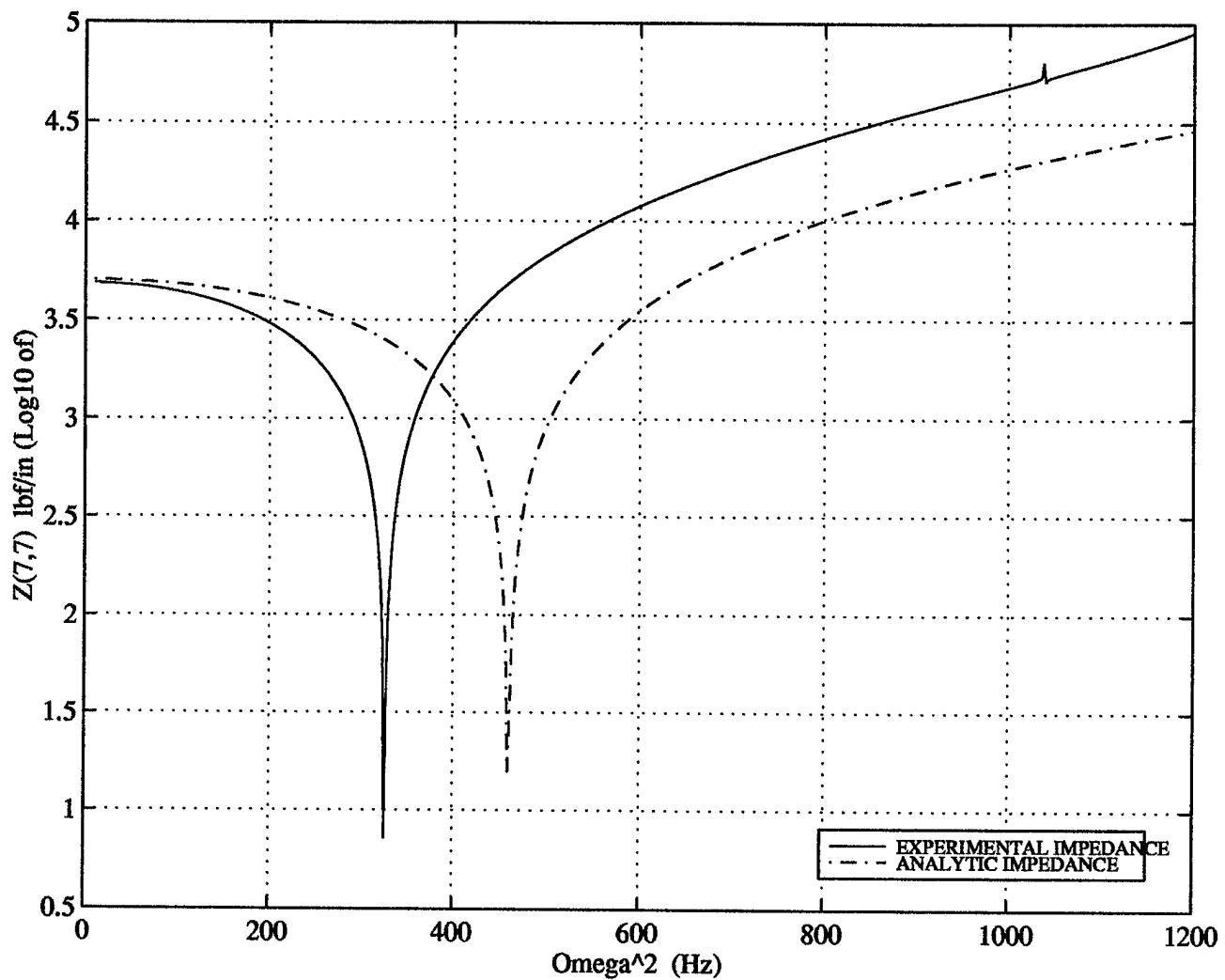


Figure 4- 7 Analytic vs experimental impedance for spatially incomplete beam.

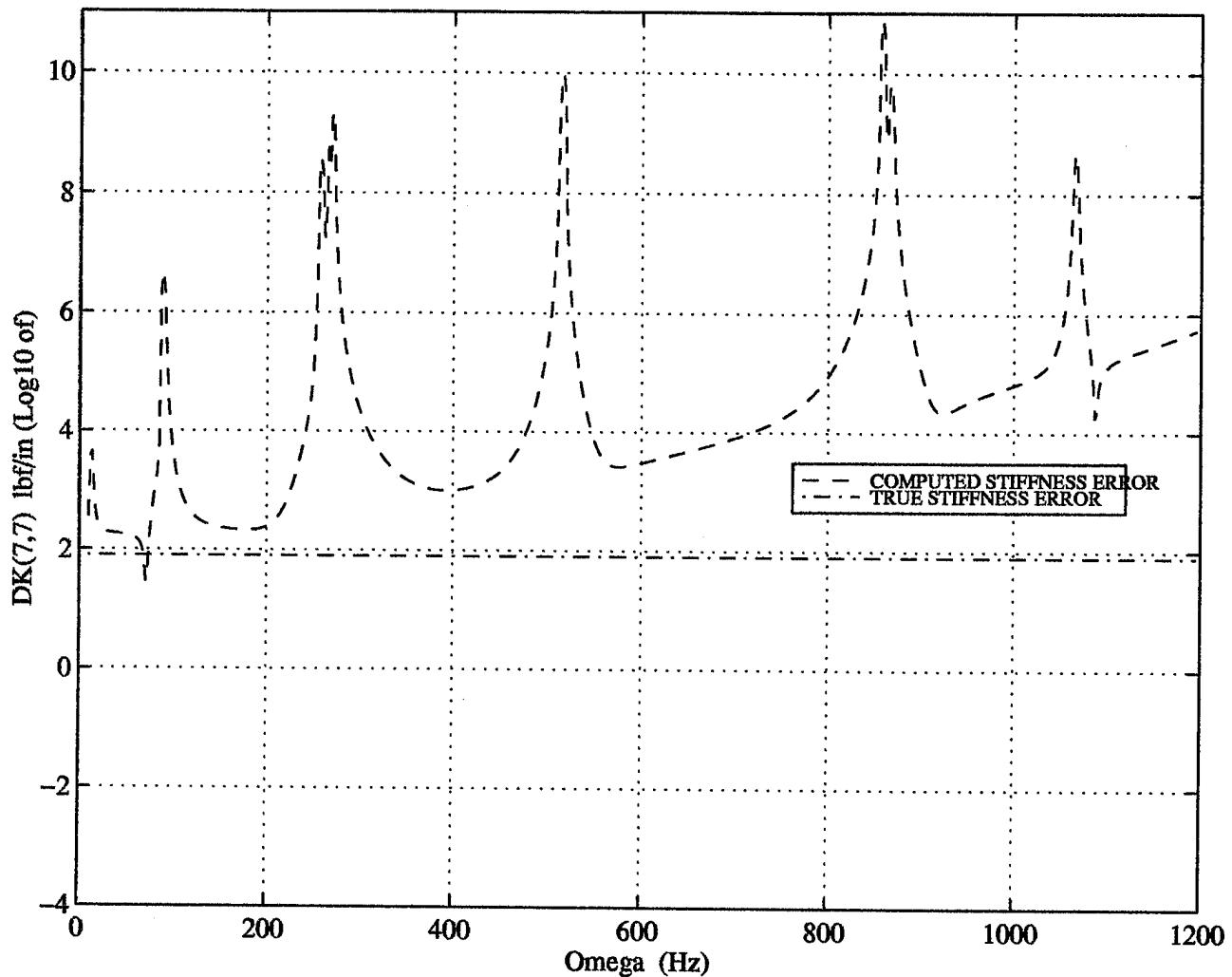


Figure 4- 8 Computed Stiffness vs True Stiffness for spatially incomplete beam.

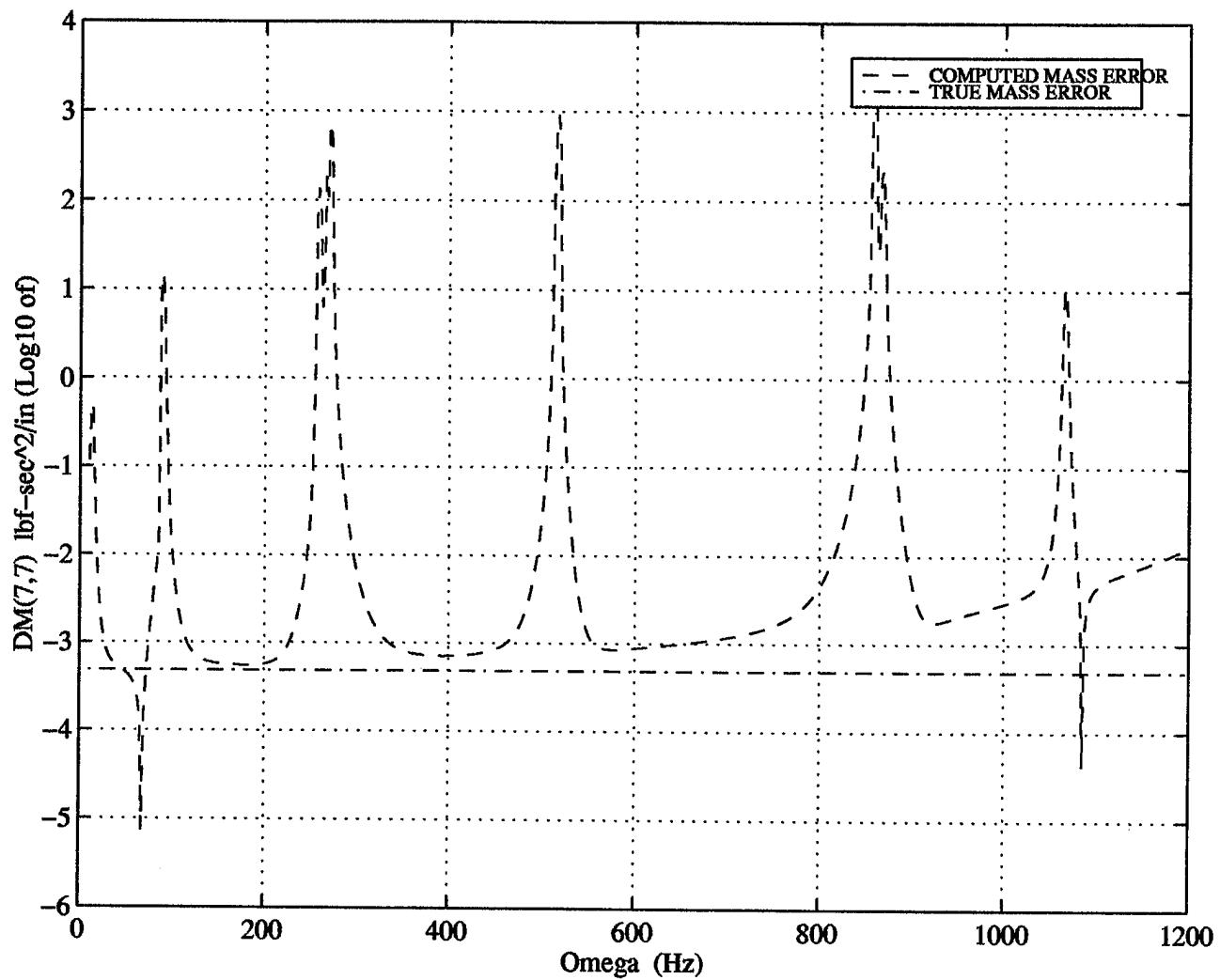


Figure 4-9 Computed Mass vs True Mass for spatially incomplete beam.

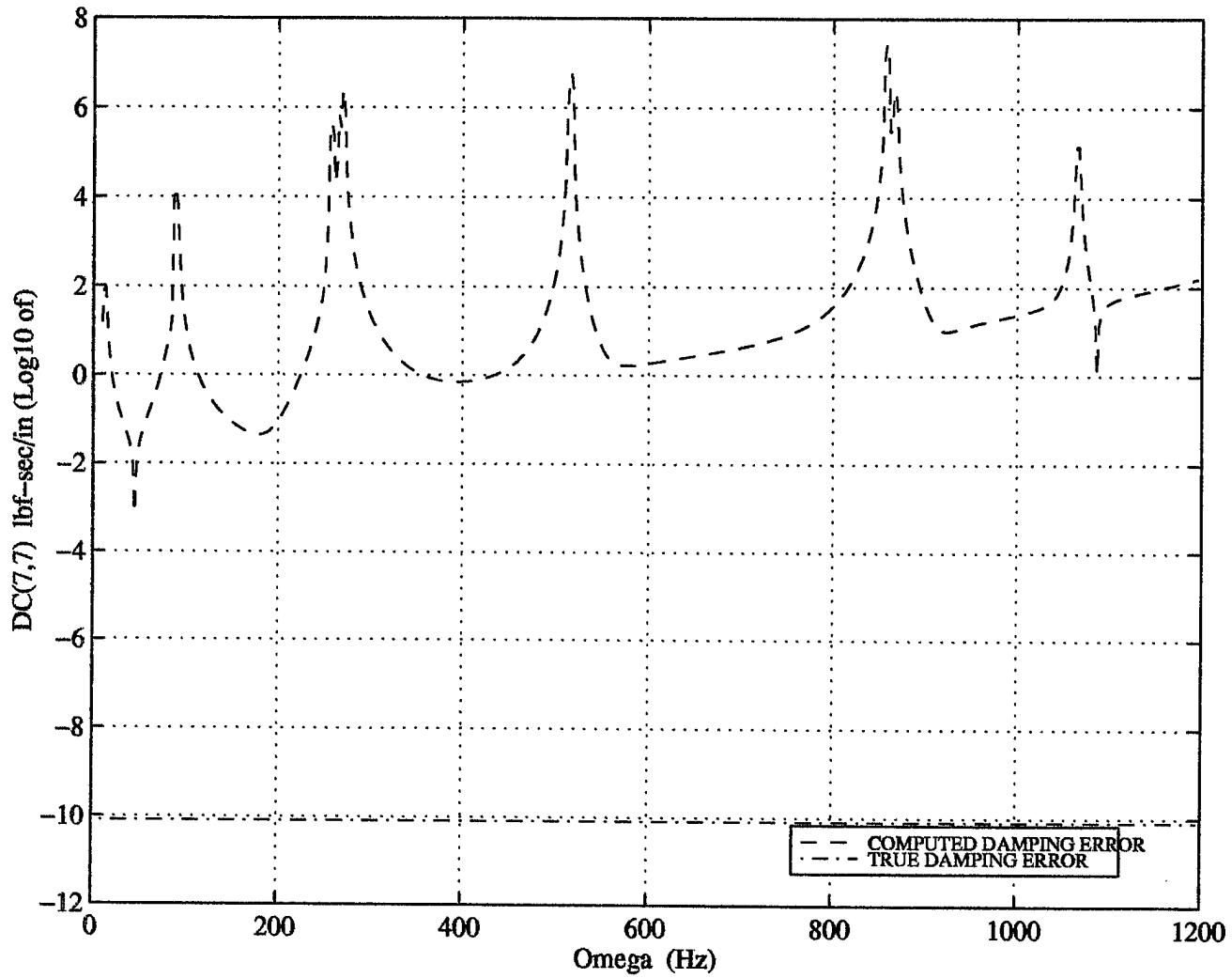


Figure 4- 10 Computed Damping vs True Damping for spatially incomplete beam.

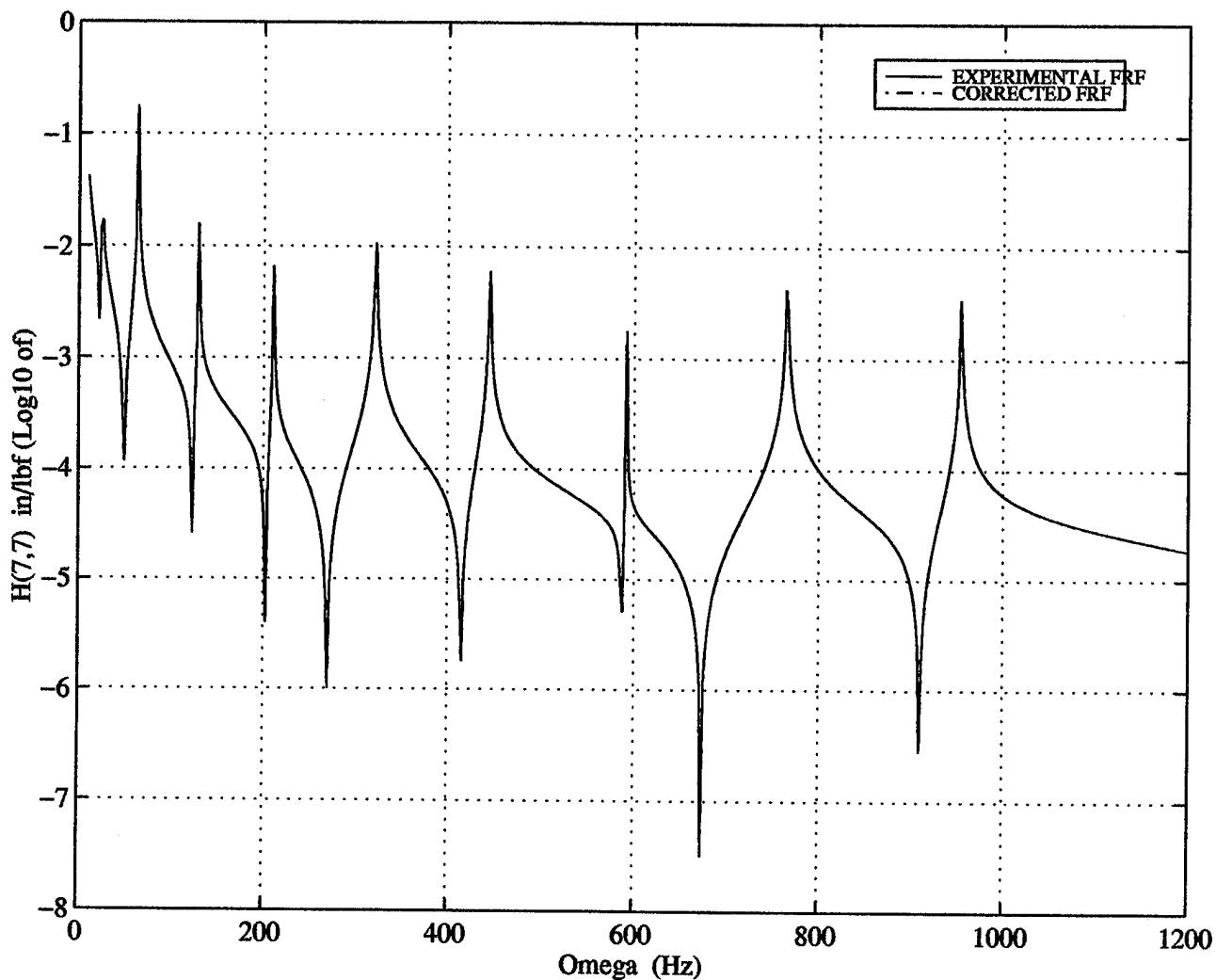


Figure 4- 11 ΔZ Corrected FRF vs experimental FRF for spatially incomplete beam. Plots are identical to within plot resolution.

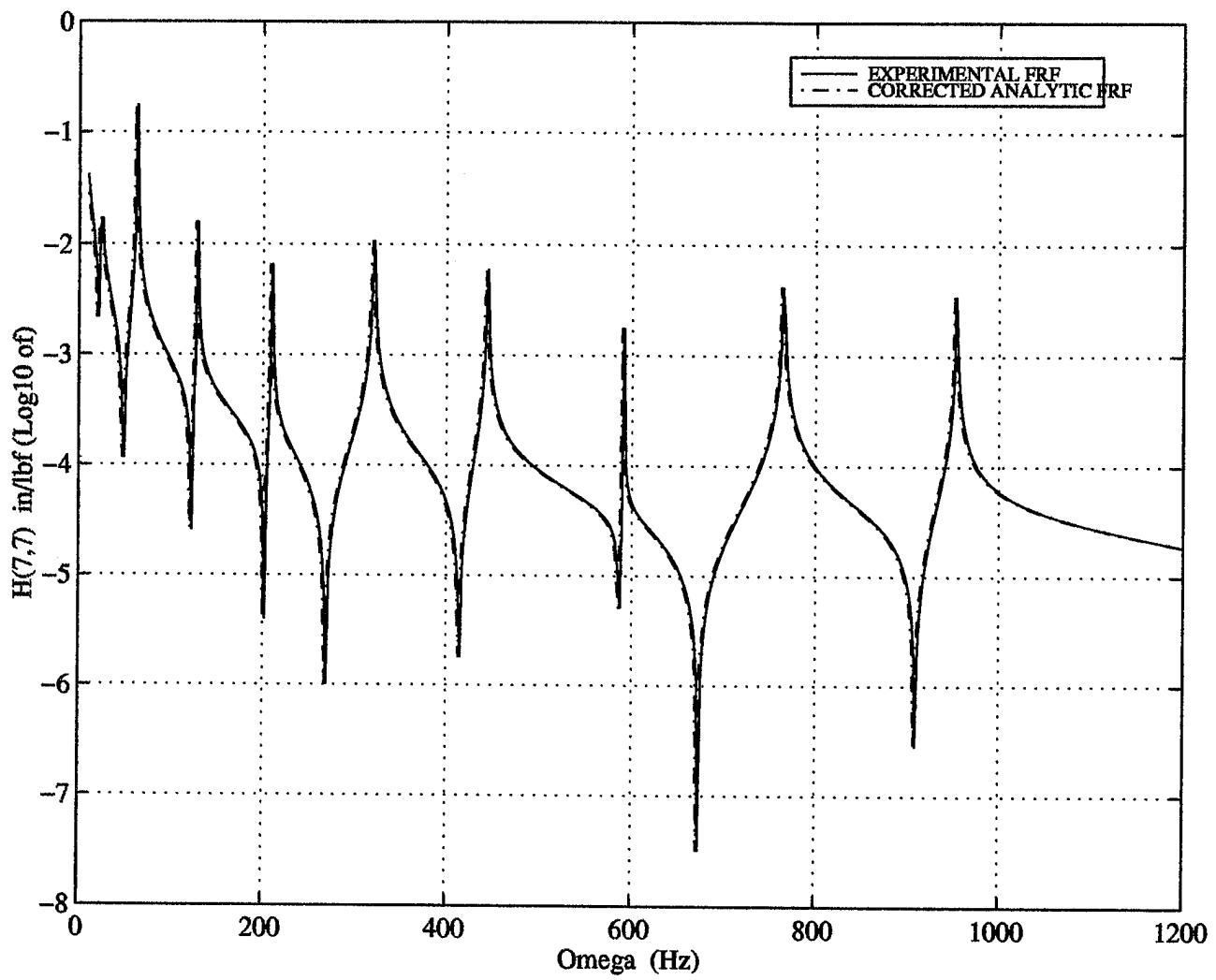


Figure 4- 12 $\Delta K/\Delta M/\Delta C$ Corrected FRF vs experimental FRF for spatially incomplete beam. The offset of the two plots is equal to the sampling frequency $\Delta\Omega$.

V. SINGLE MODE SOLUTIONS

A. SINGLE MODE MATRIX SOLUTIONS

In the case of a spatially complete system we have seen that the SST yielded frequency independent error matrices ΔK , ΔM , and ΔC that could be used to correct the stiffness, mass, and damping matrices of the FE model in such a manner that the FRF of the corrected FE model was exactly equal to that of the experimental system. In the case of a spatially incomplete system the constituent solutions ΔK , ΔM , and ΔC given in chapters III and IV were in general frequency dependent solutions which serve only as corrections to the reduced FE model. Ideally we seek frequency independent solutions which are corrections to the full FE model. For now we shall only deal with the simpler problem of trying to find frequency independent solutions which serve as corrections to the reduced FE model.

We shall employ Equation (2.30). For a given experimental system mode, ω_n , consider a frequency bandwidth $[\Omega_l, \Omega_u]$ such that $\omega_n \in [\Omega_l, \Omega_u]$. Let $\Xi = \{\Omega_1, \Omega_2, \dots, \Omega_m\}$ be a frequency sampling of the bandwidth $[\Omega_l, \Omega_u]$, i.e., $\Omega_l \leq \Omega_i \leq \Omega_u$ for each $\Omega_i \in \Xi$. For each $\Omega_i \in \Xi$ we can form the error impedance matrix $\Delta Z(\Omega_i)$. We apply Equation (2.30) to the partition yielding the following system of m equations in three unknowns:

$$\begin{bmatrix} \Delta Z_c(\Omega_1) \\ \vdots \\ \Delta Z_c(\Omega_i) \\ \vdots \\ \Delta Z_c(\Omega_m) \end{bmatrix} = \begin{bmatrix} I & -\Omega_1^2 I & j\Omega_1 I \\ \vdots & \vdots & \vdots \\ I & -\Omega_i^2 I & j\Omega_i I \\ \vdots & \vdots & \vdots \\ I & -\Omega_m^2 I & j\Omega_m I \end{bmatrix} \begin{bmatrix} \Delta K_c^{\omega_n} \\ \Delta M_c^{\omega_n} \\ \Delta C_c^{\omega_n} \end{bmatrix} \quad (5.1)$$

We will demonstrate by example that for properly chosen bandwidths $[\Omega_l, \Omega_u]$, the solution $\Delta K_c^{\omega_n}$, $\Delta M_c^{\omega_n}$, $\Delta C_c^{\omega_n}$ of Equation (5.1) approximately corrects the Analytic FRF over the frequency bandwidth $[\Omega_l, \Omega_u]$, i.e., if $H^*(\Omega)$ is the value of the FRF matrix of the experimental system at a frequency Ω , $\overline{H^a}(\Omega)$ the value of the FE reduced analytic FRF

matrix at Ω , and $\overline{H_{corr}^a(\Omega)}$ the value of the single mode corrected reduced Analytic FRF matrix at Ω for an experimental system mode ω_n where $\Omega \in [\Omega_l, \Omega_u]$ then

$$|H^x(\Omega) - \overline{H_{corr}^a(\Omega)}| \leq |H^x(\Omega) - \overline{H^a(\Omega)}| \quad (5.2)$$

We will refer to the solution $\Delta K_c^{\omega_n}$, $\Delta M_c^{\omega_n}$, $\Delta C_c^{\omega_n}$ as a single mode solution at ω_n . In the case of a spatially complete system, such as the spatially complete beam discussed in chapter III, all single mode solutions are found to be identical to the unique frequency independent solution that was obtained in chapter III, hence the corrections are valid throughout the frequency range of the experimental system and the left hand side of Equation 5.2 is zero. For a spatially incomplete system this is not the case.

In support of Equation (5.2) we chose a 1 Hz frequency bandwidth centered on mode 1 ($\omega_1=25.21$ Hz) of our spatially incomplete beam as defined in Chapter III. For our frequency sampling, Ξ , we will use a sampling frequency of .5 Hz which results in 3 points which are equally spaced over the interval, $\Xi=\{24.7178$ Hz, 25.2178 Hz, 25.7178 Hz $\}$.

Solving Equation (5.1) we get

$$\Delta K_c^{\omega_1} = \begin{bmatrix} -3.91 & -13.60 & 21.70 \\ -13.60 & 170.5 & -187.5 \\ -21.70 & -187.5 & 194.6 \end{bmatrix} \quad (5.3a)$$

$$\Delta M_c^{\omega_1} = \begin{bmatrix} -0.0001 & 0.0006 & -0.0005 \\ 0.0006 & -0.0033 & 0.0026 \\ -0.0005 & -0.0026 & -0.0016 \end{bmatrix} \quad (5.3b)$$

$$\Delta C_c^{\omega_1} = \begin{bmatrix} 0 + 0.0270j & 0 - 0.2291j & 0 + 0.2277j \\ 0 - 0.2291j & 0 + 1.5078j & 0 - 1.4627j \\ 0 + 0.2277j & 0 - 1.4627j & 0 + 1.3781j \end{bmatrix} \quad (5.3c)$$

Figure 5-1 shows a comparison of experimental, uncorrected and mode 1 corrected FRFs for our spatially incomplete beam using a 1 Hz frequency bandwidth centered on Mode 1 and a frequency sampling consisting of 3 frequencies equally spaced over the bandwidth. Figure 5-2 shows the results of including mode 1 and mode 2 in the

frequency bandwidth $[\Omega_l, \Omega_u]$. As Figure 5-2 shows Equation 5.1 is not as accurate when multiple frequencies are included in the bandwidth. Figure 5-3 is a comparison plot of the experimental and analytic FRF versus the single mode matrix solution corrected FRFs over 25, 10, and 1 Hz bandwidths using a 3 point frequency sampling. Figure 5-3 shows that the procedure is sensitive to the size of the bandwidth over which the solution is computed, i.e., better accuracy is achieved with smaller bandwidths. Figure 5-4 is a comparison plot of the experimental and uncorrected analytic FRFs versus the single mode matrix solution corrected FRFs computed over a 1 Hz bandwidth using frequency samplings of 200, 50, 10, and 3 points equally spaced over the bandwidth. Figure 5-4 shows that the procedure is fairly insensitive to sampling frequency.

Figure 5-5 is a comparison plot of the experimental and analytic FRFs versus the single mode solution corrected FRF for a 1 Hz bandwidth using a 3 point frequency sampling in the case of a spatially complete beam. As Figure 5-5 shows the procedure is exact for spatially complete systems.

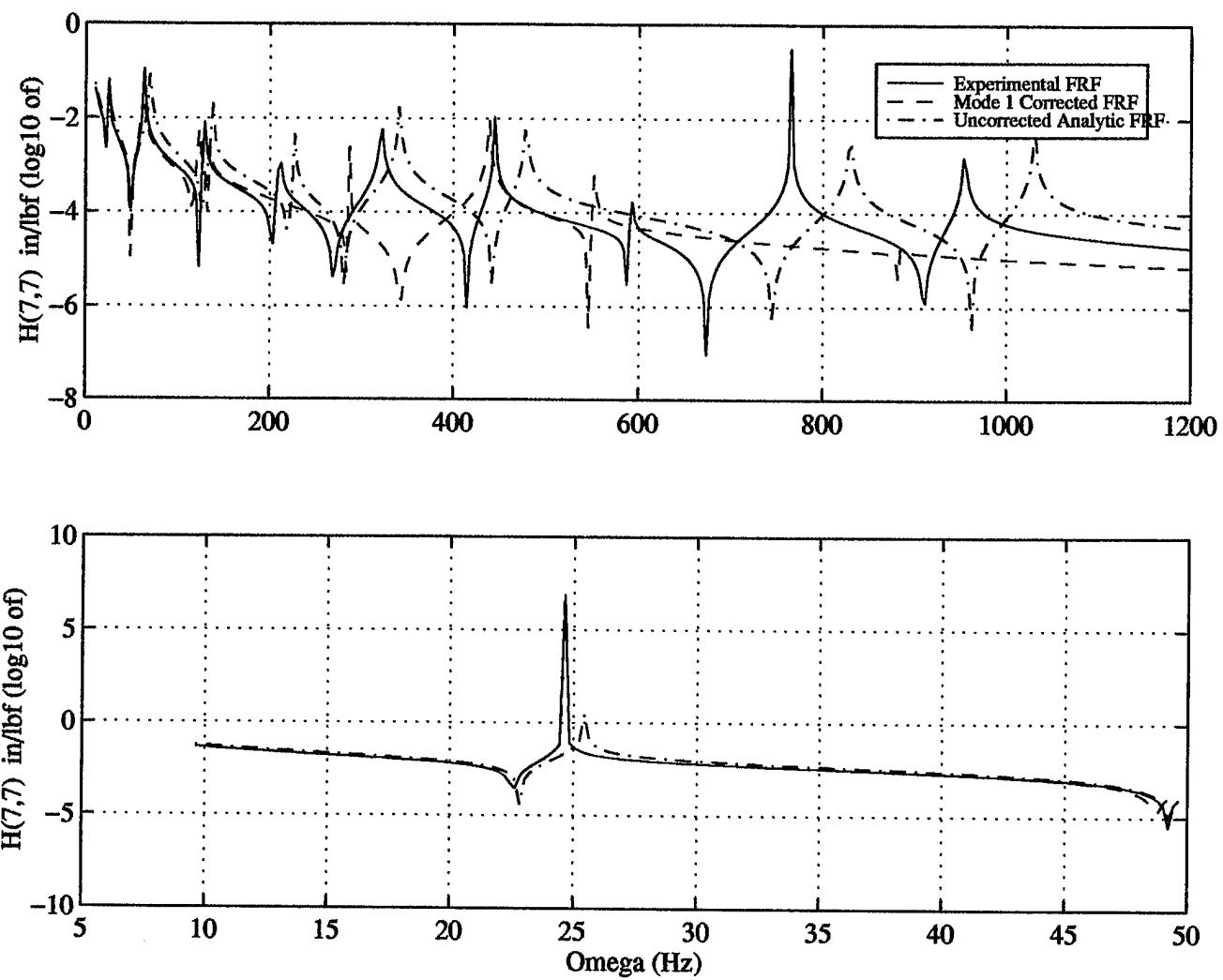


Figure 5- 1 Experimental and uncorrected Analytic FRFs vs single mode solution at mode 1 corrected FRF using a 3 point frequency sampling of a 1 Hz bandwidth.

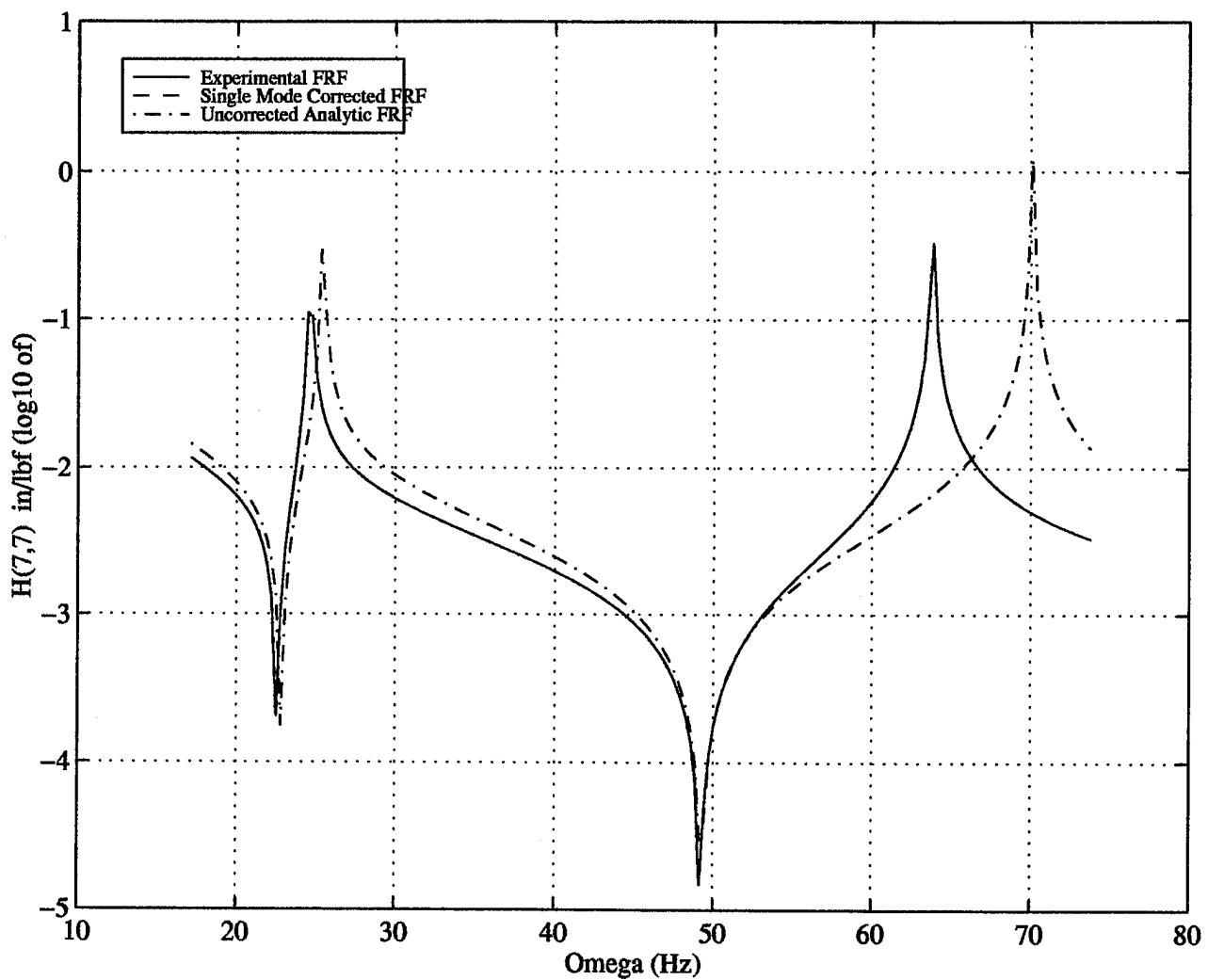


Figure 5- 2 Experimental and uncorrected analytic FRFs vs single mode corrected FRF using a 15 point frequency sampling of a bandwidth that includes modes 1 and 2.

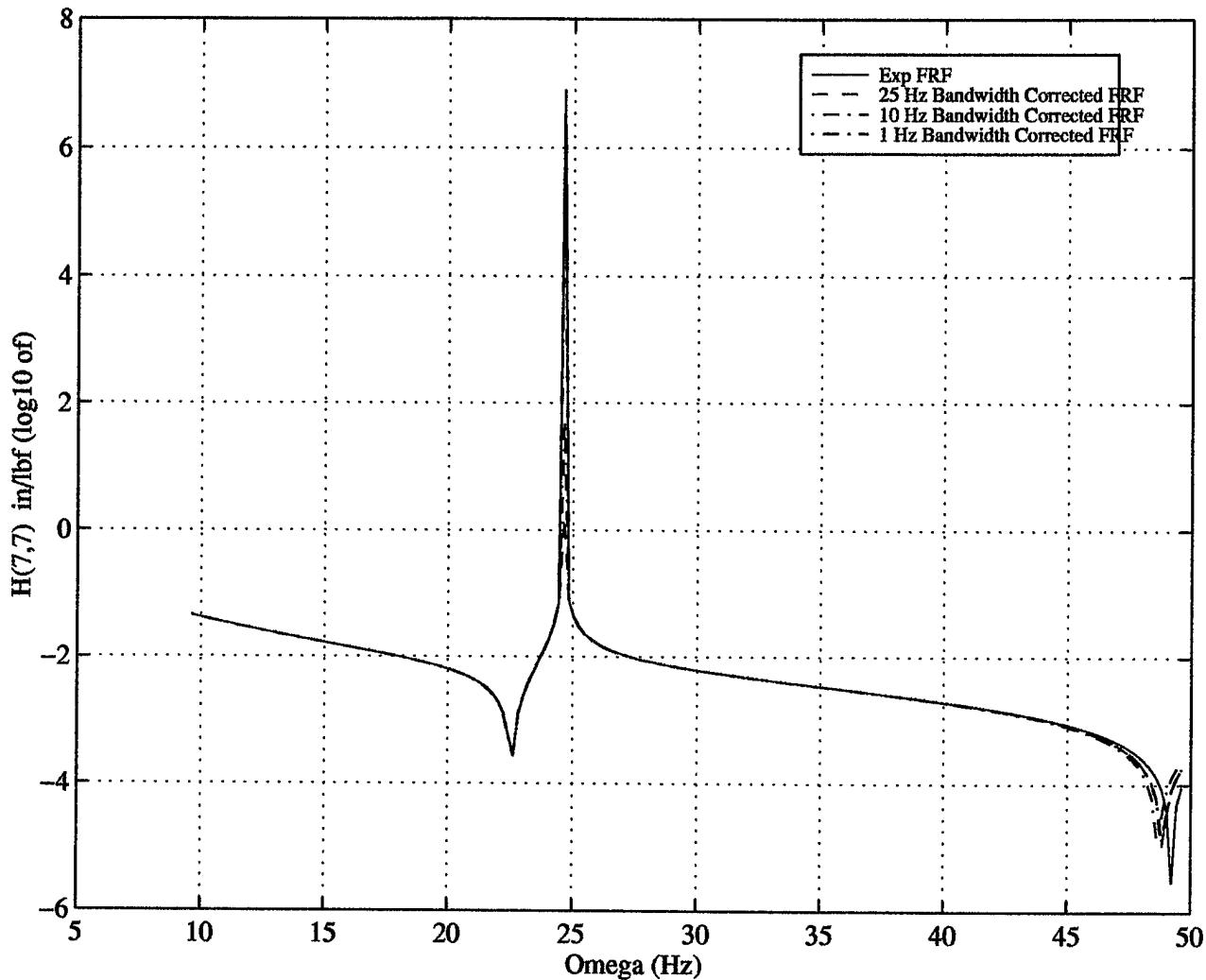


Figure 5- 3 Spatially incomplete experimental FRF vs single mode matrix solutions at mode 1 using 3 point frequency samplings of 25, 10, and 1 Hz bandwidths.

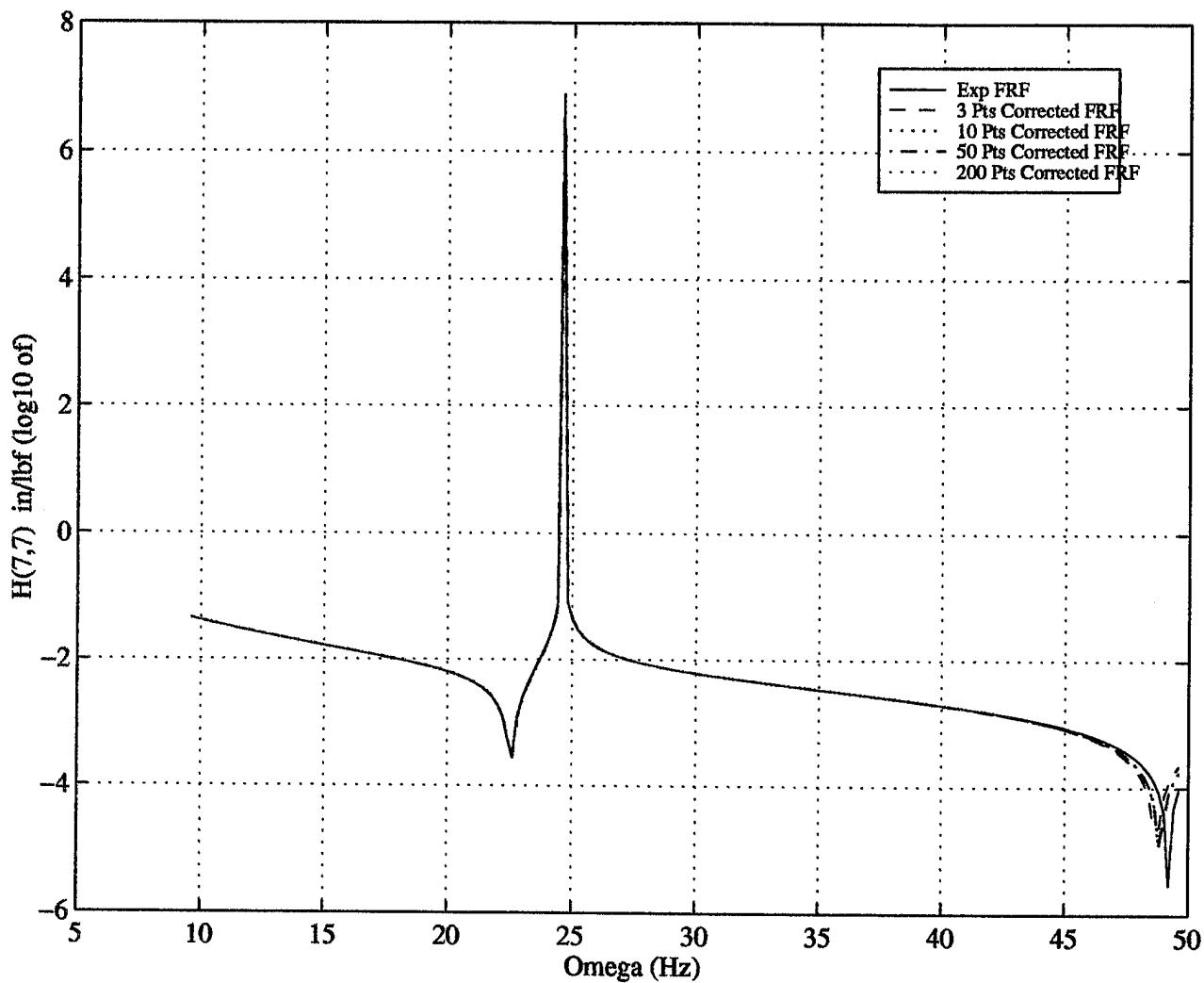


Figure 5- 4 Spatially incomplete experimental FRF vs single mode matrix solutions at mode 1 using 3, 10, 50, and 200 point frequency samplings of a 1 Hz bandwidth.

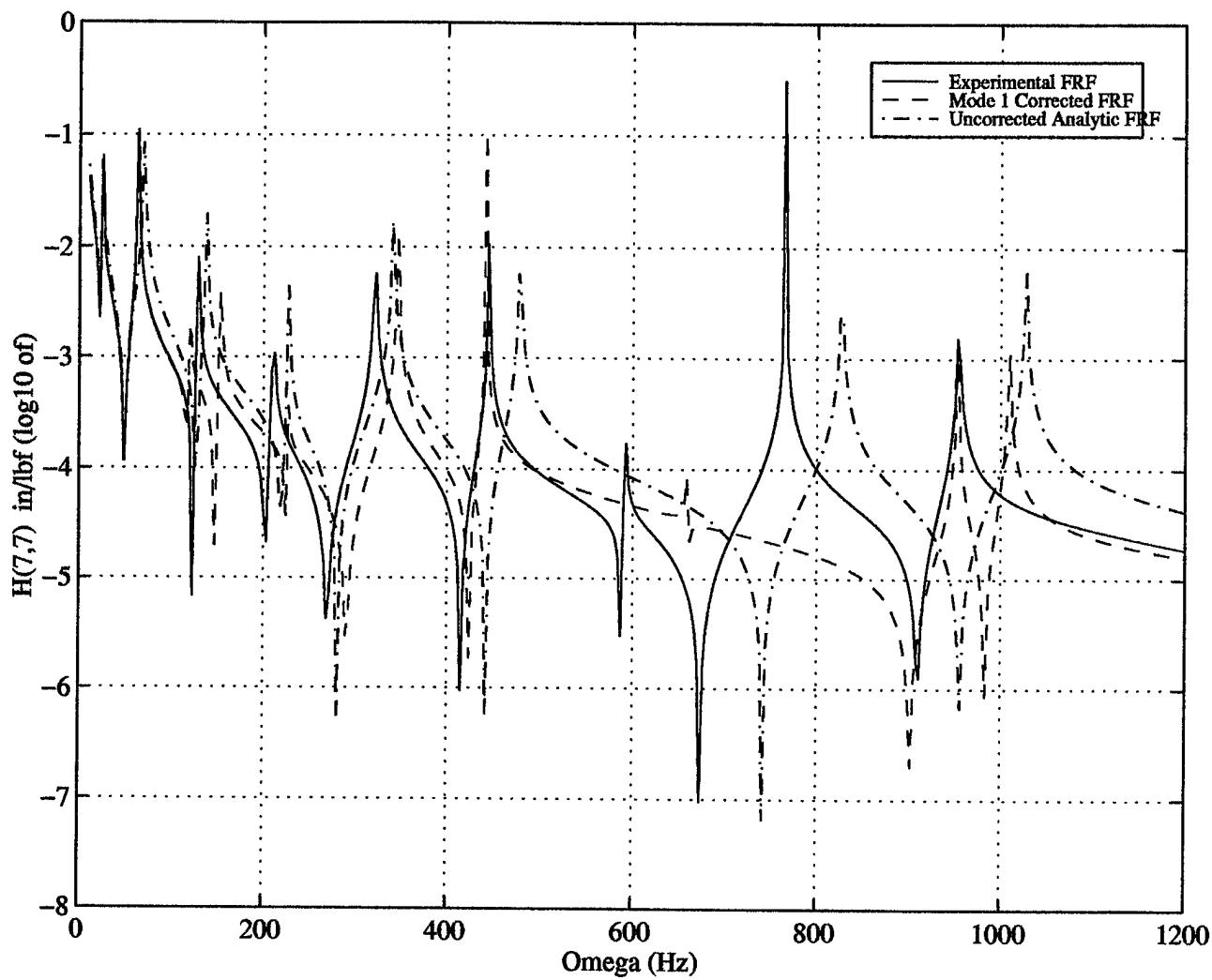


Figure 5- 5 Spatially complete experimental FRF vs single mode matrix solutions at mode 1 using a 3 point frequency samplings of a 1 Hz bandwidth.

B. SINGLE MODE INTEGRAL SOLUTIONS

In what follows we wish to employ the integral formulas of reference (5) to obtain a results similar to that of Equation (5.2). To accomplish this we shall need to recast the impedance equations of Chapter II in terms of velocity. We start with the equation of motion of a general 2nd order linear system

$$Kx + C\dot{x} + M\ddot{x} = f \quad (5.4a)$$

where

$$x = Xe^{j\Omega t} \quad (5.4b)$$

$$f = Fe^{j\Omega t} \quad (5.4c)$$

If we differentiate Equation (5.4b) we get that

$$\dot{x} = j\Omega Xe^{j\Omega t} = j\Omega x \quad (5.5)$$

hence

$$x = \frac{\dot{x}}{j\Omega} \quad (5.6)$$

differentiating Equation (5.4b) twice we get that

$$\ddot{x} = j\Omega(j\Omega Xe^{j\Omega t}) = j\Omega^2 x \quad (5.7)$$

substituting Equations (5.6) and (5.7) into Equation (5.4a) we get

$$K \frac{\dot{x}}{j\Omega} + C\dot{x} + Mj\Omega^2 x = f \quad (5.8)$$

We can rewrite equation (5.8) as

$$\left(\frac{K}{j\Omega} + C + Mj\Omega^2 \right) \dot{x} = f \quad (5.9)$$

Following reference (5) we can take the Laplace transform of equation (5.9) and thus write the impedance of the linear system in terms of the complex Laplace parameter s as

$$Z(s) = Ms + C + \frac{K}{s} \quad (5.10)$$

In reference (5) the impedance matrix of a general system is represented as an infinite Laurent series about the origin in powers of the complex Laplace parameter s as

$$Z(s) = \sum_{n=-\infty}^{\infty} A_n s^n = \lim_{n \rightarrow \infty} (A_n s^n + \cdots + A_1 s + A_0 + A_{-1} \frac{1}{s} + \cdots + A_{-n} \frac{1}{s^n}) \quad (5.11)$$

After defining a truncated Laurent series

$$\bar{Z}(s) = A_1 s + A_0 + A_{-1} \frac{1}{s} = \bar{M}s + \bar{C} + \frac{\bar{K}}{s} \quad (5.12)$$

which approximates the impedance matrix, an error function $E(s)$, and a cost J are defined as

$$E(s) = Z(s) - \bar{Z}(s) \quad (5.13)$$

and

$$J = \oint_{s=P} \|E(s)\|_E |W(s) \cdot ds| \quad (5.14)$$

where $W(s)$ is a complex valued weighting function, the subscript E denotes the euclidian norm of a matrix and the integration is performed over a prescribed path P in the complex plane. By setting the partial derivatives of J with respect to \bar{M} , \bar{K} , and \bar{C} to zero the authors of reference (5) obtained expressions for matrices \bar{M} , \bar{K} , and \bar{C} which approximate the stiffness, mass and damping matrices about a mode of the system. The expression for \bar{M} , \bar{K} , and \bar{C} are as follows:

$$\bar{M} = \frac{1}{ab - c^2} \left[b \int_{\Omega_1}^{\Omega_2} \Omega Z_I(i\Omega) |W(i\Omega)| d\Omega - c \int_{\Omega_1}^{\Omega_2} \frac{1}{\Omega} Z_I(i\Omega) |W(i\Omega)| d\Omega \right] \quad (5.15a)$$

$$\bar{C} = \frac{1}{c} \int_{\Omega_1}^{\Omega_2} Z_R(i\Omega) |W(i\Omega)| d\Omega \quad (5.15b)$$

$$\bar{K} = \frac{1}{ab - c^2} \left[c \int_{\Omega_1}^{\Omega_2} \Omega Z_I(i\Omega) |W(i\Omega)| d\Omega - a \int_{\Omega_1}^{\Omega_2} \frac{1}{\Omega} Z_I(i\Omega) |W(i\Omega)| d\Omega \right] \quad (5-15c)$$

where

$$a = \int_{\Omega_1}^{\Omega_2} \Omega^2 |W(i\Omega)| d\Omega \quad (5-15d)$$

$$b = \int_{\Omega_1}^{\Omega_2} \frac{1}{\Omega^2} |W(i\Omega)| d\Omega \quad (5-15e)$$

$$c = \int_{\Omega_1}^{\Omega_2} |W(i\Omega)| d\Omega \quad (5-15f)$$

$$Z(i\Omega) = Z_R(i\Omega) + iZ_I(i\Omega) \quad (5-15g)$$

We will apply Equation (5.15) to ΔZ as defined by the SST and obtain matrices $\Delta \bar{K}$, $\Delta \bar{M}$, and $\Delta \bar{C}$ which will serve as correction matrices of the FRF of the system about a given mode of the system.

As an example, let us use Equation (5.15) to compute the (1,1) element of the correction matrix $\Delta \bar{C}$ at mode 1 ($\omega_1=25.21$ Hz) for our spatially incomplete system. We take the weighting function to be $W(i\Omega)=1/(i\Omega)$, the path P is taken to be along the imaginary axis from 155.30 rads/sec to 161.58 rads/sec.

To compute this line integral we shall use the trapezoid rule with a three point frequency sampling

$$\Xi = \{155.30 \text{ rads/sec}, 158.44 \text{ rads/sec}, 161.58 \text{ rads/sec}\} \quad (5.16)$$

of the straight line path P (sampling frequency of π rads/sec). We first compute the value of the constant a of Equation (5.15d). Using vector notation we have that

$$\Omega(\Xi) = [155.30 \text{ rads/sec}, 158.44 \text{ rads/sec}, 161.58 \text{ rads/sec}] \quad (5.17a)$$

and

$$W(\Xi) = [0 - 0.0064j \text{ sec/rads}, 0 - 0.0063j \text{ sec/rads}, 0 - 0.0062j \text{ sec/rads}] \quad (5.17b)$$

hence the integrand is the vector

$$\Omega^2(\Xi) \cdot |W(\Xi)| = [155.30, 158.44, 161.58] \quad (5.18)$$

Using the trapezoid rule to compute the integral, we have that

$$a = \int_{155.30}^{161.58} \Omega^2(\Xi) |W(\Xi)| d\Omega = \frac{\pi}{2} \left(\frac{155.30}{2} + 158.44 + \frac{161.58}{2} \right) = 995.56 \text{ rads}^2/\text{sec}^2 \quad (5.19)$$

$b = .00000158 \text{ sec}^2/\text{rads}^2$ and $c = 0.0397$ are computed in a similar manner.

To actually compute the matrix $\Delta \bar{C}$, we use the SST to compute the matrices $\Delta Z(155.30 \text{ rads/sec})$, $\Delta Z(155.44 \text{ rads/sec})$, and $\Delta Z(161.58 \text{ rads/sec})$ for the points of the frequency sampling of the straight line path P . Using Equation (2.26) the matrix $\Delta Z(155.30 \text{ rads/sec})$ is seen to be

$$\Delta Z(155.44 \text{ rads/sec}) = \begin{bmatrix} (0.64E - 9) + 0.02j & (-0.04E - 8) - 0.04j & (0.03E - 7) + 0.01j \\ (-0.4E - 8) - 0.04j & (-0.4E - 9) - 0.04j & (0.25E - 7) - 0.10j \\ (0.3E - 8) + 0.01j & (0.25E - 7) - 0.10j & (-0.24E - 7) + 0.15j \end{bmatrix} \text{lbf/in} \quad (5.20)$$

The integral in Equation (5.15b) is computed on an element by element basis. For the (1,1) element of Equation (5.15b), if we collect the (1,1) elements of the matrices $\Delta Z(155.30 \text{ rads/sec})$, $\Delta Z(155.44 \text{ rads/sec})$, and $\Delta Z(161.58 \text{ rads/sec})$ and use the weighting vector given in Equation (5.17b) the integrand of Equation (5.15b) is the vector

$$\Delta Z_R^{(1,1)}(\Xi) |W(\Xi)| = [0.0064 \ 0.040 \ -0.848] [0.64 \ 0.63 \ 0.62] (E - 9) \text{lbf/in sec/rads} \quad (5.21a)$$

$$\Delta Z_R^{(1,1)}(\Xi) |W(\Xi)| = [0.0041 \ 0.2885 \ 0.0036] (E - 9) \text{lbf/in sec/rads} \quad (5.21b)$$

Computing the integral we have

$$\Delta \bar{C}(1,1) = \frac{1}{c} \int_{155.30}^{161.58} \Delta Z_R^{(1,1)}(\Xi) |W(\Xi)| d\Omega = \frac{1}{0.0397} \frac{\pi}{2} \left(\frac{0.0041}{2} + 0.2885 + \frac{0.0036}{2} \right) (E - 9) \text{lbf/in} \quad (5.22a)$$

$$\Delta \bar{C}(1,1) = 0.0232E - 6 \text{lbf/in} \quad (5.22b)$$

Performing the above procedures for the remaining elements of Equation (5.15b) we get

$$\Delta \bar{C} = \begin{bmatrix} 0.0232 & -0.0443 & 0.0300 \\ -0.0443 & 0.1278 & -0.0914 \\ 0.0300 & -0.0914 & 0.0645 \end{bmatrix} (E - 6) \text{ lbf/in}$$
(5.23a)

In a similar manner we have

$$\Delta \bar{M} = \begin{bmatrix} -0.0001 & -0.0001 & 0.0002 \\ -0.0001 & 0.0015 & -0.0020 \\ 0.0002 & -0.0020 & 0.0027 \end{bmatrix} \text{ lbm}$$
(5.23b)

$$\Delta \bar{K} = \begin{bmatrix} -6.0476 & 4.5347 & 3.6757 \\ 4.5347 & 51.2196 & -71.7061 \\ 3.6757 & -71.7061 & 85.4417 \end{bmatrix} \text{ lbf/in}$$
(5.23c)

Figure 5-6 is a comparison plot of experimental and uncorrected analytic FRFs versus single mode integral solutions at mode 1 using a frequency sampling of 3 points equally spaced over bandwidths of 25, 10, and 1 Hz. As with the matrix solutions, the integral solutions are sensitive to the length of the bandwidth over which the solutions are computed with smaller bandwidths yielding more accurate solutions. Figure 5-7 is a comparison plot of experimental and uncorrected analytic FRFs versus single mode integral solutions at mode 1 over a 1 Hz bandwidth using frequency samplings of 3, 10, 50, and 200 points from the bandwidth. As with the matrix solutions, the integral solutions are also insensitive to the sampling frequency.

Figures 5-8, 5-9, and 5-10 are comparison plots of matrix and integral solutions at mode 1 computed over 25, 10, and 1 Hz bandwidths respectively, using three point frequency samplings. These three plots show that for these frequency sampling, the matrix solutions are more accurate than the integral solutions. We must note that we have used the trapezoid rule for computation of the integrals. It is expected that better accuracy would be achieved from the integral solutions if a higher order integration method such as Simpson's rule were used.

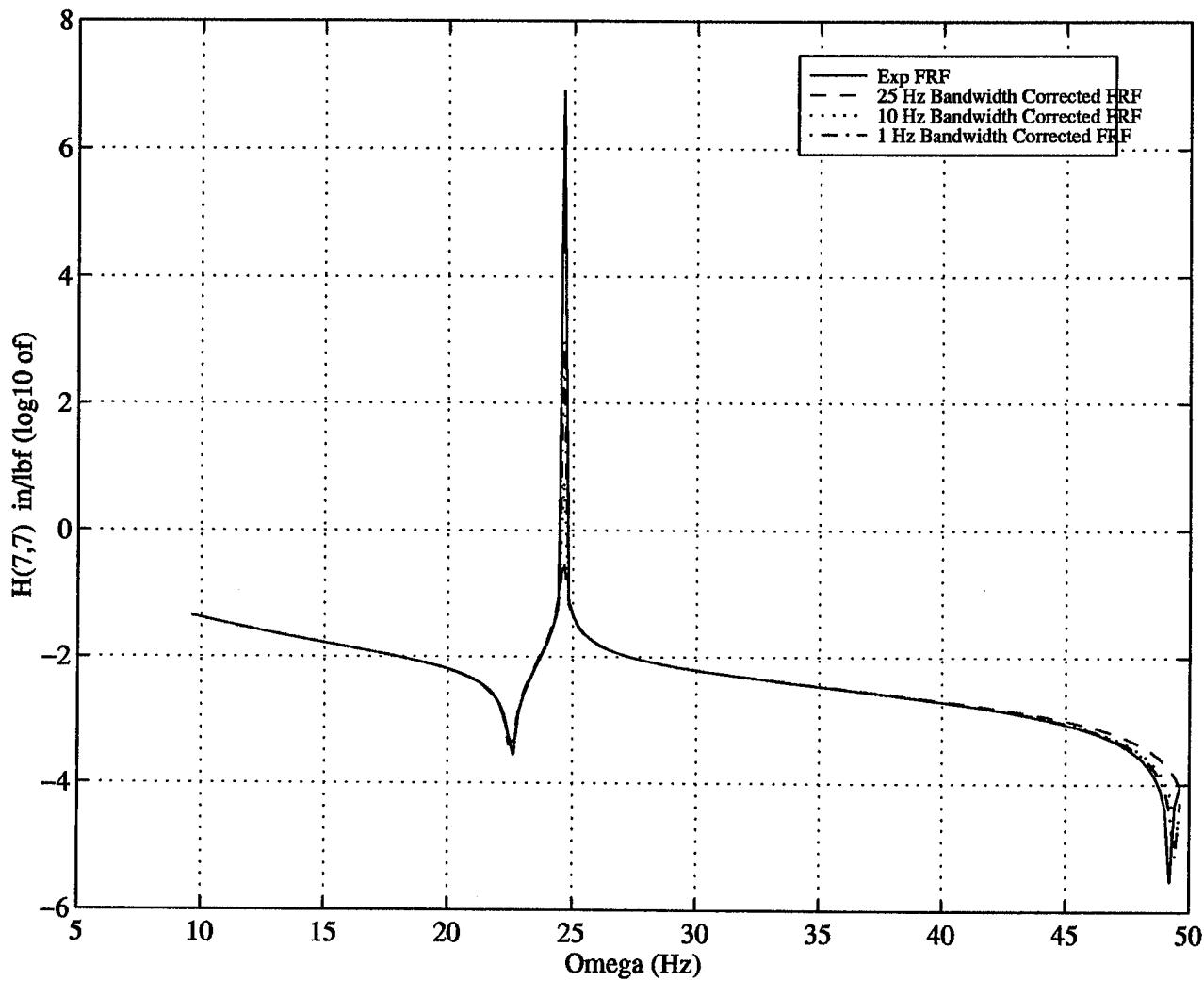


Figure 5- 6 Spatially incomplete experimental FRF vs single mode integral solutions at mode 1 using 3 point frequency samplings of 25, 10, and 1 Hz bandwidths.

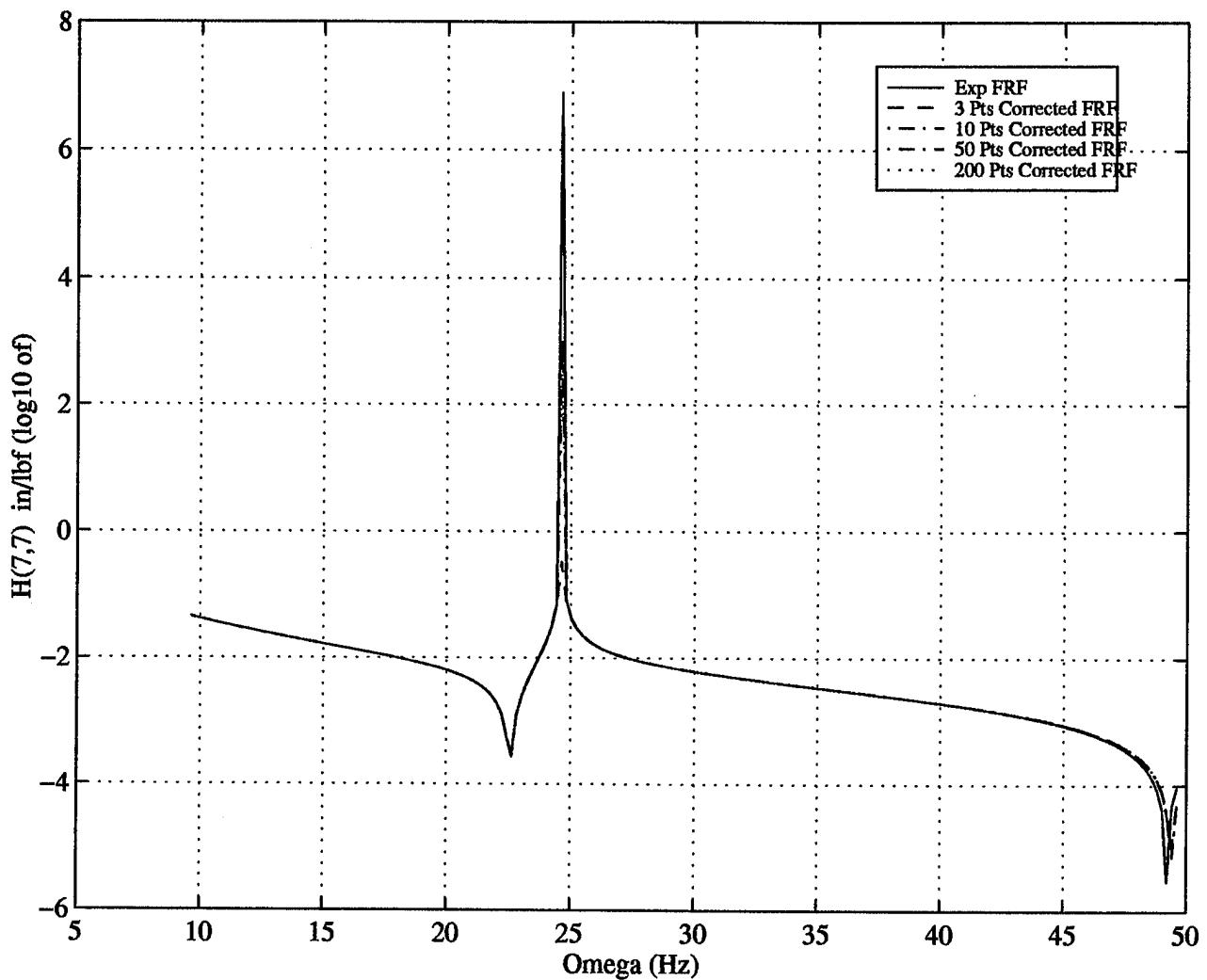


Figure 5-7 Spatially incomplete experimental FRF vs single mode integral solutions at mode 1 using 3, 10, 50 , and 200 point frequency samplings of a 1 Hz bandwidth.

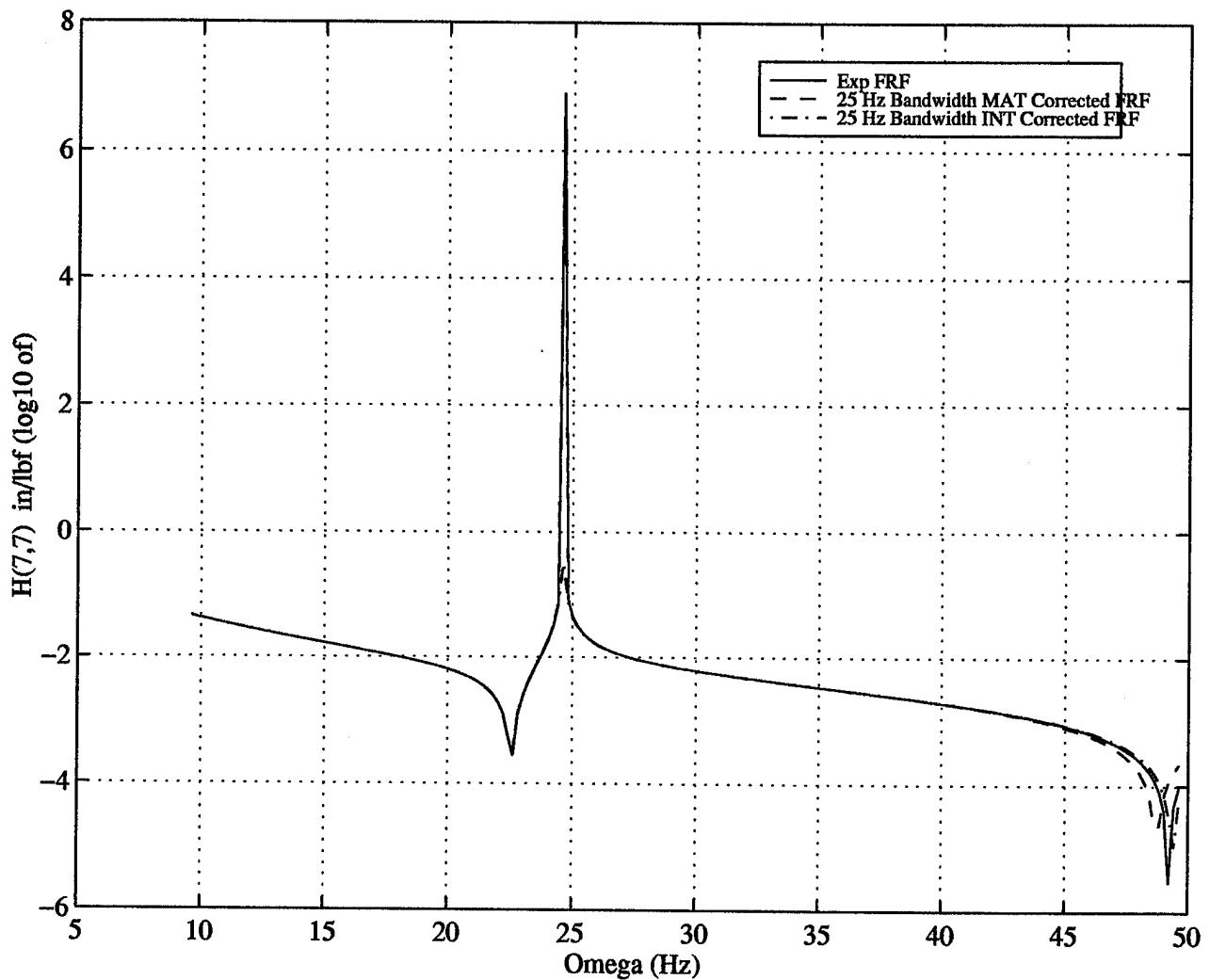


Figure 5- 8 Experimental FRF vs single mode integral and matrix solutions at mode 1 using a 25 Hz bandwidth with a 3 point frequency sampling.

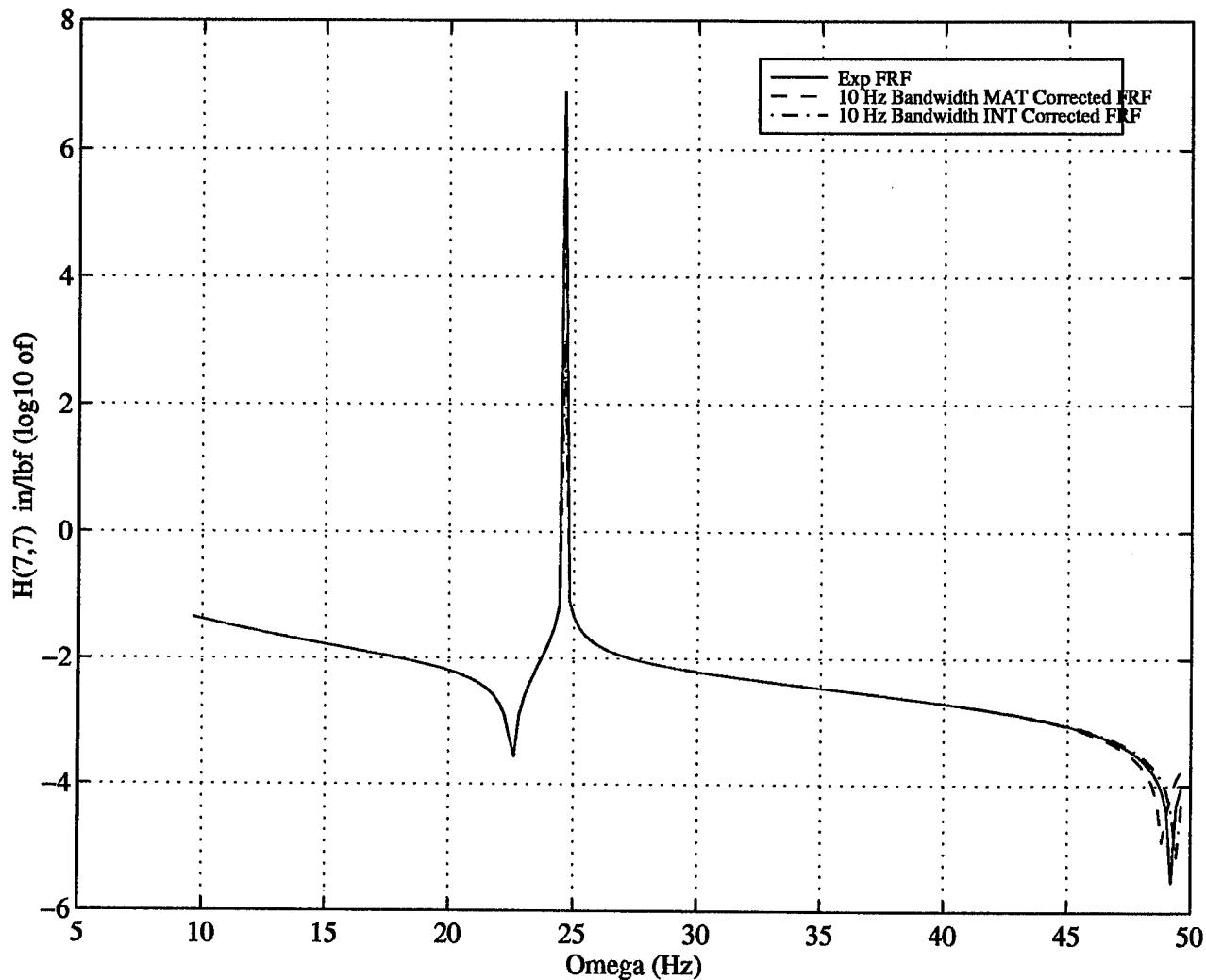


Figure 5-9 Experimental FRF vs single mode integral and matrix solutions at mode 1 using a 10 Hz bandwidth with a 3 point frequency sampling.

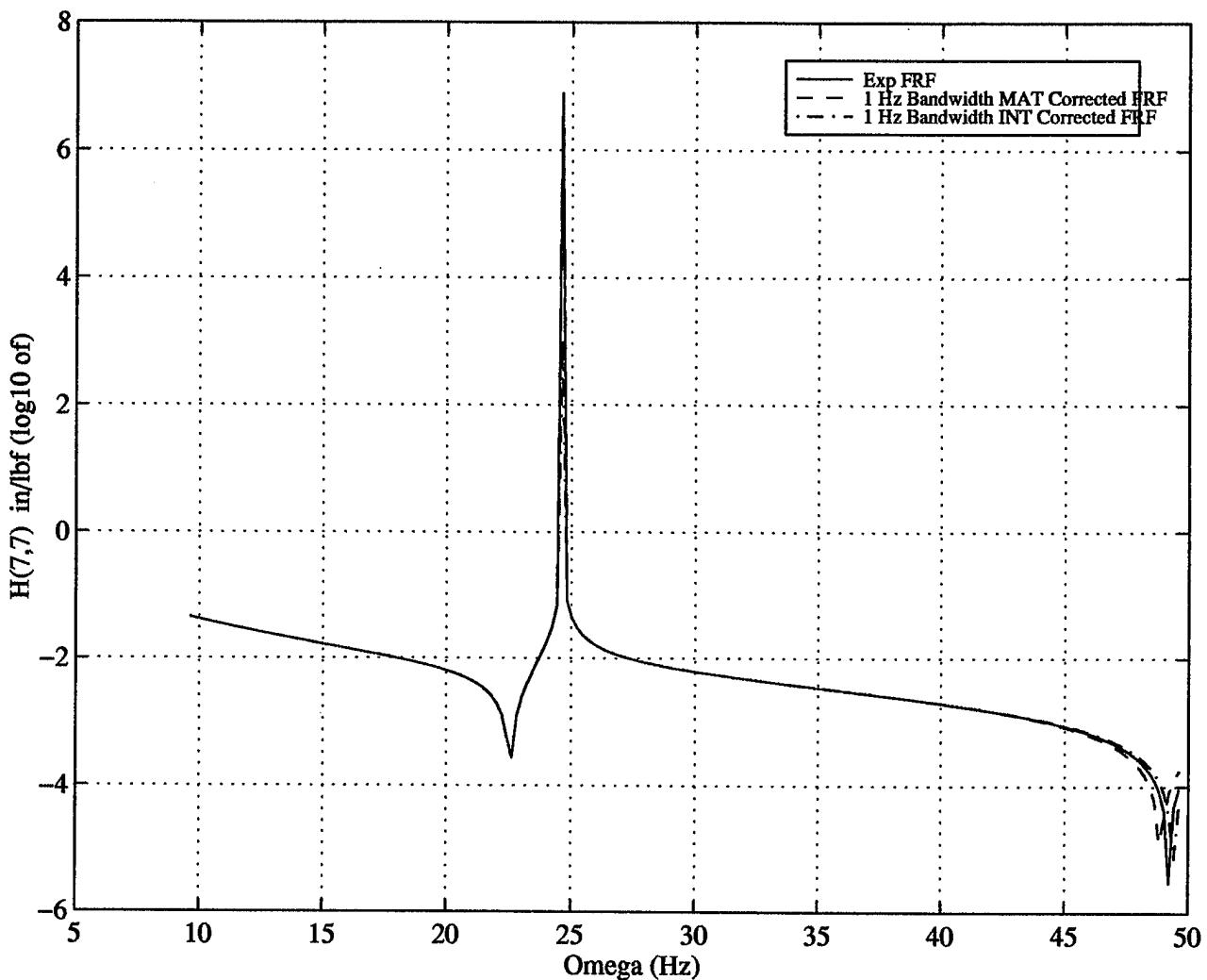


Figure 5- 10 Experimental FRF vs single mode integral and matrix solutions at mode 1 using a 1 Hz bandwidth with a 3 point frequency sampling.

VI. MULTIPLE MODE SOLUTIONS

A. MULTIPLE MODE MATRIX SOLUTIONS

We wish to extend the results of Chapter V to multiple modes of the system under consideration. To accomplish this we shall simply extend Equation (5.1) over multiple frequency samplings. For a set of system modes $\{\omega_1, \omega_2, \dots, \omega_n\}$, let $\Xi(\omega_i) = \{\Omega_{i1}, \Omega_{i2}, \dots, \Omega_{ik}\}$ be a frequency sampling of a bandwidth $[\Omega_{il}, \Omega_{iu}]$ about ω_i for $i=1, 2, \dots, n$. We shall apply Equation (2.30) to the frequency sampling

$$\Xi = \bigcup_{i=1}^n \Xi(\omega_i) \quad (6.1)$$

which is the set theoretic union of the sets $\Xi(\omega_i)$ for $i=1, 2, \dots, n$. If, for simplicity, we restrict ourselves to equally sized frequency samplings we obtain a set on nk equation in three unknowns

$$\begin{bmatrix} \Delta Z_c(\Omega_{11}) \\ \vdots \\ \Delta Z_c(\Omega_{ij}) \\ \vdots \\ \Delta Z_c(\Omega_{nk}) \end{bmatrix} = \begin{bmatrix} I & -\Omega_{11}^2 I & j\Omega_{11} I \\ \vdots & \vdots & \vdots \\ I & -\Omega_{ij}^2 I & j\Omega_{ij} I \\ \vdots & \vdots & \vdots \\ I & -\Omega_{nk}^2 I & j\Omega_{nk} I \end{bmatrix} \begin{bmatrix} \Delta K_c^\Xi \\ \Delta M_c^\Xi \\ \Delta C_c^\Xi \end{bmatrix} \quad (6.2)$$

One can solve Equation (6.2) for $\begin{bmatrix} \Delta K_c^\Xi \\ \Delta M_c^\Xi \\ \Delta C_c^\Xi \end{bmatrix}$ as we have done in Chapter V using the

familiar MATLAB pseudoinverse function but better results are obtained if we weight the equations as discussed in [Ref. 6]. A good weighting is to assign the weight ω_i to those equations associated with the frequency sampling $\Xi(\omega_i)$. Alternately we could assign the weight $1/\omega_i$ to those equations associated with the frequency sampling $\Xi(\omega_i)$. The ω_i weighting results in the solution being more accurate at the lower modes while the $1/\omega_i$ weighting gives better accuracy at the higher modes.

We note that the solution to Equation (6.2) approximately corrects the analytic FRF in the sense of Equation (5.1) when the matrices $\Delta Z(\Omega_{ij})$ as given by Equation (2.26),

are computed over a 'good' error set, C_{err} . In general, due to the smearing that occurs during reduction of the full analytic system to the reduced analytic system, the set C_{err} is not the same set as the intersection of the spatially complete error set and the A set. It is in general a larger set than this intersection and the size is a function of the type of error, i.e., mass, damping, or stiffness error and the locations of the errors. Figure 6-1 is a comparison plot of experimental and uncorrected analytic FRFs versus the multiple mode matrix solution corrected FRF with the solution having been computed over modes 1 and 2 using a 1 Hz bandwidth about each mode with a 3 point frequency sampling over the bandwidth at each of the modes. Figure 6-2 is a comparison plot using a matrix solution computed over mode 1 through 4 again using a 1 Hz bandwidth about each mode with a 3 point frequency sampling over the bandwidth at each of the modes.

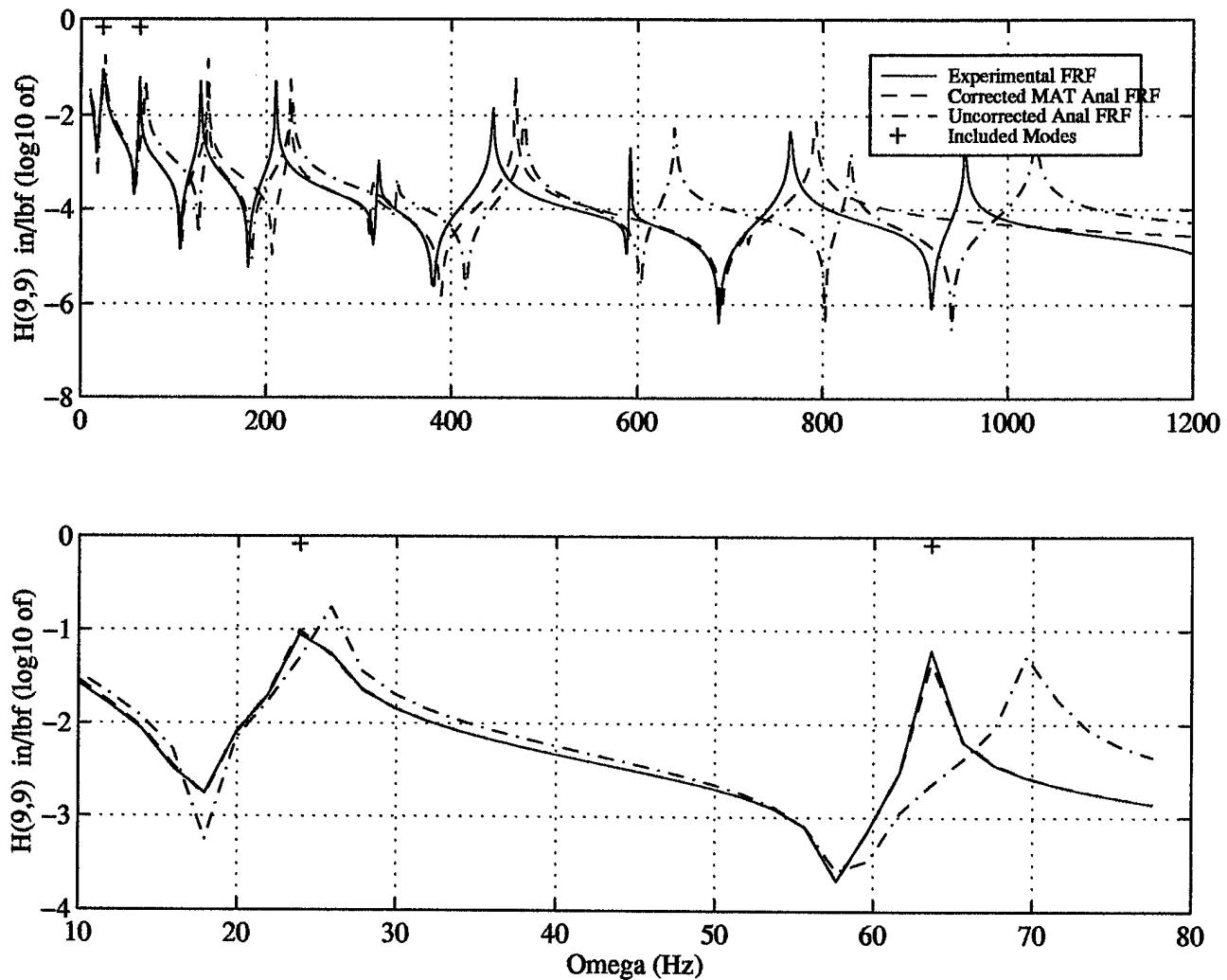


Figure 6- 1 Experimental FRF vs multiple mode matrix solutions at modes 1 and 2 using a 1 Hz bandwidth with a 3 point frequency samplings at each mode.

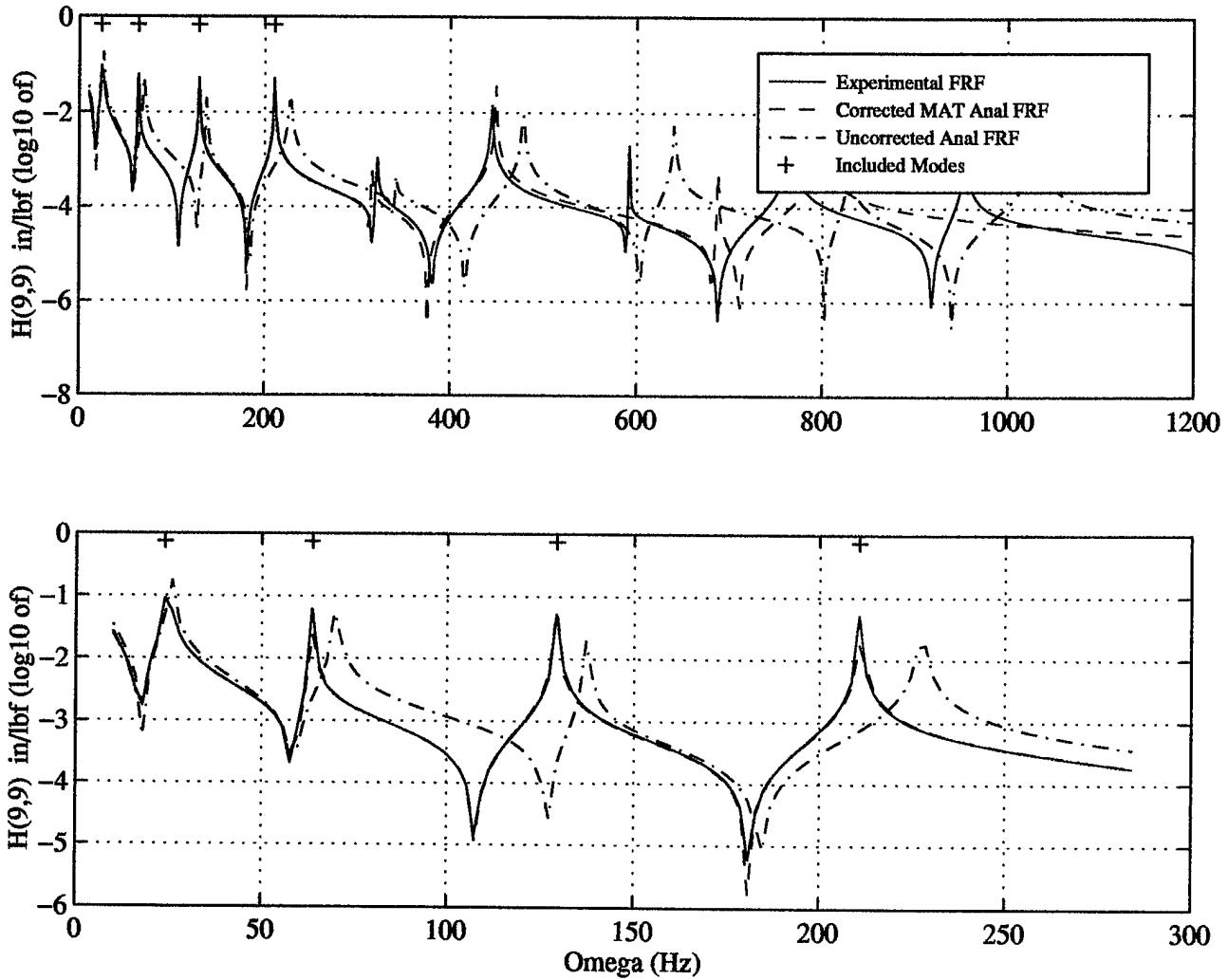


Figure 6- 2 Experimental FRF vs multiple mode matrix solutions at modes 1 through 4 using a 1 Hz bandwidth with a 3 point frequency samplings at each mode.

B. MULTIPLE MODE INTEGRAL SOLUTIONS

We wish to extend single mode integral solutions to multiple modes of the system. As with the multiple mode matrix solutions we simply take as the path of integration a path which is the set theoretic union of suitable paths at each of the modes under consideration. As with the matrix solutions one can chose the weighting function $W(s)$ to be $1/(\Omega)$ or Ω to achieve lower or higher mode accuracy. As stated in the previous section it is required that a 'good' error set is known. Figure 6-3 is a comparison of experimental and uncorrected FRF versus multiple mode integral solution corrected FRF with the solution having been computed over modes 1 and 2 using a 1 Hz bandwidth at each mode with a 3 point frequency sampling over the bandwidth at each of the modes. Figure 6-4 is a comparison using a solution computed over mode 1 through 4 again using a 1 Hz bandwidth at each mode with a 3 point frequency sampling over the bandwidth at each of the modes. Figure 6-5 is a comparison plot of the multiple mode matrix and integral solutions over modes 1 through mode 4 using a 1 Hz bandwidth at each mode with a 3 point frequency sampling over the bandwidth at each of the modes.

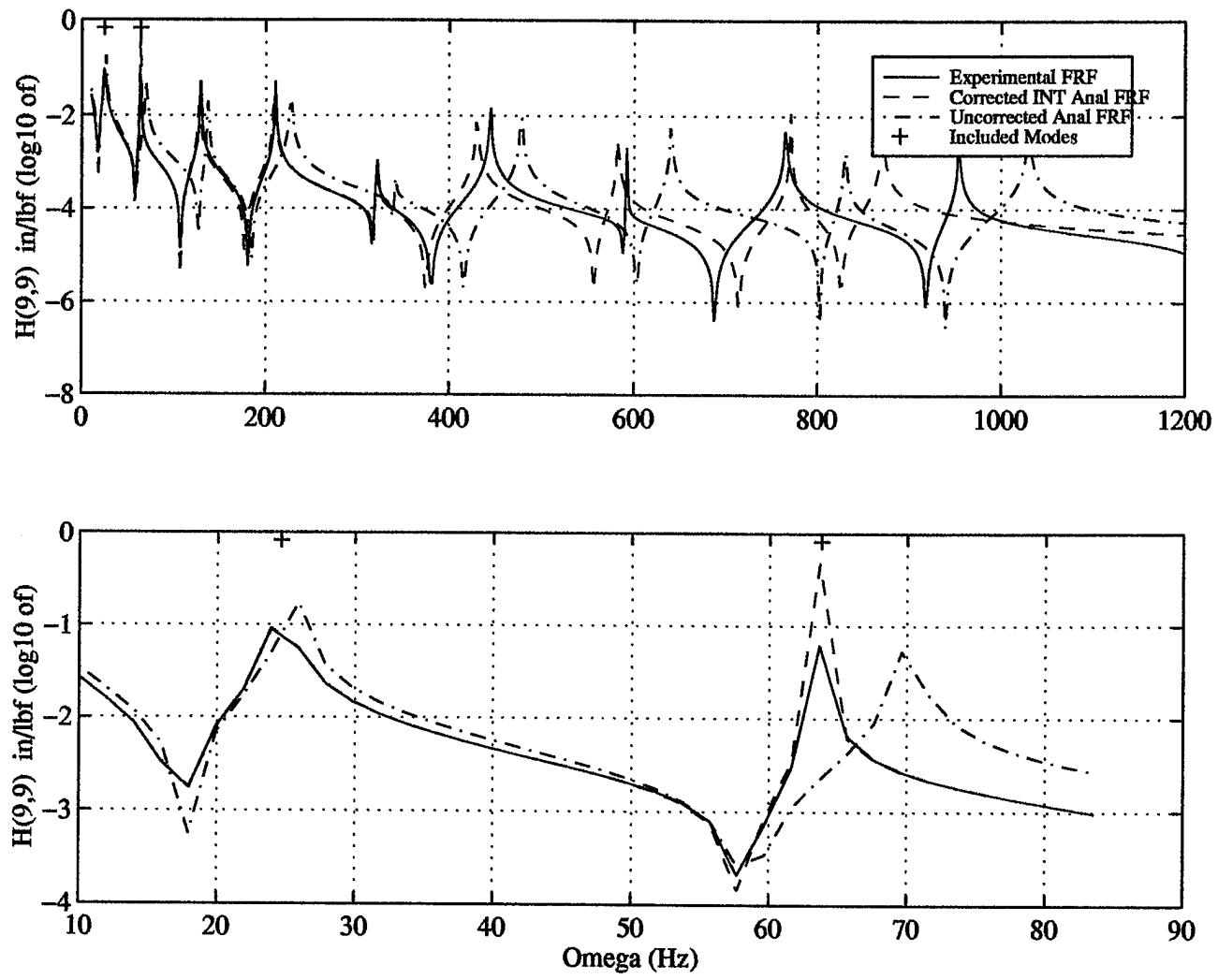


Figure 6- 3 Experimental FRF vs multiple mode integral solutions at modes 1 and 2 using a 1 Hz bandwidth with a 3 point frequency samplings at each mode.

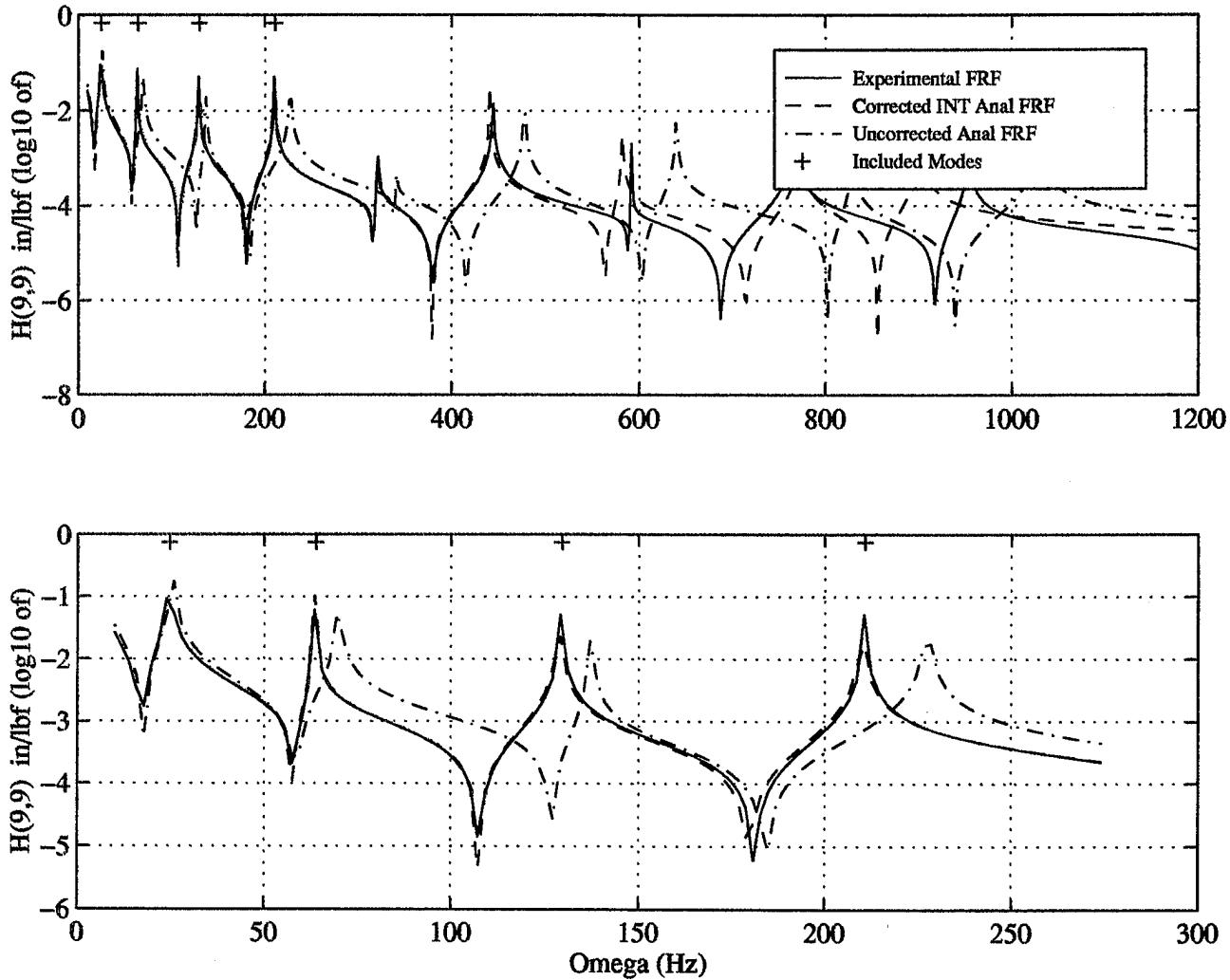


Figure 6- 4 Experimental FRF vs multiple mode integral solution at modes 1 through 4 using a 1 Hz bandwidth with a 3 point frequency samplings at each mode.

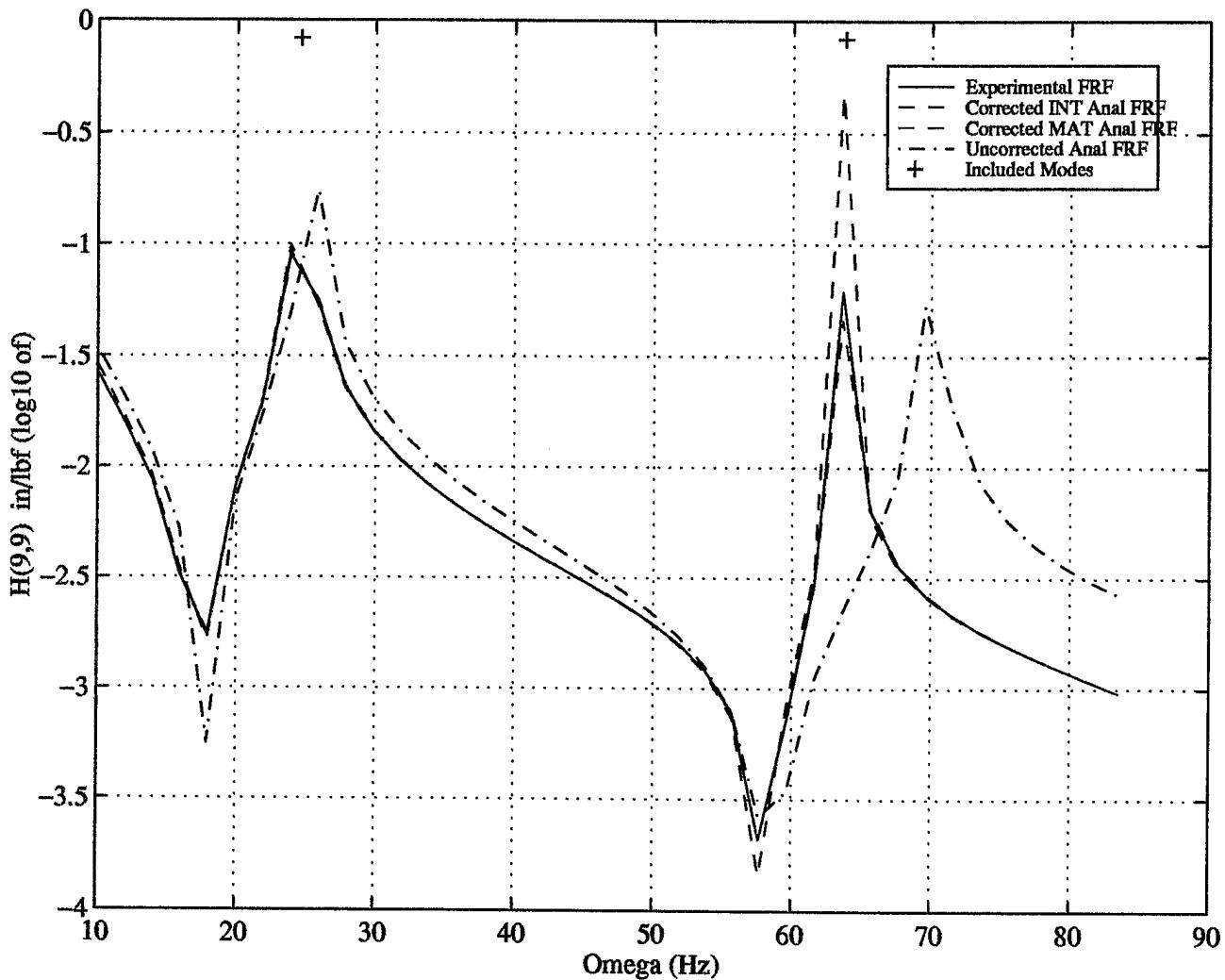


Figure 6-5 Experimental FRF vs multiple mode matrix and integral solutions at modes 1 and 2 using a 1 Hz bandwidth with a 3 point frequency samplings at each mode.

C. SINGLE POINT MULTIPLE MODE SOLUTIONS

The results of sections A and B can be performed using partitions consisting of a single point. Figure 6-6 shows the results of performing a multiple mode matrix solution over the first three modes of our spatially incomplete beam using a single point partition at each of the modes under consideration. Figure 6-7 shows the results of performing a multiple point integral solution over the first three modes of our spatially incomplete beam using a single point partition at each of the modes under consideration. Figure 6-8 compares the multiple mode matrix and integral solution computed over the first three modes of our spatially incomplete beam with the analytic FRF. In figure 6-9 we compare a multiple mode matrix solution computed over the first three modes of a spatially complete beam using a single point partition at each of the modes with the beam's spatially complete analytic FRF. In figure 6-10 we compare a multiple mode integral solution computed over the first three modes of a spatially complete beam using a single point partition at each of the modes with the beam's spatially complete analytic FRF. Figure 6-11 compares multiple mode matrix and intgral solution computed over the first three modes of a spatially complete beam with the beam's spatially complete analytic FRF. Figures 6-9 and 6-10 show that the single point multiple mode solutions are exact.

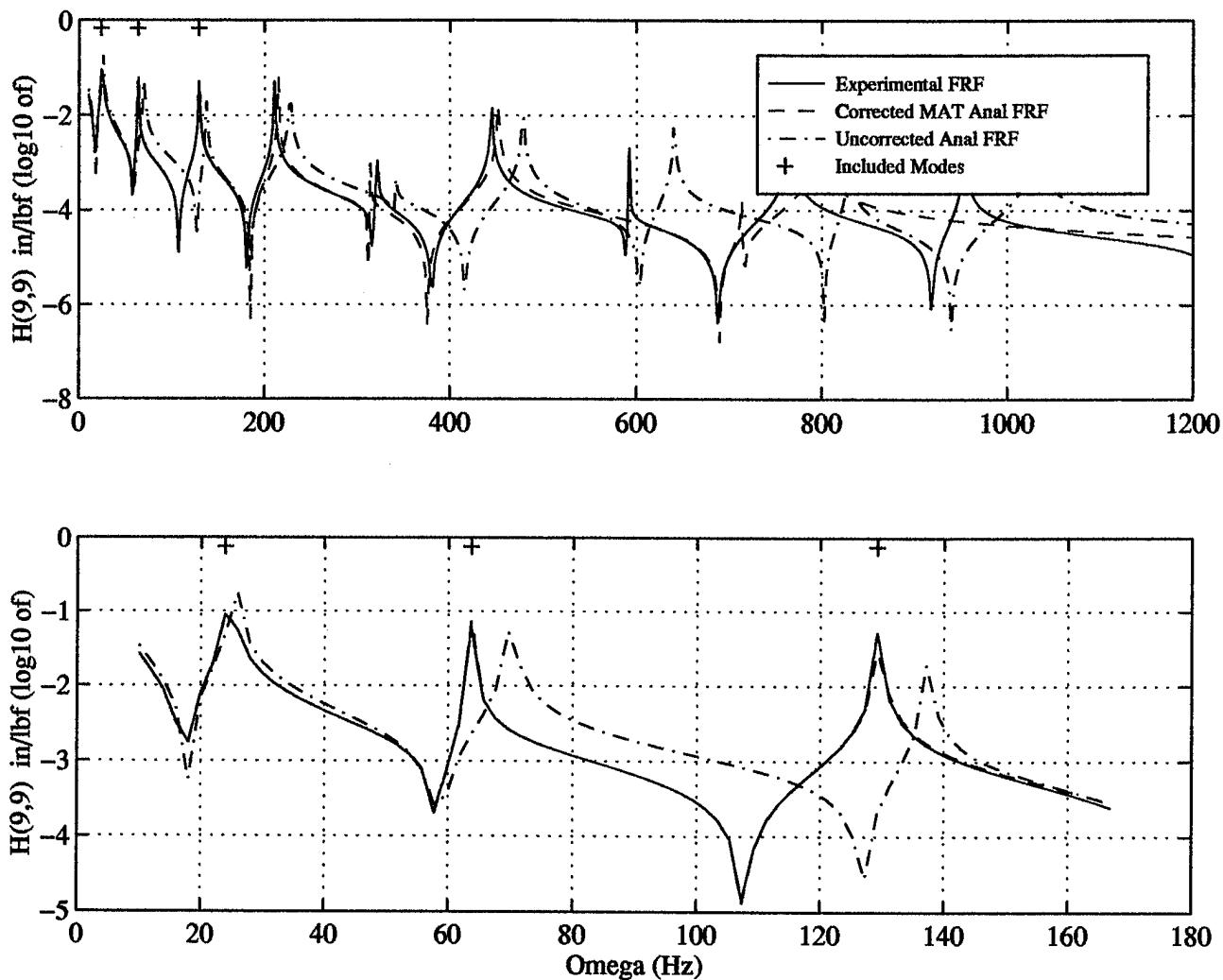


Figure 6- 6 Experimental FRF vs multiple mode matrix solution computed at modes 1 through 3 using a 1 Hz bandwidth with a single point frequency samplings at each mode for a spatially incomplete beam.

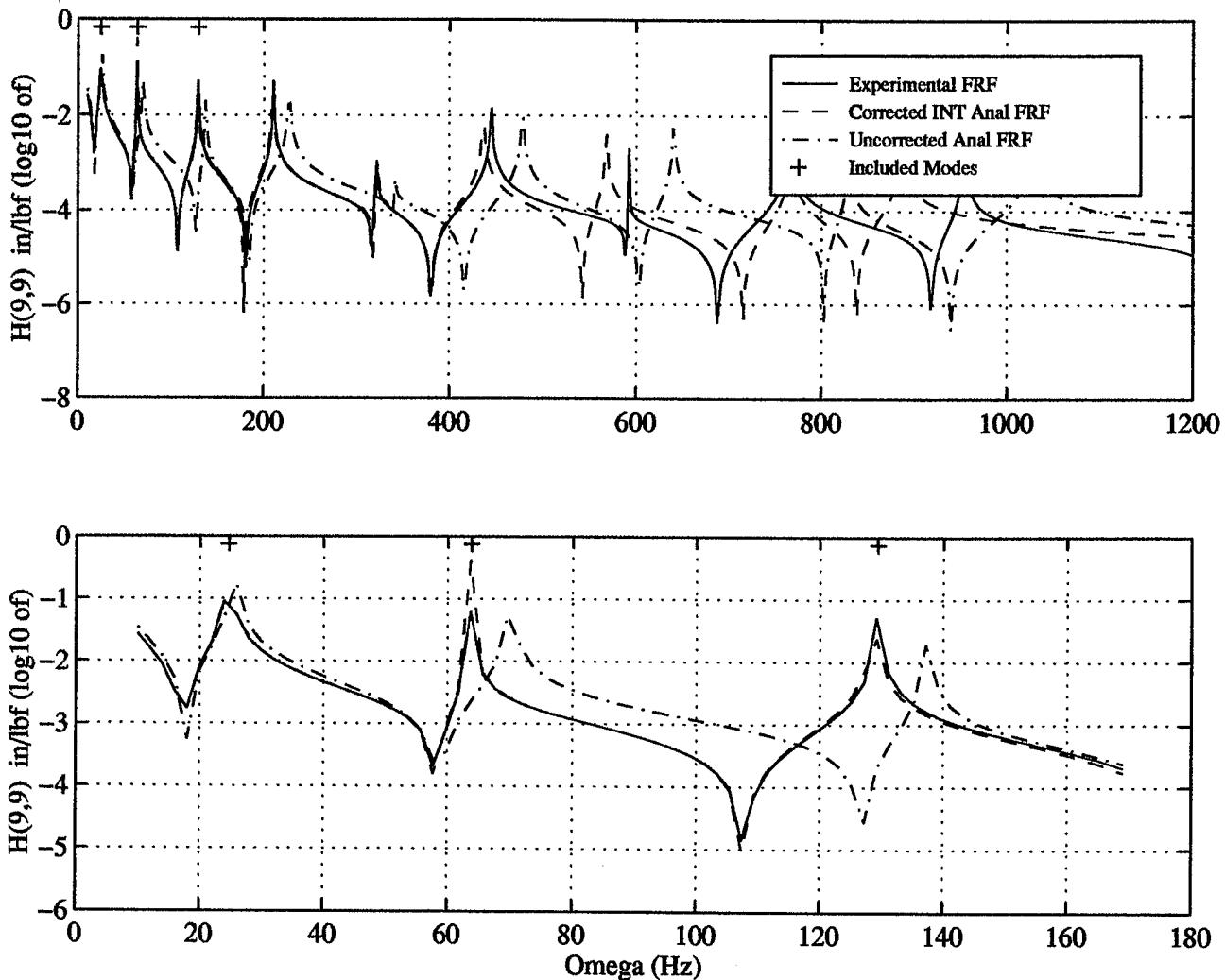


Figure 6- 7 Experimental FRF vs multiple mode integral solution computed at modes 1 through 3 using a 1 Hz bandwidth with a single point frequency samplings at each mode for a spatially incomplete beam.

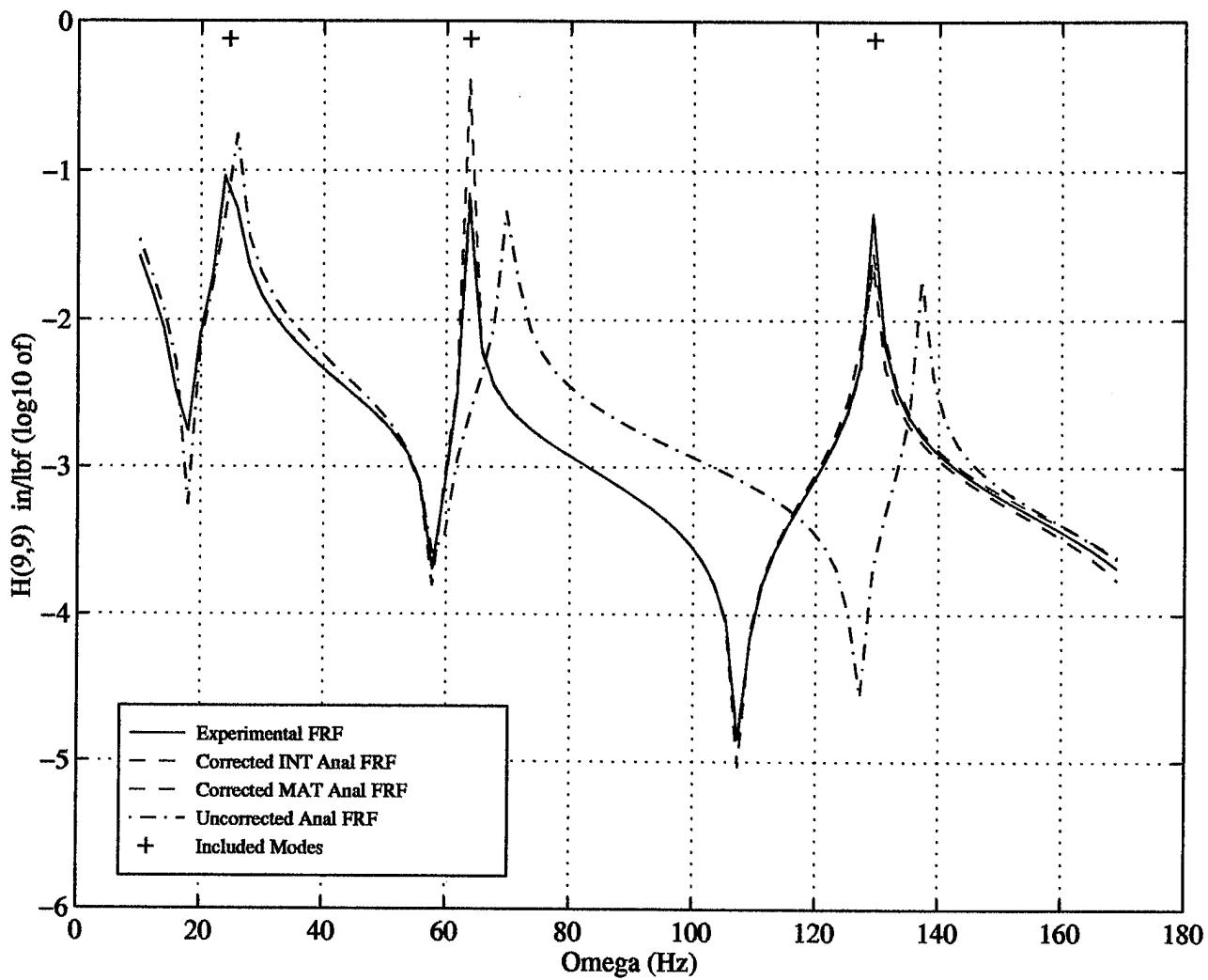


Figure 6- 8 Experimental FRF vs multiple mode matrix and integral solutions computed at modes 1 through 3 using a 1 Hz bandwidth with single point frequency samplings at each mode for a spatially incomplete beam.

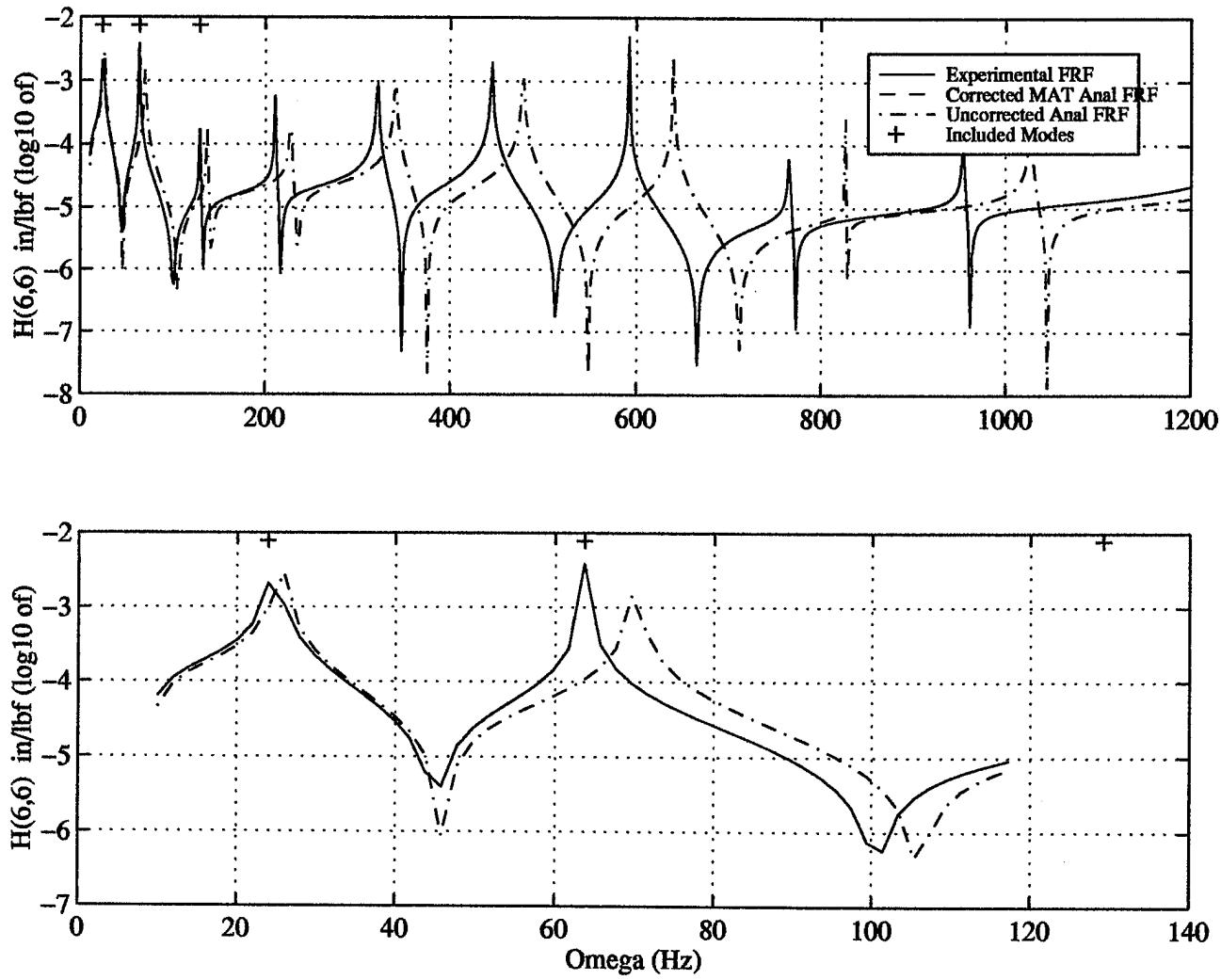


Figure 6- 9 Experimental FRF vs multiple mode matrix solution computed at modes 1 through 3 using a 1 Hz bandwidth with a single point frequency samplings at each mode for a spatially complete beam.

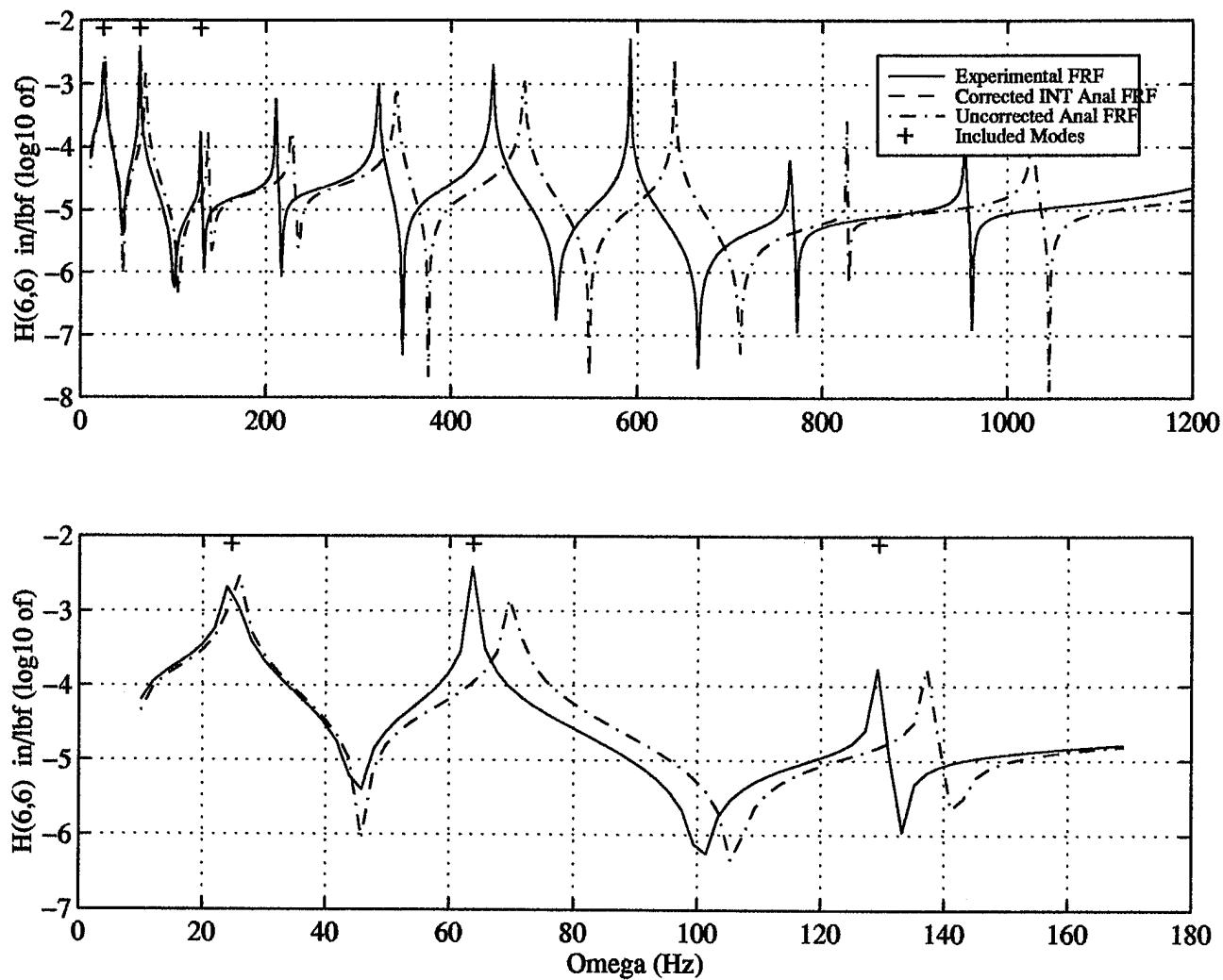


Figure 6- 10 Experimental FRF vs multiple mode integral solution computed at modes 1 through 3 using a 1 Hz bandwidth with a single point frequency samplings at each mode for a spatially complete beam.

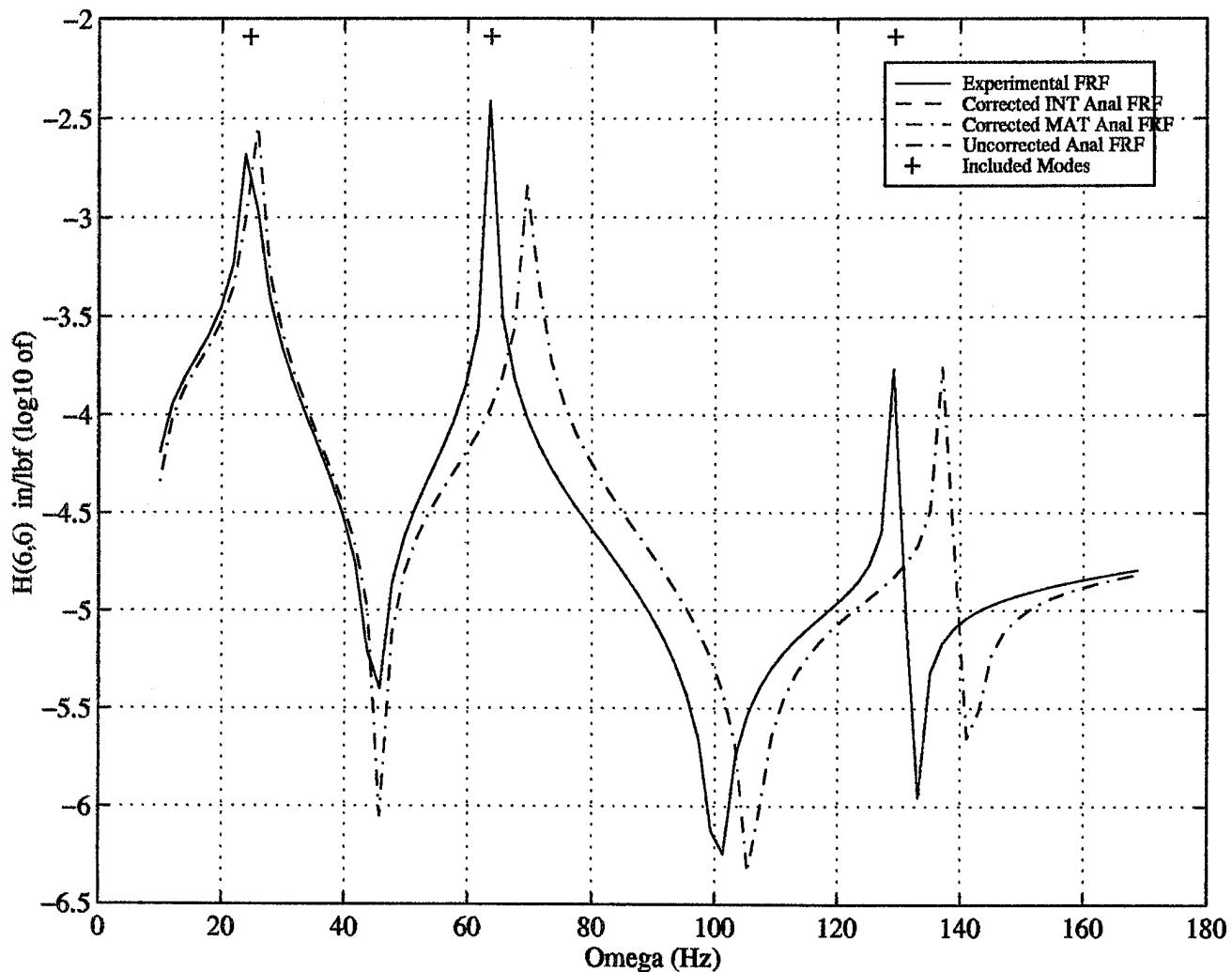


Figure 6- 11 Experimental FRF vs multiple mode matrix and integral solution computed at modes 1 through 3 using a 1 Hz bandwidth with a single point frequency samplings at each mode for a spatially complete beam.

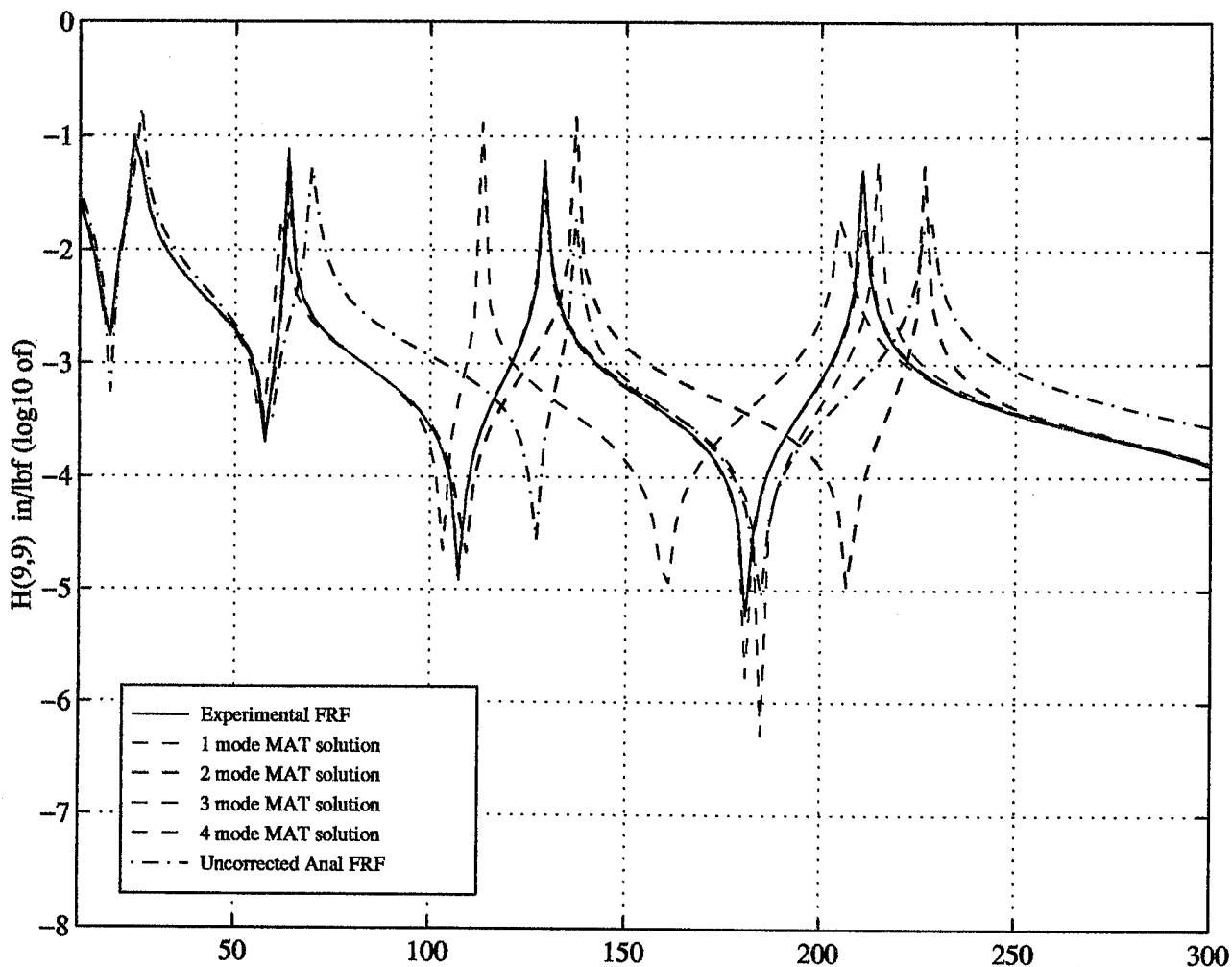


Figure 6- 12 Experimental FRF vs multiple mode matrix solutions computed over 1, 2, 3, and 4 modes using a 1 Hz bandwidth with single point frequency samplings at each mode for a spatially complete beam.

VII. CONCLUSIONS / RECOMMENDATIONS

A. SUMMARY

Frequency Domain Structural Identification using the structural synthesis transformation was performed on a simulated experimental free-free beam model. The identification was performed for both the spatially complete and the spatially incomplete case.

- SST based structural identification provides an exact solution for the identification of FE modeling errors given spatially complete data.
- For spatially incomplete systems SST based structural identification provides a frequency dependent solution which is not suitable for finite element modeling error correcting.
- Using both matrix and integral based techniques the SST can provide single mode solutions which are frequency independent correction matrices ΔK , ΔM , ΔC which approximately corrects the reduced FE model within a frequency bandwidth of the experimental system mode under consideration.

B. CONCLUSIONS

This thesis has clearly demonstrated that for spatially incomplete systems, single mode frequency independent solutions can be found which correct the reduced finite element model in a neighborhood of a given mode of the experimental system. It has also been shown that the concept of a single mode solution can be expanded to that of multiple mode solutions which are frequency independent correction matrices which approximately corrects the reduced FE model throughout a frequency bandwidth which includes more than 1 mode of the experimental system.

C. RECOMMENDATION

The purpose of this thesis was to find frequency independent multiple mode solutions which could be used to approximately correct reduced finite element models.

Although satisfactory results were obtained, investigation is still required in the following areas:

- Determine the relationship between errors in a full FE model and those of an associated reduced FE model.
- Determine a method to 'pullback' reduced FE model correction to full order FE model corrections.
- Investigate use of the integral formulation of reference (5) in analysis of the O-set system.

LIST OF REFERENCES

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4. Gordis, J.H., "Spatial, Frequency Domain Updating of Linear, Structural Dynamic Models", AIAA-93-1652-CP, 1993, pp. 3050-3058.
5. Hagood, W.H., Crawley, E.F., "Approximate Frequency Domain Analysis for Linear Damped Space Structures", Journal of the American Institute of Aeronautics and Astronautics, Vol. 28, Nov 1990, pp. 1953-1961.
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APPENDIX

The following is a brief description of MATLAB routines employed in this thesis:

- SST.M - Generates MAT files containing spatially complete or spatially incomplete experimental FRF, analytic FRF, localization matrix, and SST solutions.
- PLOTSST.M - Uses files produced by SST.M to generate plots used in chapters 3 and 4.
- CHAP3.M - Sets parameters and calls SST.M and PLOTSST.M to generate figures for chapter 3.
- CHAP4.M - Sets parameters and calls SST.M and PLOTSST.M to generate figures for chapter 4.
- CHAP5.M - Generate figures for chapter 5.
- CHAP6.M - Generate figures for chapter 6.
- SETUP.M - Generates MAT files for FE models.
- BEAMMDL.M - Setup FE models.
- FSTATIC.M - Perform static reduction of mass and stiffness matrices.
- FIRS_TAM.M - Perform IRS reduction of mass and stiffness matrices.
- FREQMODE.M - Returns FE model frequencies.
- NDX3D.M - Indexes a series of 2-D matrices into a single 3-D matrix.
- INTSUB.M - Decomposes impedance matrix into mass, stiffness, and damping matrices using integral techniques.
- MYTRAP.M - Computes integrals using trapezoidal method

SST.M

```

%%%%%General setup%%%%%General setup%%%%%General setup%%%%%
%%%%%clear workspace.
5 clear;clc      ;%%clear workspace.
closeall    ;%close any open figure windows
;
j=sqrt(-1)    ;
load sstconf  ;%load A-set and O-set
aset(ose)=[];;
10 aset_size=length(aset);
ose_size=length(ose);
load beamdata
if length(ose)== 0
    complete=1;
15 else
    complete=0;
end

%%setup plot labels%%%%%plot labels%%%%%plot labels%%%%%
20 if (ose==[])&(posofMasserr(1)~=0)&(posofStifferr(1)~=0)
    casename=(‘Spatially Complete Mass & Stiffness error’);
elseif (ose==[])&(posofMasserr(1)~=0)
    casename=(‘Spatially Complete Mass error’);
elseif (ose==[])&(posofStifferr(1)~=0)
25 casename=(‘Spatially Complete Stiffness error’);
elseif (posofMasserr(1)~=0)&(posofStifferr(1)~=0)
    casename=(‘Spatially InComplete Mass & Stiffness error’);
elseif (posofMasserr(1)~=0)
30 casename=(‘Spatially InComplete Mass error’);
elseif (posofStifferr(1)~=0)
    casename=(‘Spatially InComplete Stiffness error’);
end

%%get reduced A set & O set frequencies%%%%%%%%%%
35 kexto=k_anal(ose,ose); %% stiffness
mexto=m_anal(ose,ose); %% and mass matrices
if complete
    kstat=k_anal;
40 mstat=m_anal;
else
    if static == 0
        [kstat,mstat]=fstatic(k_anal,m_anal,ose,aset);%% get reduced K & M
    elseif static == 1
45     [kstat,mstat]=firs_tam(k_anal,m_anal,ose,aset);%% get reduced K & M
    else
        kstat=k_anal(aset,aset);
        mstat=m_anal(aset,aset);
    end
50 end
cstat=sqrt(-1)*struc_damping.*kstat;
[u,lambdaaa,c]=freqmode(k_anal,m_anal) ;%get freqs
[u,lambdaared,c]=freqmode(kstat,mstat);%
55 [u,lambdaax,c]=freqmode(k_exp,m_exp) ;%
[u,lambdaexto,c]=freqmode(kexto,mexto);%
omegaa=sqrt(lambdaaa);
omegax=sqrt(lambdaax);
omegared=sqrt(lambdaared);
omegaexto=sqrt(lambdaexto);
60 for i=1:4 % this to delete fixed body modes

```

SST.M

```

if real(omegax(1)) < 10^(-3)
    omegax=omegax(2:length(omegax));
end
65 if real(omegaa(1)) < 10^(-3)
    omegaaa=omegaaa(2:length(omegaaa));
end
if complete==0
    if real(omegared(1)) < 10^(-3)
        omegared=omegared(2:length(omegared));
    end
70    if real(omegaexto(1)) < 10^(-3)
        omegaexto=omegaexto(2:length(omegaexto));
    end
end
75 end

%%%%%find true mass and stiffness errors%%%%%
true_stiffness=k_exp-k_anal;
stiffness_cset=find(diag(true_stiffness));
80 true_mass=m_exp-m_anal;
mass_cset=find(diag(true_mass));
true_damping=c_exp-c_anal;
damping_cset=find(diag(true_damping));
if length(mass_cset) > 0
85    for i=1:length(mass_cset)
        x=find(stiffness_cset==mass_cset(i));
        if ~(length(x) > 0)
            stiffness_cset=[stiffness_cset; mass_cset(i)];
        end
90    end
end
if length(damping_cset) > 0
    for i=1:length(damping_cset)
        x=find(stiffness_cset==damping_cset(i));
        if ~(length(x) > 0)
95            stiffness_cset=[stiffness_cset; damping_cset(i)];
        end
    end
end
00 true_cset=sort(stiffness_cset)
save true_errs true_cset true_mass true_stiffness true_damping
clear true_mass true_stiffness true_damping
clear area eeee force g gc gcx gk gkx gm gmx goblc goblcx goblk goblkx
05 clear goblm goblmx ke lambdaa lambdaexto lambdaread lambdax lumpdamp
clear lumpmass lumpspring me pho posofDamperr posofMasserr posofStiffer
clear stiffness_cset

%%%%%setup frequency range for analysis%%%%%
freqtop=omegax(highmode)+1*omegax(highmode);
freqbottom=omegax(lowmode)-1*omegax(lowmode);
freqbottom=10*2*pi; %start at 10 hz
freqtop=1200*2*pi; %end at 1200 hz
%number of points to plot

5 numpoints=400;%SET NUMBER OF POINTS TO USE FOR THE SIMULATION
fineness=numpoints;

w=freqbottom:(freqtop-freqbottom)/(numpoints-1):freqtop;

0 %%Locate C set%%%%%

```

```

keepgoing='n'           ;
wfreq=omegax(lowmode) ;
nextnode=lowmode;
tol=1                  ;
125 defaultstepsize=.1;
while (keepgoing=='n')
    z_anal_red=kstat+j*wfreq*cstat-wfreq^2*instat ;
    h_anal_red=inv(z_anal_red)          ;
    z_exp=k_exp+j*wfreq*c_exp-wfreq^2*m_exp ;
130    h_exp=inv(z_exp)          ;
    h_exp=h_exp(aset,aset)          ;
    L=z_anal_red*(h_anal_red-h_exp)*z_anal_red      ;
    ci=find(abs(diag(L))>tol)          ;
    temp_cset_size=length(ci);
135    if length(ci) >= 1
        cset_size=length(ci)          ;
        figure(1)
        plot(aset, abs(diag(L)))      ;
        hold on
140    plot(aset(ci),zeros(1,length(ci)), 'x')
        tallest=max(abs(diag(L)))      ;
        title(['TOL = ',num2str(tol),' Cset size = ',int2str(cset_size),' Omega = ',num2str(wfreq/(2*pi)), ' Hz'])
        ylabel('lbf/in')
        if complete
            xlabel('DOF')
        else
            xlabel('ASET DOF')
        end
        hold off
150    k=menu( 'Choose an action',...
        'Increase Tolerance',...
        'Decrease Tolerance',...
        'Increase frequency',...
        'Decrease frequency',...
155    ' Previous mode',...
        ' Next Mode',...
        'Set Print switch on',...
        'Increase stepsize',...
        'Decrease stepsize',...
        ' Go      ');
160    if k==1
        stepsize=defaultstepsize;
        ctr=0;
        while length(ci) >= temp_cset_size
            ctr=ctr+1;
            if ctr==10
                stepsize=stepsize*10;
                ctr=0;
            end
170    tol=min(abs(tol+stepsize*(tol)),tallest) ;
        ci=find(abs(diag(L))>tol)          ;
        if length(ci) >= 1
            cset_size=length(ci)          ;
            figure(1)
            plot(aset, abs(diag(L)))      ;
            hold on
            plot(aset(ci),zeros(1,length(ci)), 'x')
            tallest=max(abs(diag(L)))      ;
            title(['TOL = ',num2str(tol),' Cset size = ',int2str(cset_size),' Omega = ',num2str(wfreq/(2*pi)), ' Hz'])
            ylabel('lbf/in')
175    end
180    end

```

SST.M

```

if complete
    xlabel('DOF')
else
    xlabel('ASET DOF')
end
hold off
end
end
elseif k==2
    stepsize=defaultstepsize;
    ctr=0;
    while length(ci) <= temp_cset_size
        ctr=ctr+1;
        if ctr==10;
            stepsize=stepsize*2;
            ctr=0;
        end
        tol=max(abs(tol-stepsize*(tol)),.00001);
        ci=find(abs(diag(L))>tol) ;
        if length(ci) >= 1
            cset_size=length(ci) ;
            figure(1)
            plot(asset, abs(diag(L))) ;
            hold on
            plot(asset(ci),zeros(1,length(ci)),'x')
            tallest=max(abs(diag(L))) ;
            title(['TOL = ',num2str(tol),' Cset size = ',int2str(cset_size),' Omega = ',num2str(wfreq/(2*pi)),', Hz']) ;
            ylabel('lbf/in')
            if complete
                xlabel('DOF')
            else
                xlabel('ASET DOF')
            end
            hold off
        end
    end
elseif k==3
    while length(ci) == temp_cset_size
        wfreq=min(wfreq+.00001*(freqtop-wfreq),freqtop);
        z_anal_red=kstat+j*w(count)*cstat-wfreq^2*mstat ;
        h_anal_red=inv(z_anal_red) ;
        z_exp=k_exp+j*wfreq*c_exp-wfreq^2*m_exp ;
        h_exp=inv(z_exp) ;
        h_exp=h_exp(asset,asset) ;
        L=z_anal_red*(h_anal_red-h_exp)*z_anal_red ;
        tallest=max(abs(diag(L))) ;
        tol=min(tol+.001*tallest,tallest) ;
        ci=find(abs(diag(L))>tol) ;
        cset_size=length(ci) ;
        figure(1)
        plot(asset, abs(diag(L))) ;
        hold on
        plot(asset(ci),zeros(1,length(ci)),'x')
        title(['TOL = ',num2str(tol),' Cset size = ',int2str(cset_size),' Omega = ',num2str(wfreq/(2*pi)),', Hz']) ;
        ylabel('lbf/in')
        if complete
            xlabel('DOF')
        else
            xlabel('ASET DOF')
        end
    end

```

SST.M

```

        end
        hold off
    end
    elseif k==4
        wfreq=max(wfreq-.05*(wfreq-freqbottom),freqbottom);
    elseif k==5
        nextmode=max(lowmode,nextmode-1);
        wfreq=omegax(nextmode);
    elseif k==6
        nextmode=min(hightmode,nextmode+1);
        wfreq=omegax(nextmode);
    elseif k==7
        pswitch='y' ;
    elseif k==8
        defaultstepsize=defaultstepsize/10;
    elseif k==9
        defaultstepsize=defaultstepsize*10;
    else
        keepgoing='y' ;
        cset=aset(ci);
        cset_rel=ci ;
    end
    else
        tol=tol*.9;
    end
end

close(1)
save extriset cset cset_rel
%%%%%FORCE THE CSET%%%%%
%cset=[7 9 11 13] %%%%%THE INCOMPLETE CASE CSET%%%%%
%cset_rel=[4 5 6 7]%%%%%
275 %%%%%%
%cset=[4 5 6 7 8 9 10 11]%%%%%THE COMPLETE CASE CSET%%%%%
%cset_rel=[4 5 6 7 8 9 10 11]%%%%%
%%%%%
280 cset_size=length(cset);
save L L
clear L
clear kexta kexto mexta mexto
285 pack
skyfull=zeros(numdof,numdof);
count=1 ;
for index1=1:numdof
    for index2=index1:numdof
        skyfull(index1,index2)=count;
        skyfull(index2,index1)=count;
        count=count+1;
    end
end
295 skyred=zeros(aset_size,aset_size);
count=1 ;
for index1=1:aset_size
    for index2=index1:aset_size
        skyred(index1,index2)=count;
        skyred(index2,index1)=count;
    end
end
300

```

SST.M

```

        count=count+1;
    end
end
skycset=zeros(cset_size,cset_size);
305 count=1 ;
for index1=1:cset_size
    for index2=index1:cset_size
        skycset(index1,index2)=count;
        skycset(index2,index1)=count;
310        count=count+1;
    end
end

full_holder=zeros(1,(numdof*(numdof+1))/2);
red_holder=zeros(1,(aset_size*(aset_size+1))/2);
cset_holder=zeros(1,(cset_size*(cset_size+1))/2);

if meters
    waitbar_handle=waitbar(0,'Computing Experimental FRF');
320 else
    disp('Getting experimental FRF')
end
H_EXP=[];
for count=1:numpoints
325 if meters
    waitbar(count/numpoints);
end
z_exp=k_exp+j*w(count)*c_exp-w(count)^2*m_exp;
h_exp=inv(z_exp);
330 h_exp=h_exp(aset,aset);
skyindex=1;
for index1=1:aset_size
    for index2=index1:aset_size
335        red_holder(skyindex)=h_exp(index1,index2);
        skyindex=skyindex+1;
    end
end
H_EXP=[H_EXP red_holder];
340 end
if meters
    close(waitbar_handle)
end
save H_EXP H_EXP
clear H_EXP
345 pack

if meters
    waitbar_handle=waitbar(0,'Computing Experimental Impedance');
else
350    disp('Getting experimental Impedance')
end
Z_EXP=[];
for count=1:numpoints
    if meters
55        waitbar(count/numpoints);
    end
    z_exp=k_exp+j*w(count)*c_exp-w(count)^2*m_exp;
    h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
60    z_exp=inv(h_exp);

```

```

skyindex=1 ;
for index1=1:aset_size
    for index2=index1:aset_size
        red_holder(skyindex)=z_exp(index1,index2);
        skyindex=skyindex+1;
    end
end
Z_EXP=[Z_EXP red_holder];
end
365 if meters
    close(waitbar_handle)
end
370 save Z_EXP Z_EXP
clear Z_EXP
375 pack

if meters
    waitbar_handle=waitbar(0,'Computing Reduced Analytic FRF');
else
380 disp('Getting Reduced FRF')
end
H_ANAL_RED=[];
for count=1:numpoints
    if meters
385         waitbar(count/numpoints);
    end
    z_anal_red=kstat+j*w(count)*cstat-w(count)^2*mstat;
    h_anal_red=inv(z_anal_red);
    skyindex=1 ;
390    for index1=1:aset_size
        for index2=index1:aset_size
            red_holder(skyindex)=h_anal_red(index1,index2);
            skyindex=skyindex+1;
        end
    end
395    end
    H_ANAL_RED=[H_ANAL_RED red_holder];
end
if meters
    close(waitbar_handle)
end
400 save H_ANAL_R H_ANAL_RED
clear Z_ANAL_RED H_ANAL_RED kexta kexto mexta mexto
pack

405 %% compute and save L matrix diagonal as a function of frequency%%%%%%%
if meters
    waitbar_handle=waitbar(0,'Computing L Diagonals');
else
    disp('Getting L Matrix')
end
410 L_DIAGS=[];
for count=1:numpoints
    if meters
        waitbar(count/numpoints);
    end
    z_anal=k_anal+j*w(count)*c_anal-w(count)^2*m_anal;
    z_anal_red=kstat+j*w(count)*cstat-w(count)^2*mstat;
    h_anal_red=inv(z_anal_red);
    z_exp=k_exp+j*c_exp-w(count)^2*m_exp;
415    h_exp=inv(z_exp);
420

```

SST.M

```

425      h_exp=h_exp(asset,asset);
        L=z_anal_red*(h_anal_red-h_exp)*z_anal_red;
        temp_1_diags=diag(L);
        L_DIAGS=[L_DIAGS temp_1_diags(:)];
    end
    if meters
        close(waitbar_handle)
    end
430    save L_DIAGS L_DIAGS
    clear L_DIAGS z_anal temp_1_diags L
    clear z_exp z_anal_red u v h_exp h_anal_red
    pack

    if meters
435        waitbar_handle=waitbar(0,'Computing Z Analytic');
    else
        disp('Getting Z anal')
    end
    Z_ANAL=[];
440    for count=1:numpoints
        if meters
            waitbar(count/numpoints);
        end
        z_anal=k_anal+j*w(count)*c_anal-w(count)^2*m_anal;
445        skyindex=1 ;
        for index1=1:numdof
            for index2=index1:numdof
                full_holder(skyindex)=z_anal(index1,index2);
                skyindex=skyindex+1;
450        end
        end
        Z_ANAL=[Z_ANAL full_holder];
    end
    if meters
455        close(waitbar_handle)
    end
    save Z_ANAL Z_ANAL
    clear Z_ANAL z_anal
    pack
460    if meters
        waitbar_handle=waitbar(0,'Computing Reduced Z Analytic');
    else
        disp('Getting Reduced Z anal')
    end
465    Z_ANAL_RED=[];
    for count=1:numpoints
        if meters
            waitbar(count/numpoints);
        end
470    z_anal_red=kstat+j*w(count)*cstat-w(count)^2*mstat;
        skyindex=1 ;
        for index1=1:asset_size
            for index2=index1:asset_size
                red_holder(skyindex)=z_anal_red(index1,index2);
                skyindex=skyindex+1;
375        end
        end
        Z_ANAL_RED=[Z_ANAL_RED red_holder];
    end
    if meters
380

```

```

    close(waitbar_handle)
end
save Z_ANAL_R Z_ANAL_RED
clear Z_ANAL_RED z_anal_red
pack

485

%%%compute DZ%%%%%%%%%%%%%%%
if meters
    waitbar_handle=waitbar(0,'Computing DZ...');

490 else
    disp('Getting DZ')
end
DZ=[];
for count=1:numpoints
495 if meters
    waitbar(count/numpoints);
end
z_anal_red=kstat+j*w(count)*cstat-w(count)^2*mstat;
h_anal_red=inv(z_anal_red);
500 z_exp=k_exp+j*w(count)*c_exp-w(count)^2*m_exp;
h_exp=inv(z_exp);
h_exp=h_exp(asset,asset);
hacc=h_anal_red(cset_rel,cset_rel);
hxcc=h_exp(cset_rel,cset_rel);
505 dz=inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc;
skyindex=1 ;
for index1=1:cset_size
    for index2=index1:cset_size
        cset_holder(skyindex)=dz(index1,index2);
510 skyindex=skyindex+1;
    end
end
DZ=[DZ cset_holder'];
end
515 if meters
    close(waitbar_handle)
end
save DZ DZ
clear hacc hxcc dz h_exp z_anal_red h_anal_red z_exp
520 pack

%%%decompose DZ into DM, DK, & DC%%%%%%%%%%%%%%
if meters
    waitbar_handle=waitbar(0,'Decomposing DZ...');

525 else
    disp('Decomposing DZ')
end
DK=[];
DM=[];
DC=[];
530 dz1=zeros(cset_size,cset_size);
dz2=zeros(cset_size,cset_size);
dz3=zeros(cset_size,cset_size);
for i=1:numpoints-2
535 if meters
    waitbar(i/(numpoints-2))
end
dztemp=DZ(ndx3d([1 cset_size*(cset_size+1)/2 numpoints],1,1:cset_size*(cset_size+1)/2,i));
skyindex=1;
540 for index1=1:cset_size

```

SST.M

```

for index2=index1:cset_size
    dz1(index1,index2)=dztemp(skyindex);
    dz1(index2,index1)=dztemp(skyindex);
    skyindex=skyindex+1;
545 end
dztemp=DZ(ndx3d([1 cset_size*(cset_size+1)/2 numpoints],1,1:cset_size*(cset_size+1)/2,i+1));
skyindex=1;
for index1=1:cset_size
550     for index2=index1:cset_size
        dz2(index1,index2)=dztemp(skyindex);
        dz2(index2,index1)=dztemp(skyindex);
        skyindex=skyindex+1;
    end
555 end
dztemp=DZ(ndx3d([1 cset_size*(cset_size+1)/2 numpoints],1,1:cset_size*(cset_size+1)/2,i+2));
skyindex=1;
for index1=1:cset_size
560     for index2=index1:cset_size
        dz3(index1,index2)=dztemp(skyindex);
        dz3(index2,index1)=dztemp(skyindex);
        skyindex=skyindex+1;
    end
565 end
temp=[eye(cset_size)-w(i)^2*eye(cset_size) +j*w(i)*eye(cset_size);
eye(cset_size)-w(i+1)^2*eye(cset_size) +j*w(i+1)*eye(cset_size);
eye(cset_size)-w(i+2)^2*eye(cset_size) +j*w(i+2)*eye(cset_size)]\dz1;
ktemp=temp(1:cset_size,:);
570 mtemp=temp(cset_size+1:2*cset_size,:);
ctemp=temp(2*cset_size+1:3*cset_size,:);

skyindex=1 ;
for index1=1:cset_size
575     for index2=index1:cset_size
        cset_holder(skyindex)=ktemp(index1,index2);
        skyindex=skyindex+1;
    end
end
580 DK=[DK cset_holder(:)];

skyindex=1 ;
for index1=1:cset_size
585     for index2=index1:cset_size
        cset_holder(skyindex)=mtemp(index1,index2);
        skyindex=skyindex+1;
    end
end
590 DM=[DM cset_holder(:)];

skyindex=1 ;
for index1=1:cset_size
595     for index2=index1:cset_size
        cset_holder(skyindex)=ctemp(index1,index2);
        skyindex=skyindex+1;
    end
end
DC=[DC cset_holder(:)];
end
if meters
    close(waitbar_handle)
00

```

SST.M

```

end
save DM DM
save DK DK
save DC DC
605 clear DC DK DM
save exp c_exp k_exp m_exp
clear c_exp k_exp m_exp
save stat kstat mstat cstat
clear kstat mstat cstat
610 save anal k_anal m_anal c_anal
clear k_anal m_anal c_anal
pack

if complete
615 load Z_ANAL
Z=Z_ANAL;
clear Z_ANAL
else
load Z_ANAL_R
620 Z=Z_ANAL_RED;
clear Z_ANAL_RED
end
CORRHA=[];
tempza=zeros(asset_size,asset_size);
625 tempz=zeros(cset_size,cset_size);
if meters
    waitbar_handle=waitbar(0,'Installing DZ...');
else
    disp('Installing DZ')
630 end
for i=1:numpoints
if meters
    waitbar(i/numpoints)
end
635 ztemp=Z(ndx3d([1 asset_size*(asset_size+1)/2 (numpoints-(i-1))],1,1:asset_size*(asset_size+1)/2,1));
skyindex=1;
for index1=1:asset_size
    for index2=index1:asset_size
        tempza(index1,index2)=ztemp(skyindex);
        tempza(index2,index1)=ztemp(skyindex);
        skyindex=skyindex+1;
    end
end
640 Z(:,1)=[];
ztemp=DZ(ndx3d([1 cset_size*(cset_size+1)/2 (numpoints-(i-1))],1,1:cset_size*(cset_size+1)/2,1));
skyindex=1;
for index1=1:cset_size
    for index2=index1:cset_size
        tempz(index1,index2)=ztemp(skyindex);
        tempz(index2,index1)=ztemp(skyindex);
        skyindex=skyindex+1;
    end
end
645 tempza(cset_rel,cset_rel)=tempza(cset_rel,cset_rel)+tempz;
DZ(:,1)=[];
tempha=inv(tempza);
CORRHA=[CORRHA tempha(:)];
650 end
if meters
close(waitbar_handle)
655

```

SST.M

```

end
clear DZ Z
save CORRHA CORRHA
clear CORRHA
665
load DK;
load DM;
load DC;

670 load stat
CORRHAD=[];
tempza=zeros(aset_size,aset_size);
tempz=zeros(cset_size,cset_size);
tempk=zeros(cset_size,cset_size);
675 tempm=zeros(cset_size,cset_size);
tempc=zeros(cset_size,cset_size);
if meters
    waitbar_handle=waitbar(0,'Installing DK, DM, & DC...');
else
    disp('Installing DK, DM, DC')
end
for i=1:numpoints-2
    if meters
        waitbar(i/(numpoints-2))
    end
    kcorrected=kstat;
    mcorrected=mstat;
    ccorrected=cstat;
685
temp=DK(ndx3d([1 cset_size*(cset_size+1)/2 (numpoints-(2))],1,1:cset_size*(cset_size+1)/2,1));
skyindex=1;
for index1=1:cset_size
    for index2=index1:cset_size
        tempk(index1,index2)=temp(skyindex);
        tempk(index2,index1)=temp(skyindex);
        skyindex=skyindex+1;
    end
end
700 DK(:,1)=[];
temp=DM(ndx3d([1 cset_size*(cset_size+1)/2 (numpoints-(2))],1,1:cset_size*(cset_size+1)/2,1));
skyindex=1;
for index1=1:cset_size
    for index2=index1:cset_size
        tempm(index1,index2)=temp(skyindex);
        tempm(index2,index1)=temp(skyindex);
        skyindex=skyindex+1;
    end
end
705 DM(:,1)=[];
temp=DC(ndx3d([1 cset_size*(cset_size+1)/2 (numpoints-(2))],1,1:cset_size*(cset_size+1)/2,1));
skyindex=1;
for index1=1:cset_size
    for index2=index1:cset_size
        tempc(index1,index2)=temp(skyindex);
        tempc(index2,index1)=temp(skyindex);
        skyindex=skyindex+1;
    end
end
710
end

```

SST.M

```
DC(:,1)=[];  
kcorrected(cset_rel,cset_rel)=kcorrected(cset_rel,cset_rel)+tempk;  
mcorrected(cset_rel,cset_rel)=mcorrected(cset_rel,cset_rel)+tempm;  
ccorrected(cset_rel,cset_rel)=ccorrected(cset_rel,cset_rel)+tempc;  
725  
tempza=kcorrected+j*w(i+1)*ccorrected-w(i+1)^2*mcorrected;  
tempfa=inv(tempza);  
CORRHAD=[CORRHAD tempfa(:)];  
end  
730 if meters  
close(waitbar_handle)  
end  
clear tempk tempm tempc tempza tempfa  
clear kcorrected mcorrected ccorrected  
735 save CORRHAD CORRHAD  
clear CORRHAD DK DM DC  
  
%%%%%save the  
Workspace%%%%%  
740 save INT
```

745

PLOTSST.M

```

clear
closeall
load INT
titles=0;
5 if pswitch=='y'
    whitebg('white')
    close
end
h=0;
10 if complete
    fignum=1;
else
    fignum=21;
end
15 %% write frequencies and system diag to diary file prt.out%%%%%%%%%%%%%
if exist('fig000.out')
    delete fig000.out
end
diary fig000.out
20 fprintf(' \n')
fprintf(' \n')
fprintf('A set\n')
for index=1:length(aset)
    fprintf([blanks(4-length(int2str(aset(index))))],int2str(aset(index))])
25 if rem(index,12)==0
    fprintf('\n')
end
end
fprintf('\n')
30 fprintf('\n')
fprintf('O set\n')
for index=1:length(oset)
    fprintf([blanks(4-length(int2str(oset(index))))],int2str(oset(index))])
35 if rem(index,12)==0
    fprintf('\n')
end
end
fprintf('\n')
fprintf('\n')
40 fprintf('Computed C set\n')
for index=1:length(cset)
    fprintf([blanks(4-length(int2str(cset(index))))],int2str(cset(index))])
45 if rem(index,12)==0
    fprintf('\n')
end
end
fprintf('\n')
fprintf('\n')
50 fprintf('True C set\n')
for index=1:length(true_cset)
    fprintf([blanks(4-length(int2str(true_cset(index))))],int2str(true_cset(index))])
55 if rem(index,12)==0
    fprintf('\n')
end
end
fprintf('\n')
fprintf('\n')
bigger=max(length(omegaa),length(omegax));
if complete
    allfreqs=[ ...

```

PLOTSST.M

```

[omegaa/(2*pi); zeros(bigger-length(omegaa),1)] ...
[omegax/(2*pi); zeros(bigger-length(omegax),1)] ...
];
else
allfreqs=[ ...
65 [omegaa/(2*pi); zeros(bigger-length(omegaa),1) ]...
[omegax/(2*pi); zeros(bigger-length(omegax),1) ]...
[omegared/(2*pi); zeros(bigger-length(omegared),1) ]...
[omegaexto/(2*pi); zeros(bigger-length(omegaexto),1)]...
];
end
[aaa bbb]=size(allfreqs);
sprintf('System Frequencies (Hz)\n')
if complete
75 sprintf(' Anal Exp\n')
bbb=2;
else
    sprintf(' Anal Exp Red Oset\n')
    bbb=4;
80 end
for index1=1:aaa
prtline=[];
for index2=1:bbb
if allfreqs(index1,index2)==0
85 prtfreq='      ';
else
    prtfreq=[blanks(8-length(sprintf('%4g',allfreqs(index1,index2)))),...
    sprintf('%4g',allfreqs(index1,index2))];
end
90 prtline=[prtline, prtfreq];
end
prtline=[prtline, '\n'];
fprintf(prtline)
end
95 diary off
%dos('type fig000.out > lpt1 &');
xx=omegax(find(omegaa(lowmode) <= omegax & omegax <= omegaa(highmode))/(2*pi);
yx1=ones(length(xx),1);
aa=omegaa(find(omegaa(lowmode) <= omegaa & omegaa <= omegaa(highmode))/(2*pi);
ya1=ones(length(aa),1);
100 red=omegared(find(omegaa(lowmode) <= omegared & omegared <= omegaa(highmode))/(2*pi);
yred1=ones(length(red),1);
exto=omegaexto(find(omegaa(lowmode) <= omegaexto & omegaexto <= omegaa(highmode))/(2*pi);
yexto1=ones(length(exto),1);
105 if complete
x0=max([xx(length(xx)) aa(length(aa))]);
else
x0=max([xx(length(xx)) aa(length(aa)) red(length(red)) exto(length(exto))]);
end
110 clear omegax omegaa omegared omegaexto
clear L Z_ANAL c_anal c_exp conn h_exp h_anal_red k_anal k_exp
clear kexta kstat kexto m_anal m_exp mexto instat temp_l_diags
if complete
mid_index=round(length(cset)/2);
115 else
mid_index=round(length(cset)/2);
end
load H_ANAL_R
load H_EXP
120

```

PLOTSST.M

```

h=figure(h+1);
fignum=fignum+1 ;
plot(w/(2*pi),log10(abs(H_ANAL_RED(ndx3d([1 asset_size*(asset_size+1)/2 numpoints],...
1,skyred(cset_rel(mid_index),cset_rel(mid_index),':'))),r-',...
w/(2*pi),log10(abs(H_EXP(ndx3d([1 asset_size*(asset_size+1)/2 numpoints],...
1,skyred(cset_rel(mid_index),cset_rel(mid_index),':'))),b-')
v=axis;
ylabel(['H(' int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/bf(Log10 of)']);
xlabel('Omega (Hz)');
if titles
    title('Analytic FRF vs Experimental FRF')
end
hold on
if abs(v(4)-v(3)) <= 1
    v2(1)=v(1);
    v2(2)=v(2);
    v2(3)=v(3)-.1*abs(v(4));
    v2(4)=v(4)+.1*abs(v(4));
    axis(v2);
end
grid on
hh=slegend('ANALYTIC','EXPERIMENTAL');
axes(hh);
hold off
if pswitch=='y'
    prtfig(fignum)
    delete(h)
end
clear H_ANAL_RED
clear H_EXP

if complete
    h=figure(h+1);
    fignum=fignum+1;
else
    h=figure(h+2);
    fignum=fignum+2;
end
load L
plot(asset,abs(diag(L)),asset,abs(diag(L)),'*')
ylabel(' L(DOF) lbf/in')
if complete
    xlabel('DOF')
else
    xlabel('ASET DOF')
end
grid on
if titles
    title(['Localization Matrix Diagonal at Omega = ',num2str(wfreq/(2*pi)), ' Hz']);
end
if pswitch=='y'
    prtfig(fignum)
    delete(h)
end
clear L
h=figure(h+1);
fignum=fignum+1;
load L_DIAGS

```

PLOTSST.M

```

mesh(w/(2*pi),aset,log10(abs(L_DIAGS)))
if titles
    title('Frequency Dependence of Localization Matrix Diagonals')
end
185 xlabel('Omega (Hz)')
ylabel('DOF')
zlabel('L(DOF, omega) lbf/in (Log10 of)')
grid on
if pswitch=='y'
190    prtfig(signum)
    delete(h)
end

h=figure(h+1);
195 signum=signum+1;
subplot(2,1,1)
plot(w/(2*pi),log10(L_DIAGS(ndx3d([aset_size 1 numpoints],cset_rel(mid_index+2),1,:))))
ylabel(['Error Coord L(',int2str(cset(mid_index+2)),',';int2str(cset(mid_index+2)),')'])
if titles
200    title('lbf/in (Log10 of)');
end
grid on
v1=axis;
subplot(2,1,2)
205 if length(cset_rel) < length(aset)
    holdset=1:length(aset) ;
    holdset(cset_rel)=[] ;
    index=round(length(holdset)/2);
210 plot(w/(2*pi),log10(L_DIAGS(ndx3d([aset_size 1 numpoints],holdset(index),1,:))))
ylabel(['Non-Error Coord L(',int2str(aset(holdset(index))),',';int2str(aset(holdset(index))),')'])
if titles
    title('Frequency Dependence of L Matrix Non-Error set Diagonal Elements' )
215 end
v=axis;
hold on
v2(1)=v(1);
v2(2)=v(2);
220 v2(3)=v(3)+((v(4)-v(3))*5)-(v1(4)-v1(3))*5 ;
v2(4)=v(3)+((v(4)-v(3))*5)+(v1(4)-v1(3))*5;
axis(v2);
grid on
end
225 xlabel('Omega (Hz)')
hold off
if pswitch=='y'
    prtfig(signum)
    delete(h)
230 end
clear L_DIAGS

load Z_ANAL_R
load Z_EXP
235 h=figure(h+1);
signum=signum+1;
plot(w/(2*pi),log10(Z_ANAL_RED(ndx3d([1 aset_size*(aset_size+1)/2
numpoints],1,skyred(cset_rel(mid_index),cset_rel(mid_index)),':')),'r-',...
w/(2*pi),log10(Z_EXP(ndx3d([1 aset_size*(aset_size+1)/2
240 numpoints],1,skyred(cset_rel(mid_index),cset_rel(mid_index)),':')),'b-.')

```

PLOTSST.M

```

xlabel('Omega^2 (Hz)')
ylabel(['Z(',int2str(cset(mid_index)),',';int2str(cset(mid_index)),') lbf/in (Log10 of)'])
if titles
    title('Analytic Impedance vs Experimental Impedance')
end
245 v=axis;
hold on
if abs(v(4)-v(3)) <= 1
    v2(1)=v(1);
    v2(2)=v(2);
    v2(3)=v(3)-.1*abs(v(4));
    v2(4)=v(4)+.1*abs(v(4));
    axis(v2);
end
250 grid on
if complete== 0
    hh=legend('ANALYTIC IMPEDANCE','EXPERIMENTAL IMPEDANCE');
else
    hh=legend('ANALYTIC IMPEDANCE','EXPERIMENTAL IMPEDANCE');
end
255 axes(hh)
hold off
if pswitch=='y'
    prtfi(figurenum)
    delete(h)
end
260 clear Z_ANAL_RED Z_EXP

load true_errs
270 load DK
h=figure(h+1);
figurenum=figurenum+1;
if complete
    plot(w(1:numpoints-2)/(2*pi),DK(ndx3d([1 cset_size*(cset_size+1)/2 numpoints-2],1,skyCset(mid_index,mid_index),'')),r-
275 ,...
    w(1:numpoints-2)/(2*pi),true_stiffness(cset(mid_index),cset(mid_index)).*ones(1,numpoints-2),'b-')
    ylabel(['DK(',int2str(cset(mid_index)),',';int2str(cset(mid_index)),') lbf/in'])
    xlabel('Omega (Hz)')
else
    plot(w(1:numpoints-2)/(2*pi),log10(abs(DK(ndx3d([1 cset_size*(cset_size+1)/2 numpoints-
280 2],1,skyCset(mid_index,mid_index),''))))),r-',...
    w(1:numpoints-2)/(2*pi),log10((true_stiffness(cset(mid_index),cset(mid_index))== 0)+...
    true_stiffness(cset(mid_index),cset(mid_index)).*ones(1,numpoints-2),'b-'))
    ylabel(['DK(',int2str(cset(mid_index)),',';int2str(cset(mid_index)),') lbf/in (Log10 of)'])
    xlabel('Omega (Hz)')
end
285 v=axis ;
if titles
    title('Computed Stiffness Error vs True Stiffness Error')
end
290 v2(1)=v(1);
v2(2)=v(2);
if complete
    if abs(v(4)-v(3)) < 1
        v2(3)=v(3)-100*abs(v(4)-v(3));
        v2(4)=v(4)+100*abs(v(4)-v(3));
        v(3)=v2(3);
        v(4)=v2(4);
    end
295 00

```

PLOTSST.M

```

if abs(true_stiffness(cset(mid_index),cset(mid_index)) - v(3)) < .25*abs(v(4)-v(3))
    v2(3)=v(3)-.5*abs(v(4)-v(3));
    v2(4)=v(4);
elseif abs(true_stiffness(cset(mid_index),cset(mid_index)) - v(4)) < .25*abs(v(4)-v(3))
    v2(4)=v(4)+.5*abs(v(4)-v(3));
    v2(3)=v(3);
else
    v2(3)=v(3);
    v2(4)=v(4);
end
310 else
    if abs(log10((true_stiffness(cset(mid_index),cset(mid_index)) == 0)+...
        true_stiffness(cset(mid_index),cset(mid_index)) - v(3)) < .25*abs(v(4)-v(3)))
        v2(3)=v(3)-.5*abs(v(4)-v(3));
        v2(4)=v(4);
    elseif abs(log10((true_stiffness(cset(mid_index),cset(mid_index)) == 0)+...
        true_stiffness(cset(mid_index),cset(mid_index)) - v(4)) < .25*abs(v(4)-v(3)))
        v2(4)=v(4)+.5*abs(v(4)-v(3));
        v2(3)=v(3);
    else
        v2(3)=v(3);
        v2(4)=v(4);
    end
end
325 axis(v2)
grid on
hh=slegend('COMPUTED STIFFNESS ERROR','TRUE STIFFNESS ERROR');
axes(hh)
if pswitch=='y'
    prtfi(gignum)
    delete(h)
end
clear DK
load DM
335 h=figure(h+1) ;
gignum=gignum+1 ;
if complete
    plot(w(1:numpoints-2)/(2*pi),DM(ndx3d([1 cset_size*(cset_size+1)/2 numpoints-2],1,sky(cset(mid_index,mid_index),'')),r',...
    w(1:numpoints-2)/(2*pi),true_mass(cset(mid_index),cset(mid_index)).*ones(1,numpoints-2),'b-')
    ylabel(['DM(',int2str(cset(mid_index)),',' ,int2str(cset(mid_index)),') lbf-sec^2/in'])
    xlabel('Omega (Hz)')
else
    plot(w(1:numpoints-2)/(2*pi),log10(abs(DM(ndx3d([1 cset_size*(cset_size+1)/2 numpoints-2],1,sky(cset(mid_index,mid_index),''))),r',...
2],1,sky(cset(mid_index,mid_index),'')))),r',...
    w(1:numpoints-2)/(2*pi),log10(... ...
    true_mass(cset(mid_index),cset(mid_index))==0 ...
    )+true_mass(cset(mid_index),cset(mid_index))...
    ).*ones(1,numpoints-2),'b-')
    ylabel(['DM(',int2str(cset(mid_index)),',' ,int2str(cset(mid_index)),') lbf-sec^2/in (Log10 of)'])
    xlabel('Omega (Hz)')
end
345 v=axis;
if titles
    title('Computed Mass Error vs True Mass Error')
end
hold on
v2(1)=v(1);
v2(2)=v(2);
355

```

PLOTSST.M

```

if complete
  if abs(v(4)-v(3)) < 1
    v2(3)=v(3)-10000*abs(v(4)-v(3));
    v2(4)=v(4)+10000*abs(v(4)-v(3));
    v(3)=v2(3);
    v(4)=v2(4);
  end
  if abs(true_mass(cset(mid_index),cset(mid_index)) - v(3)) < .25*abs(v(4)-v(3))
    v2(3)=v(3)-.5*abs(v(4)-v(3));
    v2(4)=v(4);
  elseif abs(true_mass(cset(mid_index),cset(mid_index)) - v(4)) < .25*abs(v(4)-v(3))
    v2(4)=v(4)+.5*abs(v(4)-v(3));
    v2(3)=v(3);
  else
    v2(3)=v(3);
    v2(4)=v(4);
  end
else
  if abs(log10((true_mass(cset(mid_index),cset(mid_index)) == 0)+...
    true_mass(cset(mid_index),cset(mid_index))) - v(3)) < .25*abs(v(4)-v(3))
    v2(3)=v(3)-.5*abs(v(4)-v(3));
    v2(4)=v(4);
  elseif abs(log10((true_mass(cset(mid_index),cset(mid_index)) == 0)+...
    true_mass(cset(mid_index),cset(mid_index))) - v(4)) < .25*abs(v(4)-v(3))
    v2(4)=v(4)+.5*abs(v(4)-v(3));
    v2(3)=v(3);
  else
    v2(4)=v(4);
    v2(3)=v(3);
  end
end
axis(v2)
grid on
hh=slegend('COMPUTED MASS ERROR','TRUE MASS ERROR');
axes(hh)
if pswitch=='y'
  prtf1g(signum)
  delete(h)
end
clear DM

load DC
h=figure(h+1);
signum=signum+1;
if complete
  plot(w(1:numpoints-2)/(2*pi),imag(DC(ndx3d([1 cset_size*(cset_size+1)/2 numpoints-2],1,skycset(mid_index,mid_index),'')),'r-',...
    w(1:numpoints-2)/(2*pi),imag(true_damping(cset(mid_index),cset(mid_index)).*ones(1,numpoints-2)), 'b-')
  ylabel(['DC(' int2str(cset(mid_index)),',',int2str(cset(mid_index)),') lbf-sec/in'])
  xlabel('Omega (Hz)')
else
  plot(w(1:numpoints-2)/(2*pi),log10(abs(DC(ndx3d([1 cset_size*(cset_size+1)/2 numpoints-2],1,skycset(mid_index,mid_index),'')),'r-',...
    w(1:numpoints-2)/(2*pi),log10(abs(...
    ...
    true_damping(cset(mid_index),cset(mid_index)) == 0 ...
    )+true_damping(cset(mid_index),cset(mid_index))...
    ).*ones(1,numpoints-2),'b-')
  ylabel(['DC(' int2str(cset(mid_index)),',',int2str(cset(mid_index)),') lbf-sec/in (Log10 of)'])
  xlabel('Omega (Hz)')

```

PLOTSST.M

```

end
v=axis;
if titles
    title('Computed Damping Error vs True Damping Error')
425 end
v2(1)=v(1);
v2(2)=v(2);
if complete
    if abs(v(4)-v(3)) < 1
430        v2(3)=v(3)-100*abs(v(4)-v(3));
        v2(4)=v(4)+100*abs(v(4)-v(3));
        v(3)=v2(3);
        v(4)=v2(4);
    end
435    if abs(true_damping(cset(mid_index),cset(mid_index))-v(3)) < .25*abs(v(4)-v(3))
        v2(3)=v(3)-.25*abs(v(4)-v(3));
        v2(4)=v(4);
    elseif abs(true_damping(cset(mid_index),cset(mid_index))-v(4)) < .25*abs(v(4)-v(3))
440        v2(4)=v(4)+.25*abs(v(4)-v(3));
        v2(3)=v(3);
    else
        v2(3)=v(3);
        v2(4)=v(4);
    end
445 else
    if abs(log10((true_damping(cset(mid_index),cset(mid_index))==0)+...
        true_damping(cset(mid_index),cset(mid_index))-v(3)) < .1*abs(v(4)-v(3))
        v2(3)=(v(3)-.25*abs(v(4)-v(3)));
        v2(4)=v(4);
    elseif abs(log10((true_damping(cset(mid_index),cset(mid_index))==0)+...
        true_damping(cset(mid_index),cset(mid_index))-v(4)) < .1*abs(v(4)-v(3))
        v2(4)=(v(4)+.25*abs(v(4)-v(3)));
        v2(3)=v(3);
    else
455        v2(3)=v(3);
        v2(4)=v(4);
    end
end
axis(v2);
460 grid on
hh=legend('COMPUTED DAMPING ERROR','TRUE DAMPING ERROR');
axes(hh);
if pswitch=='y'
    prfign(signum)
    delete(h)
465 end
clear DC true_damping true_mass true_stiffness

470 %%%%%%%%
%%%%%%
load H_EXP
load CORRHA
%for mid_index=1:min(cset_size,multiplot)
475 h=figure(h+1);
signum=signum+1;
plot(w/(2*pi),log10(abs(H_EXP(ndx3d([1 asset_size*(asset_size+1)/2
numpoints],1,skyred(cset_rel(mid_index),cset_rel(mid_index)),':')))),r--,...
w/(2*pi),log10(abs(CORRHA(ndx3d([asset_size asset_size numpoints],cset_rel(mid_index),cset_rel(mid_index),':')))),b-')
480 ylabel(['H(',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (Log10 of)'])

```

PLOTSST.M

```

485 xlabel('Omega (Hz)')
if titles
    title('Experimental FRF vs DZ Corrected FRF')
end
grid on
v=axis;
hh=slegend(EXPERIMENTAL FRF,'CORRECTED FRF');
axes(hh) ;
if pswitch=='y'
490    prtfi(figurenum)
    delete(h)
end

clear CORRHA
495 load CORRHAD
h=figure(h+1);
figurenum=figurenum+1;
plot(w/(2*pi),log10(abs(H_EXP(ndx3d([1 asset_size*(asset_size+1)/2
numpoints],1,skyred(cset_rel(mid_index),cset_rel(mid_index),'i'))),r-'...
500 w(1:numpoints-2)/(2*pi),log10(abs(CORRHAD(ndx3d([asset_size asset_size (numpoints-
2]),cset_rel(mid_index),cset_rel(mid_index,'b'))),b-'...
ylabel(['H','int2str(cset(mid_index)),','int2str(cset(mid_index)),') in/lbf (Log10 of')])
xlabel('Omega (Hz)')
if titles
505    title('Experimental FRF vs DK/DM/DC Corrected FRF')
end
grid on
v=axis;
hh=slegend(EXPERIMENTAL FRF,'CORRECTED ANALYTIC FRF');
510 axes(hh);
if pswitch=='y'
    prtfi(figurenum)
    delete(h)
end
515 clear CORRHAD

save plotnum figurenum
clear

```

CHAP3.M

CHAP4.M

```

%%%%%SSTCONF FILE FOR A SPATIALLY INCOMPLETE BEAM
beammdl      ;%beammdl is a cantilevered 20 element beam model
%simmdl      ;%simmdl is a masses and springs model
pswitch='n'   ;%%do we print?
5           titles=1    ;%%display titles
meters=1     ;%use progress meters
whitebg('black') ;%switch to black figure background
close
lowmode=1;
10          highmode=10;

static=1;%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction
%define A set & O set%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction
15          aset=1:numdof;
oset=2:2:numdof; %%A SPATIALLY INCOMPLETE BEAM%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction
oset=sort(oset); %%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction

save sstconf
sst
20          plotsst

beammdl      ;%beammdl is a cantilevered 20 element beam model
%simmdl      ;%simmdl is a masses and springs model
pswitch='n'   ;%%do we print?
25           titles=1    ;%%display titles
meters=1     ;%use progress meters
whitebg('black') ;%switch to black figure background
close
lowmode=1;
30           highmode=10;

%define A set & O set%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction
aset=1:numdof;
oset=2:2:numdof; %%A SPATIALLY INCOMPLETE BEAM%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction
35           oset=sort(oset); %%%%%%%%reduction method 0=Guyan,1=IRS,2=Extraction

static=2;
save sstconf
dofig4_3

```

CHAP5.M

```

%%%%% COMPARE SINGLE MODE MATRIX AND INTEGRAL %%%%%%
%%%%% SOLUTIONS UNDER VARIOUS CONDITIONS %%%%%%
5   %%%%%%
812%%%%%
clear
closeall
load INT
10 whitebg('white')
clear L Z_ANAL c_anal c_exp conn h_exp h_anal_red k_anal k_exp
clear kexta kexto m_anal m_exp mexta mext0 temp_l_diags
clear true_damping true_mass true_stiffness z_anal z_anal_red z_exp
clear L Z_ANAL c_anal c_exp conn h_exp h_anal_red k_anal
15 clear kexta kexto m_anal mexta mext0 temp_l_diags

load H_EXP
load exp
cset=[5 7 9 11 13 15]
20 cset_rel=[3 4 5 6 7 8]
cset_size=length(cset)
mid_index=round(length(cset)/2);
fineness=200
ODZ1=[];
25 temp1=[];
odfreq=omegax(1);
center_freq=odfreq/(2*pi)
w1=odfreq;
odlength=2*pi;
30 odivisions=2;
lowerfreq=odfreq-15*odlength
upperfreq=odfreq+25*odlength
owl=odfreq-.5*odlength:odlength:oddivisions:odfreq+.5*odlength;
partition=owl/(2*pi)
35 df=(odlength/oddivisions)/(2*pi)
for count=1:length(owl)
    z_anal_red=kstat-owl(count)^2*mstat;
    h_anal_red=inv(z_anal_red);
    z_exp=k_exp+j*owl(count)*c_exp-owl(count)^2*m_exp;
40 h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
    hacc=h_anal_red(cset_rel,cset_rel);
    hxcc=h_exp(cset_rel,cset_rel);
    dz=inv(inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc);
45 ODZ1=[ODZ1; dz];
    temp1=[temp1; eye(cset_size) -owl(count)^2*eye(cset_size)+j*owl(count)*eye(cset_size)];
end
%theleftdz=ODZ1
50 %thebigone=temp1
DKDMDC1=temp1\ODZ1;
ODK1=DKDMDC1(1:cset_size,:);
ODM1=DKDMDC1(cset_size+1:2*cset_size,:);
ODC1=DKDMDC1(2*cset_size+1:3*cset_size,:);
55 save ODM1 ODM1 DKDMDC1 ODZ1 owl wl
save ODK1 ODK1
save ODC1 ODC1

%+++++
60 load stat;

```

CHAP5.M

```

odcorrha=[];
w=lowerfreq:(upperfreq-lowerfreq)/fineness:upperfreq;
numpoints=length(w);
kcorrected=kstat;
mcorrected=mstat;
ccorrected=cstat;
65 kcorrected(cset_rel,cset_rel)=kcorrected(cset_rel,cset_rel)+ODK1;
mcorrected(cset_rel,cset_rel)=mcorrected(cset_rel,cset_rel)+ODM1;
ccorrected(cset_rel,cset_rel)=ccorrected(cset_rel,cset_rel)+ODC1;
70 for i=1:numpoints
    tempza=kcorrected+j*w(i)*ccorrected-w(i)^2*mcorrected;
    tempha=inv(tempza);
    odcorrha=[odcorrha tempha(:)];
end
75 uncorrha=[];
for i=1:numpoints
    tempza=kstat+j*w(i)*cstat-w(i)^2*mstat;
    tempha=inv(tempza);
80 uncorrha=[uncorrha tempha(:)];
end
85 if meters
    waitbar_handle=waitbar(0,'Computing Experimental FRF');
else
    disp('Getting experimental FRF')
end
H_EXP=[];
90 for count=1:length(w)
    if meters
        waitbar(count/length(w));
    end
    z_exp=k_exp+j*w(count)*c_exp-w(count)^2*m_exp;
    h_exp=inv(z_exp);
95 h_exp=h_exp(aset,aset);
    skyindex=1;
    for index1=1:aset_size
        for index2=index1:aset_size
            red_holder(skyindex)=h_exp(index1,index2);
            skyindex=skyindex+1;
        end
    end
100 H_EXP=[H_EXP red_holder];
end
105 if meters
    close(waitbar_handle)
end
clear tempha tempza kcorrected mcorrected ccorrected
110 figure(1);
plot(w/(2*pi),log10(abs(H_EXP(ndx3d([1 aset_size*(aset_size+1)/2 numpoints],...
1,skyred(cset_rel(mid_index),cset_rel(mid_index),':'))),':',...
w/(2*pi),log10(abs(odcorrha(ndx3d([aset_size aset_size numpoints],cset_rel(mid_index),cset_rel(mid_index),':'))),':',...
w/(2*pi),log10(abs(uncorrha(ndx3d([aset_size aset_size numpoints],cset_rel(mid_index),cset_rel(mid_index),':'))),':',...
120 xlabel(['H(',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in lbf (log10 of)'])
125 ylabel(['H(',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in lbf (log10 of)'])
130 xlabel('Omega (Hz)')
135

```

CHAP5.M

```

%title('Single Mode Corrected FRF (Mode 1)')
grid
hh=slegend('Experimental FRF', 'Mode 1 Corrected FRF', 'Uncorrected Analytic FRF');

125 axes(hh)
print -dmfile fig5_1

%%%%%%%%%%%%%%%
%<--> MPDZ into DM, DK, & DC %%%%%%
130 %% USING MATRIX TECHNIQUE %%%%%%
%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
ODZ1=[];
temp1=[];
odlength=2*pi;
135 owl=omegax(1)-5*odlength:(omegax(2)-omegax(1)+10*odlength)/14:omegax(2)+5*odlength;

for count=1:length(owl)
    z_anal_red=kstat-owl(count)^2*mstat;
    h_anal_red=inv(z_anal_red);
140 z_exp=k_exp+j*owl(count)*c_exp-owl(count)^2*m_exp;
    h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
    hacc=h_anal_red(cset_rel,cset_rel);
    hxcc=h_exp(cset_rel,cset_rel);
    dz=inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc;
    ODZ1=[ODZ1; dz];
    temp1=[temp1; eye(cset_size) -owl(count)^2*eye(cset_size) +j*owl(count)*eye(cset_size)];
145
end
150
DKDMDC1=temp1\ODZ1;
ODK1=DKDMDC1(1:cset_size,:);
ODM1=DKDMDC1(cset_size+1:2*cset_size,:);
ODC1=DKDMDC1(2*cset_size+1:3*cset_size,:);
155 save ODM1 ODM1 DKDMDC1 ODZ1 owl w1
save ODK1 ODK1
save ODC1 ODC1

%%%%%%%%%%%%%%%
160 %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
odcorrh=[];
w=omegax(1)-15*pi:(omegax(2)-omegax(1)+35*pi)/fineness:omegax(2)+20*pi;
numpoints=length(w);
kcorrected=kstat;
165 mcorrected=mstat;
ccorrected=cstat;
kcorrected(cset_rel,cset_rel)=kcorrected(cset_rel,cset_rel)+ODK1;
mcorrected(cset_rel,cset_rel)=mcorrected(cset_rel,cset_rel)+ODM1;
ccorrected(cset_rel,cset_rel)=ccorrected(cset_rel,cset_rel)+ODC1;
170 for i=1:numpoints
    tempza=kcorrected+j*w(i)*ccorrected-w(i)^2*mcorrected;
    tempha=inv(tempza);
    odcorrh=[odcorrh tempha(:)];
end
175
uncorrrha=[];
for i=1:numpoints
    tempza=kstat+j*w(i)*cstat-w(i)^2*mstat;
    tempha=inv(tempza);
    uncorrrha=[uncorrrha tempha(:)];
180

```

CHAP5.M

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```

h_anal_red=inv(z_anal_red);
z_exp=k_exp+j*ow(count)*c_exp-ow(count)^2*m_exp;
h_exp=inv(z_exp);
h_exp=h_exp(aset,aset);
hacc=h_anal_red(cset_rel,cset_rel);
hxcc=h_exp(cset_rel,cset_rel);
dz=inv(inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc);
ODZ=[ODZ; dz];
temp=[temp; eye(cset_size) -ow(count)^2*eye(cset_size) +j*ow(count)*eye(cset_size)];
245 end

DKDMDC=temp\ODZ;
ODK=DKDMDC(1:cset_size,:);
ODM=DKDMDC(cset_size+1:2*cset_size,:);
ODC=DKDMDC(2*cset_size+1:3*cset_size,:);
clear hacc hxcc h_exp z_exp z_anal_red h_anal_red dz
save ODM ODM DKDMDC ODZ ow
255 save ODK ODK
260 save ODC ODC
%%%%%%%%%%%%%
%!/ncl@6%/%/%/%
waitbar_handle=waitbar(0,'Computing Experimental FRF');
else
265 disp('Getting experimental FRF')
end
H_EXP=[];
for count=1:length(plotw)
if meters
270 waitbar(count/length(plotw));
end
z_exp=k_exp+j*plotw(count)*c_exp-plotw(count)^2*m_exp;
h_exp=inv(z_exp);
h_exp=h_exp(aset,aset);
275 skyindex=1;
for index1=1:aset_size
for index2=index1:aset_size
red_holder(skyindex)=h_exp(index1,index2);
skyindex=skyindex+1;
280 end
end
H_EXP=[H_EXP red_holder'];
end
if meters
285 close(waitbar_handle)
end

odcorrh=[];
kcorrected=kstat;
290 mcorrected=mstat;
ccorrected=cstat;
kcorrected(cset_rel,cset_rel)=kcorrected(cset_rel,cset_rel)+ODK;
mcorrected(cset_rel,cset_rel)=mcorrected(cset_rel,cset_rel)+ODM;
ccorrected(cset_rel,cset_rel)=ccorrected(cset_rel,cset_rel)+ODC;
295 for i=1:length(plotw)
tempza=kcorrected+j*plotw(i)*ccorrected-plotw(i)^2*mcorrected;
tempha=inv(tempza);
odcorrh=[odcorrh tempha(:)];
end
300 if ind==1

```

CHAP5.M

```

plot1=odcorrha;
elseif ind==2
    plot2=odcorrha;
elseif ind==3
    plot3=odcorrha;
else
    plot4=odcorrha;
end
end
end

310
figure(3);
plot(plotw/(2*pi),log10(abs(H_EXP(ndx3d([1 asset_size*(asset_size+1)/2 length(plotw],...
1,skyred(cset_rel(mid_index),cset_rel(mid_index),':'))),'r',...
315 plotw/(2*pi), log10(abs(plot1(ndx3d([asset_size asset_size length(plotw],cset_rel(mid_index),cset_rel(mid_index),':'))),'g-',...
',...
    plotw/(2*pi), log10(abs(plot2(ndx3d([asset_size asset_size...
length(plotw],cset_rel(mid_index),cset_rel(mid_index),':'))),'b',...
    plotw/(2*pi), log10(abs(plot3(ndx3d([asset_size asset_size length(plotw],cset_rel(mid_index),cset_rel(mid_index),':'))),'k-')
ylabel(['H',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (log10 of)])]

xlabel('Omega (Hz)')
grid on
325 %title('EXPERIMENTAL FRF AND CORRECTED FRFS USING MODE 5 SOLUTIONS')
hh=legend('Exp FRF', ...
    [num2str(lenctrl(1)), ' Hz Bandwidth Corrected FRF'], ...
    [num2str(lenctrl(2)), ' Hz Bandwidth Corrected FRF'], ...
    [num2str(lenctrl(3)), ' Hz Bandwidth Corrected FRF']);
330 axes(hh)
print -dmfile fig5_3

odlength=1*2*pi;
countctrl=[3 10 50 200];
335 for ind=1:length(countctrl)
    odivisions=countctrl(ind)-1;
    ODZ=[];
    temp=[];
    ow=odfreq-.5*odlength:odlength/odivisions:odfreq+.5*odlength;
    for count=1:length(ow)
        z_anal_red=kstat-ow(count)^2*mstat;
        h_anal_red=inv(z_anal_red);
        z_exp=k_exp+j*ow(count)*c_exp-ow(count)^2*m_exp;
        h_exp=inv(z_exp);
        340 h_exp=h_exp(asset,asset);
        hacc=h_anal_red(cset_rel,cset_rel);
        hxcc=h_exp(cset_rel,cset_rel);
        dz=inv(inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc);
        ODZ=[ODZ; dz];
        temp=[temp; eye(cset_size) -ow(count)^2*eye(cset_size)+j*ow(count)*eye(cset_size)];
    end
345
350
355
60

```

DKDMDC=temp\ODZ;
ODK=DKDMDC(1:cset_size,:);
ODM=DKDMDC(cset_size+1:2*cset_size,:);
ODC=DKDMDC(2*cset_size+1:3*cset_size,:);
clear hacc hxcc h_exp z_exp z_anal_red h_anal_red dz
save ODM ODM DKDMDC ODZ ow
save ODK ODK

CHAP5.M

CHAP5.M

```

odlength=llenctr(ind)*2*pi;
owl=odfreq-.5*odlength:odlength/(odivisions):odfreq+.5*odlength;
W1=owl;
425 %%compute integrals %%%%%%%%
%weight=ones(size(owl))*(omegax(1));
weight=owl;
DZ_R=[];
DZ_I=[];
for count=1:length(owl)
430 z_anal_red=kstat/(j*owl(count))+j*owl(count)*mstat;
h_anal_red=inv(z_anal_red);
z_exp=k_exp/(j*owl(count))+c_exp/(j*owl(count))+j*owl(count)*m_exp;
h_exp=inv(z_exp);
h_exp=h_exp(aset,aset);
435 hacc=h_anal_red(cset_rel,cset_rel);
hxcc=h_exp(cset_rel,cset_rel);
dz=inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc;
dz_r=real(dz);
dz_i=imag(dz);
440 DZ_R=[DZ_R dz_r(:)];
DZ_I=[DZ_I dz_i(:)];
end

[INTK INTM INTC]=intsub(DZ_I,DZ_R,owl,W1,1./(j*owl),cset_size);
445 odcorha=[];
load stat
kcorrected=kstat;
mincorrected=mstat;
ccorrected=cstat;
450 kcorrected(cset_rel,cset_rel)=kcorrected(cset_rel,cset_rel)+INTK;
mincorrected(cset_rel,cset_rel)=mincorrected(cset_rel,cset_rel)+INTM;
ccorrected(cset_rel,cset_rel)=ccorrected(cset_rel,cset_rel)+INTC;
for i=1:length(plotw)
455 tempza=kcorrected+j*plotw(i)*ccorrected-plotw(i)^2*mincorrected;
temph=inv(tempza);
odcorha=[odcorha temph(:)];
end
if ind==1
460 intplot1=odcorha;
save INTM1 INTM
save INTK1 INTK
save INTC1 INTC
elseif ind==2
465 intplot2=odcorha;
save INTM2 INTM
save INTK2 INTK
save INTC2 INTC
elseif ind==3
70 intplot3=odcorha;
save INTM3 INTM
save INTK3 INTK
save INTC3 INTC
else
75 intplot4=odcorha;
save INTM4 INTM
save INTK4 INTK
save INTC4 INTC
end
end

```

CHAP5.M

```

if meters
    waitbar_handle=waitbar(0,'Computing Experimental FRF');
else
    disp('Getting experimental FRF')
485 end
H_EXP=[];
for count=1:length(plotw)
    if meters
        waitbar(count/length(plotw));
    end
490 z_exp=k_exp+j*plotw(count)*c_exp-plotw(count)^2*m_exp;
h_exp=inv(z_exp);
h_exp=h_exp(aset,aset);
skyindex=1;
495 for index1=1:aset_size
    for index2=index1:aset_size
        red_holder(skyindex)=h_exp(index1,index2);
        skyindex=skyindex+1;
    end
500 end
H_EXP=[H_EXP red_holder'];
end
if meters
    close(waitbar_handle)
505 end

figure(5);
plot(plotw/(2*pi),log10(abs(H_EXP(ndx3d([1 aset_size*(aset_size+1)/2 length(plotw],...
510 1,skyred(cset_rel(mid_index),cset_rel(mid_index),':'))),r',...
    plotw/(2*pi), log10(abs(intplot1(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),':'))),'g',...
    plotw/(2*pi), log10(abs(intplot2(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),':'))),'b',...
515 plotw/(2*pi), log10(abs(intplot3(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),':'))),'k',...
ylabel(['H',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (log10 of')])

520 xlabel('Omega (Hz)')
grid on
%title('EXPERIMENTAL FRF AND CORRECTED FRFS USING MODE 1 INT SOLUTIONS')
hh=legend('Exp FRF',...
    [num2str(lenctrl(1)),' Hz Bandwidth Corrected FRF',...
    [num2str(lenctrl(2)),' Hz Bandwidth Corrected FRF',...
525    [num2str(lenctrl(3)),' Hz Bandwidth Corrected FRF']);
axes(hh)
print -dmfile fig5_5

530 odlength=2*pi;
countctrl=[3 10 50 200];
for ind=1:length(countctrl)
    odivisions=countctrl(ind);
    owl=odfreq-.5*odlength:odlength/(odivisions-1):odfreq+.5*odlength;
    W1=owl;
535 %%compute integrals %%%%%%%%
    %weight=ones(size(owl))*(omegax(1));
    weight=owl;
    if ind == 1
        owl
540 df=(odlength/(odivisions-1))/(2*pi)

```

CHAP5.M

```

end
%% compute integrals
DZ_R=[];
DZ_I=[];
for count=1:length(owl)
    z_anal_red=kstat/(j*owl(count))+j*owl(count)*mstat;
    h_anal_red=inv(z_anal_red);
    z_exp=k_exp/(j*owl(count))+c_exp/(j*owl(count))+j*owl(count)*m_exp;
    h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
    hacc=h_anal_red(cset_rel,cset_rel);
    hxcc=h_exp(cset_rel,cset_rel);
    dz=inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc;
    dz_r=real(dz);
    dz_i=imag(dz);
    DZ_R=[DZ_R dz_r(:)];
    DZ_I=[DZ_I dz_i(:)];
end
if ind == 1
    DZ_R;
    DZ_I;
    for i=1:length(owl)
        dztemp1=DZ_R(ndx3d([1 cset_size*(cset_size+1)/2 length(owl)],1,1:cset_size*(cset_size+1)/2,i));
        dztemp2=DZ_I(ndx3d([1 cset_size*(cset_size+1)/2 length(owl)],1,1:cset_size*(cset_size+1)/2,i));
        skyindex=1;
        for index1=1:cset_size
            for index2=index1:cset_size
                dz1(index1,index2)=dztemp1(skyindex);
                dz1(index2,index1)=dztemp1(skyindex);
                dz2(index1,index2)=dztemp2(skyindex);
                dz2(index2,index1)=dztemp2(skyindex);
                skyindex=skyindex+1;
            end
        end
        dz1;
        dz2;
    end
end
[INTK INTM INTC]=intsub(DZ_I,DZ_R,owl,W1,1./(j*owl),cset_size);
if ind == 1
    [INTK INTM INTC]=indsub(DZ_LDZ_R,owl,W1,1./(j*owl),cset_size);
    INTM
    INTK
    INTC
end

% kstat
kcorrected=kstat;
mcorrected=mstat;
ccorrected=cstat;
kcorrected(cset_rel,cset_rel)=kcorrected(cset_rel,cset_rel)+INTK;
mcorrected(cset_rel,cset_rel)=mcorrected(cset_rel,cset_rel)+INTM;
ccorrected(cset_rel,cset_rel)=ccorrected(cset_rel,cset_rel)+INTC;
for i=1:length(plotw)
    tempza=kcorrected+j*plotw(i)*ccorrected-plotw(i)^2*mcorrected;
    temphaa=inv(tempza);

```

CHAP5.M

```

odcorrha=[odcorrha tempaha(:)];
end
if ind==1
    intplot1=odcorrha;
    save INTM1 INTM
    save INTK1 INTK
    save INTC1 INTC
elseif ind==2
    intplot2=odcorrha;
    save INTM2 INTM
    save INTK2 INTK
    save INTC2 INTC
elseif ind==3
    intplot3=odcorrha;
    save INTM3 INTM
    save INTK3 INTK
    save INTC3 INTC
else
    intplot4=odcorrha;
    save INTM4 INTM
    save INTK4 INTK
    save INTC4 INTC
end
end
625
if meters
    waitbar_handle=waitbar(0,'Computing Experimental FRF');
else
    disp('Getting experimental FRF')
end
630
H_EXP=[];
for count=1:length(plotw)
    if meters
        waitbar(count/length(plotw));
    end
    635
    z_exp=k_exp+j*plotw(count)*c_exp-plotw(count)^2*m_exp;
    h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
    skyindex=1;
    640
    for index1=1:aset_size
        for index2=index1:aset_size
            red_holder(skyindex)=h_exp(index1,index2);
            skyindex=skyindex+1;
        end
    end
    645
    H_EXP=[H_EXP red_holder];
end
if meters
    close(waitbar_handle)
end
650
figure(6);
plot(plotw/(2*pi),log10(abs(H_EXP(ndx3d([1:aset_size*(aset_size+1)/2:length(plotw)],...
655      1,skyred(cset_rel(mid_index),cset_rel(mid_index),'.'))),'r',...
    plotw/(2*pi), log10(abs(intplot1(ndx3d([aset_size:aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),'.'))),'g',...
    plotw/(2*pi), log10(abs(intplot2(ndx3d([aset_size:aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),'.'))),'b',...

```

CHAP5.M

```

660 plotw/(2*pi), log10(abs(intplot3(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),''))),k-,...
    plotw/(2*pi), log10(abs(intplot4(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),''))),m-
ylabel(['H',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (log10 of)'])

 xlabel('Omega (Hz)')
 grid on
%title('EXPERIMENTAL FRF AND CORRECTED FRFS USING MODE 1 INT SOLUTIONS')
hh=legend('Exp FRF', ...
670 [num2str(countctrl(1)), ' Pts Corrected FRF'], ...
    [num2str(countctrl(2)), ' Pts Corrected FRF'], ...
    [num2str(countctrl(3)), ' Pts Corrected FRF'], ...
    [num2str(countctrl(4)), ' Pts Corrected FRF']);
axes(hh);
print -dmfile fig5_6
figure(7);
plot(plotw/(2*pi),log10(abs(H_EXP(ndx3d([1 aset_size*(aset_size+1)/2 length(plotw],...
675
1,skyred(cset_rel(mid_index),cset_rel(mid_index),''))),r-,...
    plotw/(2*pi), log10(abs(plot1(ndx3d([aset_size aset_size length(plotw]),cset_rel(mid_index),cset_rel(mid_index),''))),g-
',...
    plotw/(2*pi), log10(abs(intplot1(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),''))),b-
ylabel(['H',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (log10 of)'])

 xlabel('Omega (Hz)')
if titles
% title('EXPERIMENTAL FRF AND CORRECTED FRFS USING MODE 1 OD&INTGRL SOLTNS')
end
grid on
hh=legend('Exp FRF', ...
680 [num2str(lenctrl(1)), ' Hz Bandwidth MAT Corrected FRF'], ...
    [num2str(lenctrl(1)), ' Hz Bandwidth INT Corrected FRF']);
axes(hh)
if pswitch=='y'
    print -dcnjcolor
end
print -dmfile fig5_7
690
95 figure(8);
plot(plotw/(2*pi),log10(abs(H_EXP(ndx3d([1 aset_size*(aset_size+1)/2 length(plotw],...
00
1,skyred(cset_rel(mid_index),cset_rel(mid_index),''))),r-,...
    plotw/(2*pi), log10(abs(plot2(ndx3d([aset_size aset_size length(plotw]),cset_rel(mid_index),cset_rel(mid_index),''))),g-
',...
    plotw/(2*pi), log10(abs(intplot2(ndx3d([aset_size aset_size
length(plotw]),cset_rel(mid_index),cset_rel(mid_index),''))),b-
ylabel(['H',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (log10 of)'])

 xlabel('Omega (Hz)')
if titles
%title('EXPERIMENTAL FRF AND CORRECTED FRFS USING MODE 1 OD&INTGRL SOLTNS')
end
grid on
hh=legend('Exp FRF', ...
10 [num2str(lenctrl(2)), ' Hz Bandwidth MAT Corrected FRF'], ...
    [num2str(lenctrl(2)), ' Hz Bandwidth INT Corrected FRF']);
axes(hh)
if pswitch=='y'
15

```

CHAP5.M

```
720 print -dcnjcolor
end
print -dmfile fig5_8

725 figure(9);
plot(plotw/(2*pi),log10(abs(H_EXP(ndx3d([1 asset_size*(asset_size+1)/2 length(plotw],...
1,skyred(cset_rel(mid_index),cset_rel(mid_index),':'))]),r',...
plotw/(2*pi), log10(abs(plot3(ndx3d([asset_size asset_size length(plotw],cset_rel(mid_index),cset_rel(mid_index),':')))),g...
',...
plotw/(2*pi), log10(abs(intplot3(ndx3d([asset_size asset_size
length(plotw],cset_rel(mid_index),cset_rel(mid_index),':')))),b:')
ylabel(['H',int2str(cset(mid_index)),',',int2str(cset(mid_index)),') in/lbf (log10 of)'])

730 xlabel('Omega (Hz)')
735 if titles
    %title('EXPERIMENTAL FRF AND CORRECTED FRFS USING MODE 1 OD&INTGRL SOLTNS')
end
grid on
hh=legend('Exp FRF',...
740 [num2str(lenctrl(3)),' Hz Bandwidth MAT Corrected FRF',...
    [num2str(lenctrl(3)),' Hz Bandwidth INT Corrected FRF']);
axes(hh)
if pswitch=='y'
    print -dcnjcolor
end
745 print -dmfile fig5_9
```

CHAP6.M

```
clear
odivisions=3
startloop=1
skiploop=1
5 endloop=4
cd incomp1
save odconf odivisions startloop skiploop endloop
chap6_1
chap6_2
10 cd ..
clear
odivisions=1
startloop=3
skiploop=1
15 endloop=4
cd comp1
save odconf odivisions startloop skiploop endloop
chap6_1
chap6_3
```

CHAP6.M

```

20 %%CHAP6_1%%%%%%%%%%%%%%%
21 %%Decompose DZ into DM, DK, & DC%%%%%%%%%%%%%%
22 %%using matrix formulation%%%%%%%%%%%%%%
23 %%%%%%%%%%%%%%%%
24 %%%%%%%%%%%%%%%%
25 %%%%%%%%%%%%%%%%
26 %%clc
27 %%clear
28 %%closeall
29 %%load INT
30 %%if pswitch=='y'
31 %%whitebg('white')
32 %%close
33 %%end
34 %%h=0;
35 %%fignum=1;
36 %%use_antires=1;
37 %%titles=0;
38 %%mid_index=round(length(cset)/2);
39
40 clear L Z_ANAL c_exp conn h_anal_red
41 clear kexto m_exp mexto temp_1_diags
42 clear true_damping true_mass true_stiffness z_anal z_anal_red z_exp
43 clear L Z_ANAL c_exp conn h_anal_red
44 clear kexta kexto mext0 temp_1_diags
45
46 %%FORCE THE CSET%%%%%%%%%%%%%%
47 %%SPATIALLY INCOMPLETE%%%%%%%%%%%%%%
48 if complete
49 %%SPATIALLY COMPLETE%%%%%%%%%%%%%%
50 cset=[4 5 6 7 8 9 10 11]%%%%%%%%%%%%%%
51 cset_rel=cset%%%%%%%%%%%%%%
52 cset_size=length(cset)%%%%%%%%%%%%%%
53
54 else
55 %%SPATIALLY INCOMPLETE%%%%%%%%%%%%%%
56 cset=[1 3 5 7 9 11 13 15]%%%%%%%%%%%%%%
57 cset_rel=[1 2 3 4 5 6 7 8]%%%%%%%%%%%%%%
58 cset_size=length(cset)%%%%%%%%%%%%%%
59
60 end
61
62 load exp
63 odivisions=3 ;
64 load odconf
65 numnodes=8 ;
66 odlength=2.001*pi ;
67 firsttime=1;
68 dw=odlength/(odivisions+1);
69 dow=(freqtop-freqbottom)/(fineness-1);
70 owref=freqbottom:dow:freqtop;
71
72 if firsttime
73 if meters
74 waitbar_handle=waitbar(0,'Computing Experimental FRF');
75 else
76 disp('Getting experimental FRF')
77 end

```

CHAP6.M

```

80 H_T_EXP=[];
for count=1:length(owref)
    if meters
        waitbar(count/length(owref));
    end
85 z_exp=k_exp+j*owref(count)*c_exp-owref(count)^2*m_exp;
    h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
% skyindex=1;
% for index1=1:aset_size
%     for index2=index1:aset_size
%         red_holder(skyindex)=h_exp(index1,index2);
%         skyindex=skyindex+1;
%     end
% end
90 % H_T_EXP=[H_T_EXP red_holder];
95 H_T_EXP=[H_T_EXP h_exp(:)];
end
if meters
    close(waitbar_handle)
end
100 save H_T_EXP H_T_EXP
clear H_T_EXP
%load H_ANAL_RED;
%uncorrrha=H_ANAL_RED;
%clear H_ANAL_RED
105 uncorrrha=[];
if meters
    waitbar_handle=waitbar(0,'Computing Uncorrected FRF');
else
    disp('Getting Uncorrected FRF')
end
load stat
110 for i=1:length(owref)
    if meters
        waitbar(i/length(owref));
    end
    tempza=kstat+j*owref(i)*cstat-owref(i)^2*mstat;
    tempma=inv(tempza);
    uncorrrha=[uncorrrha tempma(:)];
    115 end
    if meters
        close(waitbar_handle)
    end
    save uncorrrha uncorrrha
    clear uncorrrha
20    25 end
load H_T_EXP
temp=[];
for index=1:nummodes
30    for coord=1:aset_size
        if index == 1
            antiresfreqs=find(owref>0 & owref<omegax(index));
        else
            antiresfreqs=find(owref>omegax(index-1) & owref<omegax(index));
        end
        antires=min(log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref],...
            coord,coord,antiresfreqs))))));
        whre=find(log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref],...
            coord,coord,antiresfreqs)))))==antires);
35

```

CHAP6.M

```

140      temp=owref(antiresfreqs);
      antiresfreq(coord,index)=temp(whre(1));
      end
end
temp=[];
145 for index=1:nummodes
      resfreqs=find(owref>omegax(index)-dow & owref<omegax(index)+dow);
      res=max(log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
      cset_rel(1),cset_rel(1),resfreqs))))) ;
      whre=find(log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
      cset_rel(1),cset_rel(1),resfreqs))))==res);
      temp=owref(resfreqs) ;
      resfreq(index)=temp(whre(1));
      end
clear H_T_EXP
155
%clear omegax
%omegax=resfreq;
for loopindex=startloop:skiploop:endloop
160      ODZ=[];
      temp=[];
      rdiag=[];
      ow=[];
      lowwer=owref(1)-.5*odlengt;
      upppter=owref(1)+.5*odlengt ;
165      % if odivisions == 1
      %   ow=[ow owref(1)]
      % else
      %   ow=[ow lowwer+dw:dw:upppter-dw];
      % end
      % rdiag=[rdiag ones(1,odivisions*cset_size).*owref(1)];
170
      for index=1:loopindex
175      %   lowwer=omegax(index)-.5*odlengt ;
      %   upppter=omegax(index)+.5*odlengt ;

      lowwer=resfreq(index)-.5*odlengt ;
      upppter=resfreq(index)+.5*odlengt ;
      ow=[ow lowwer+dw:dw:upppter-dw];
      rdiag=[rdiag ones(1,odivisions*cset_size).*resfreq(index)];
      if use_antires

180      for coord=1:aset_size
          lowwer=antiresfreq(coord,index)-.5*odlengt;
          upppter=antiresfreq(coord,index)+.5*odlengt;
          size(lowwer+dw:dw:upppter-dw)
          ow=[ow lowwer+dw:dw:upppter-dw]
          rdiag=[rdiag ones(1,odivisions*cset_size).*resfreq(index)];
185      end
      end
      R=diag(rdiag);
      load stat
190      if meters
          waitbar_handle=waitbar(0,'Computing DK/DM/DC');
      else
          disp('Getting DK/DM/DC')
      end

```

CHAP6.M

```

200
for count=1:length(ow)
if meters
    waitbar(count/length(ow));
end
205
z_anal_red=kstat-ow(count)^2*mstat;
h_anal_red=inv(z_anal_red);
z_exp=k_exp+j*ow(count)*c_exp-ow(count)^2*m_exp;
h_exp=inv(z_exp);
h_exp=h_exp(aset,aset);
hacc=h_anal_red(cset_rel,cset_rel);
hxcc=h_exp(cset_rel,cset_rel);
dz=inv(inv(hacc)*(hacc-hxcc)*inv(hacc))-hacc;
ODZ=[ODZ; dz];
temp=[temp; eye(cset_size) -ow(count)^2*eye(cset_size)+j*ow(count)*eye(cset_size)];
end
15
if meters
    close(waitbar_handle)
end
DKDMDC1=inv(temp*inv(R)*temp)*temp*inv(R)*ODZ;
20
%DKDMDC1=temp\ODZ ;
ODK1=DKDMDC1(1:cset_size,:);
ODM1=DKDMDC1(cset_size+1:2*cset_size,:);
ODC1=DKDMDC1(2*cset_size+1:3*cset_size,:);

25
clear hacc hxcc h_exp z_exp z_anal_red h_anal_red dz
clear temp ODZ

30
%%%%%%%%%%%%%
%%%%%
load stat;
odcorrha=[];
35
kstat(cset_rel,cset_rel)=kstat(cset_rel,cset_rel)+ODK1;
mstat(cset_rel,cset_rel)=mstat(cset_rel,cset_rel)+ODM1;;
cstat(cset_rel,cset_rel)=cstat(cset_rel,cset_rel)+ODC1;

40
if meters
    waitbar_handle=waitbar(0,'Computing Corrected FRF');
else
    disp('Getting DK/DM/DC')
end
45
for i=1:length(owref)
if meters
    waitbar(i/length(owref));
end
50
tempza=kstat+j*owref(i)*cstat-owref(i)^2*mstat;
tempha=inv(tempza);
odcorrha=[odcorrha tempha(:)];
55
end
0
if meters
    close(waitbar_handle)
end
if loopindex == 1
    odcorrha1=odcorrha;
    save od1 odcorrha1
    clear odcorrha1
5
elseif loopindex == 2
    odcorrha2=odcorrha;
    save od2 odcorrha2

```

CHAP6.M

```

260 clear odcorraha2
elseif loopindex == 3
    odcorraha3=odcorraha;
    save od3 odcorraha3
    clear odcorraha3
265 elseif loopindex == 4
    odcorraha4=odcorraha;
    save od4 odcorraha4
    clear odcorraha4
elseif loopindex == 5
270    odcorraha5=odcorraha;
    save od5 odcorraha5
    clear odcorraha5
end

275 clear tempaha tempza kcorrected mcorrected ccorrected
clear ODM1 ODC1 DKDMDC1 ODZ temp kstat instat

load uncorrha
load H_T_EXP
280 if complete
    plotwhichcoord=[cset(mid_index)];
else
    plotwhichcoord=[cset(mid_index)];
end
285 for plotindex=1:length(plotwhichcoord)
    which_coord=plotwhichcoord(plotindex);
    h=figure(h+1);
    fignum=fignum+1 ;
    subplot(2,1,1);
    plot(owref/(2*pi),...
        log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!)))),r',...
        owref/(2*pi),...
        log10(abs(odcorraha(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!)))),g',...
        owref/(2*pi),...
        log10(abs(uncorrha(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!)))),b',...
        xlabel(['H',int2str(aset(which_coord)),...
        ',',int2str(aset(which_coord)),') in/lbf (log10 of)'])
295
300 hold on
v=axis;
% xlabel('Omega (Hz)')
if titles
305     if use_antires
        title([int2str(odivisions),' PT MATRIX SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT'])
    else
        title([int2str(odivisions),' PT MATRIX SOLTNS WEIGHTED LEFT TO RIGHT'])
    end
310 end
plot(resfreq(1:loopindex)/(2*pi),...
    ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),k+')
% if use_antires
% plot(diag(antiresfreq(1:loopindex,1:loopindex))/(2*pi),ones(1,loopinde%x)*(v(3)+abs(v(4)-v(3)).*.98),k*)
315 % end
grid on
hh=legend('Experimental FRF','Corrected MAT Anal FRF','Uncorrected Anal FRF','Included Modes');
axes(hh)
hold off

```

CHAP6.M

```

320      %if pswitch=='y'
%   prtf1(signum)
%   delete(h)
%   % else
%   delete(h)
325   % end
subplot(2,1,2)
thetop=find(abs(owref-ow(length(ow))*1.5*ones(1,length(owref)))==min(abs(owref-
ow(length(ow))*1.5*ones(1,length(owref)))));

330   plot(owref(1:thetop)/(2*pi),...
log10(abs(H_T_EXP(ndx3d([aset_size asset_size length(owref]),...
which_coord,which_coord,1:thetop)))),r',...
owref(1:thetop)/(2*pi),...
log10(abs(odcorrh(ndx3d([aset_size asset_size length(owref],...
which_coord,which_coord,1:thetop)))),g',...
owref(1:thetop)/(2*pi),...
log10(abs(uncorrrh(ndx3d([aset_size asset_size length(owref],...
which_coord,which_coord,1:thetop)))),b',...
ylabel(['H','int2str(aset(which_coord)),','int2str(aset(which_coord)),') in/lbf (log10 of')])
340   hold on
v=axis;
xlabel('Omega (Hz)')
if titles
345   if use_antires
      title(['int2str(odivisions),' PT MATRIX SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT'])
    else
      title(['int2str(odivisions),' PT MATRIX SOLTNS WEIGHTED LEFT TO RIGHT'])
    end
end
350   plot(resfreq(1:loopindex)/(2*pi),ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),k'+')
%   if use_antires
%   %plot(antiresfreq(which_coord,1:loopindex)/(2*pi),ones(1,loopindex)*(v(%3)+abs(v(4)-v(3)).*.98),k*)
%   end
355   grid on
%hh=legend('Experimental FRF','Corrected MAT Anal FRF','Uncorrected Anal FRF','Included Modes');
%axes(hh)
hold off
if loopindex == 2
360   print -dcdjcolor
   print -dmfile fig6_1
elseif loopindex == 3
365   if odivisions == 1
      print -dcdjcolor
      if complete
        print -dmfile fig6_9
      else
        print -dmfile fig6_6
      end
    end
70   elseif loopindex == 4
      print -dcdjcolor
      print -dmfile fig6_2
    end
75   % if pswitch=='y'
%   prtf1(signum)
%   delete(h)
%   % else
%   delete(h)
%   % end

```

CHAP6.M

```
380    end  
      delete(h)  
      clear H_T_EXP  
      clear odcorrra  
      clear uncorrra  
385    end
```

CHAP6.M

```

%%%%%CHAP6_2.M%%%%%%%%%%%%%%%
%%%%% Decompose DZ into DM, DK, & DC %%%%%%
%%%%% using matrix formulation %%%%%%
%%%%% %%%%%%
%%%%% %%%%%%
390    clc
395    clear
    closeall
    load INT
    whitebg('white')
    close
    h=0;
400    signum=1;
    use_antires=1
    titles=0
    mid_index=round(length(cset)/2);

405    clear L Z_ANAL c_exp conn h_anal_red
    clear kexto m_exp mexto temp_l_diags
    clear true_damping true_mass true_stiffness z_anal z_anal_red z_exp
    clear L Z_ANAL c_exp conn h_anal_red
    clear kexta kexto mext0 temp_l_diags
410
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%FORCE THE CSET%%%%%%%%%%%%%%%%%%%%
%%%%%%%%%%%%%%%%%%%%SPATIALLY INCOMPLETE%%%%%%%%%%%%%%%%%%%%
if complete
%%%%%%%%%%%%%%%%%%%%%%%%%SPATIALLY COMPLETE%%%%%%%%%%%%%%%%%%%%
415    cset=[4 5 6 7 8 9 10 11]%%%%%
    cset_rel=cset%%%%%
    cset_size=length(cset)%%%%%
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
420    else
%%%%%%%%%%%%%%%%%%%%SPATIALLY INCOMPLETE%%%%%%%%%%%%%%%%%%%%%%%%%
        cset=[1 3 5 7 9 11 13 15]%%%%%
        cset_rel=[1 2 3 4 5 6 7 8]%%%%%
        cset_size=length(cset)%%%%%
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
425    end
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
30
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%SPATIALLY COMPLETE%%%%%%%%%%%%%%%%%%%%
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
30    %cset=[7 8 9 10 11 12 13 14]%%%%%
    %cset_rel=cset%%%%%
    %cset_size=length(cset)%%%%%
%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%%
35
load exp
odivisions=3 ;
load odconf
nummnodes=9 ;
odlength=2*pi ;
firsttime=1;
dw=odlength/(odivisions+1);
dow=(freqtop-freqbottom)/(fineness-1);
owref=freqbottom:dow:freqtop;

40
45    if firsttime
        if meters

```

CHAP6.M

```

    waitbar_handle=waitbar(0,'Computing Experimental FRF');
else
    disp('Getting experimental FRF')
end
450

H_T_EXP=[];
for count=1:length(owref)
    if meters
        waitbar(count/length(owref));
    end
    z_exp=k_exp+j*owref(count)*c_exp-owref(count)^2*m_exp;
    h_exp=inv(z_exp);
    h_exp=h_exp(aset,aset);
455
% skyindex=1;
% for index1=1:aset_size
%     for index2=index1:aset_size
%         red_holder(skyindex)=h_exp(index1,index2);
%         skyindex=skyindex+1;
460
%     end
% end
% H_T_EXP=[H_T_EXP red_holder];
H_T_EXP=[H_T_EXP h_exp(:)];
end
470

if meters
    close(waitbar_handle)
end
475

save H_T_EXP H_T_EXP
clear H_T_EXP
end
480
%load H_ANAL_RED;
%uncorrrha=H_ANAL_RED;
%clear H_ANAL_RED
if firsttime
    uncorrrha=[];
485
if meters
    waitbar_handle=waitbar(0,'Computing Uncorrected FRF');
else
    disp('Getting Uncorrected FRF')
end
490

load stat
for i=1:length(owref)
    if meters
        waitbar(i/length(owref));
    end
    tempza=kstat+j*owref(i)*cstat-owref(i)^2*mstat;
    tempha=inv(tempza);
    uncorrrha=[uncorrrha tempha(:)];
495
end
500

if meters
    close(waitbar_handle)
end
505

save uncorrrha uncorrrha
clear uncorrrha

```

CHAP6.M

```

end

510 load H_T_EXP
for index=1:nummodes
    for coord=1:aset_size
        if index == 1
            antiresfreqs=find(owref>0 & owref<omegax(index));
        else
            antiresfreqs=find(owref>omegax(index-1) & owref<omegax(index));
        end
        antires=min(log10(abs(H_T_EXP(ndx3d([1 aset_size*(aset_size+1)/2 length(owref],...
1,skyred(coord,coord),antiresfreqs))))));
        whre=find(log10(abs(H_T_EXP(ndx3d([1 aset_size*(aset_size+1)/2 length(owref],...
1,skyred(coord,coord),antiresfreqs)))))==antires);
        temp=owref(antiresfreqs);
        antiresfreq(coord,index)=temp(whre);
    end
end

525 for index=1:nummodes
    resfreqs=find(owref>omegax(index)-dow & owref<omegax(index)+dow);
    res=max(log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref],...
cset_rel(1),cset_rel(1),resfreqs))))) );
    whre=find(log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref],...
cset_rel(1),cset_rel(1),resfreqs)))))==res);
    temp=owref(resfreqs);
    resfreq(index)=temp(whre);
end

535 clear H_T_EXP
%clear omegax
%omegax=resfreq;

540 for loopindex=startloop:skiploop:endloop
    subdivisions=9; % should be odd for simpson's rule
    intlength=2*pi; % 1 Hz bandwidth
    dW=intlength/subdivisions; % sampling freq =.25 HZ
    W=[];
    W1=[];
    weight=[];
    lowwer=owref(1)-.5*odlength;
    upper=owref(1)+.5*odlength;
    W=[W lowwer+dW:dW:upper-dW];
    W1=[W1 lowwer+dW:dW:upper-dW];
    weight=[weight ones(size(lowwer+dW:dW:upper-dW))];

    for index=1:loopindex
        lowwer=omegax(index)-.5*intlength;
        upper=omegax(index)+.5*intlength;
        W=[W lowwer+dW:dW:upper-dW];
        W1=[W1 lowwer+dW:dW:upper-dW];
        weight=[weight ones(size(lowwer+dW:dW:upper-dW))];
        lower=antiresfreq(index)-.5*intlength;
        upper=antiresfreq(index)+.5*intlength;
        W=[W lowwer+dW:dW:upper-dW];
        W1=[W1 lowwer+dW:dW:upper-dW];
        weight=[weight ones(size(lowwer+dW:dW:upper-dW))];
    end

555 %%compute integrals%%%%%%%

```

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```

end

if meters
  close(waitbar_handle)
end

if loopindex == 1
  lodcorrh1=odcorrh1;
  save lod1 lodcorrh1
  clear lodcorrh1
elseif loopindex == 2
  lodcorrh2=odcorrh2;
  save lod2 lodcorrh2
  clear lodcorrh2
elseif loopindex == 3
  lodcorrh3=odcorrh3;
  save lod3 lodcorrh3
  clear lodcorrh3
elseif loopindex == 4
  lodcorrh4=odcorrh4;
  save lod4 lodcorrh4
  clear lodcorrh4
elseif loopindex == 5
  lodcorrh5=odcorrh5;
  save lod5 lodcorrh5
  clear lodcorrh5
end

clear temph1 tempza kcorrected mcorrected ccorrected
clear INTM INTC INTK temp kstat mstat cstat
load uncorrha
load H_T_EXP

if complete
  plotwhichcoord=[cset(mid_index)];
else
  plotwhichcoord=[cset(mid_index)];
end

for plotindex=1:length(plotwhichcoord)
  which_coord=plotwhichcoord(plotindex);
  h=figure(h+1);
  subplot(2,1,1)
  plot(owref/(2*pi),...
    log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
      which_coord,which_coord,'')))),r',...
    owref/(2*pi),...
    log10(abs(odcorrh(ndx3d([aset_size aset_size length(owref]),...
      which_coord,which_coord,'')))),g',...
    owref/(2*pi),...
    log10(abs(uncorrha(ndx3d([aset_size aset_size length(owref]),...
      which_coord,which_coord,'')))),b',')
  ylabel(['H',int2str(aset(which_coord)),...
    ',',int2str(aset(which_coord)),') in/lbf (log10 of')'])
  hold on
  v=axis;
  %xlabel('Omega (Hz)')
  if titles
    if use_antires == 'on'

```

CHAP6.M

```

    title([int2str(odivisions),' PT INTEGRAL SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT])
else
    title([int2str(odivisions),' PT INTEGRAL SOLTNS WEIGHTED LEFT TO RIGHT])
end
end
plot(omegax(1:loopindex)/(2*pi),...
    ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),'k+')
% if use_antires
% %plot(antiresfreq(which_coord,1:loopindex)/(2*pi),ones(1,loopindex)*(v(%3)+abs(v(4)-v(3)).*.98),'k*')
% end
grid on
hh=legend('Experimental FRF','Corrected INT Anal FRF','Uncorrected Anal FRF','Included Modes');
axes(hh)
hold off
%if pswitch=='y'
%  prtf1l(fignum)
%  delete(h)
%else
%  delete(h)
%end
subplot(2,1,2)
thetop=find(abs(owref-W(length(W))*1.3*ones(1,length(owref)))==min(abs(owref-
W(length(W))*1.3*ones(1,length(owref)))));

% h=figure(h+1);
% fignum=fignum+1 ;
plot(owref(1:thetop)/(2*pi),...
    log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,1:thetop)))), 'r',...
owref(1:thetop)/(2*pi),...
    log10(abs(odcorrha(ndx3d([aset_size aset_size length(owref)],...
        which_coord,which_coord,1:thetop)))), 'g',...
owref(1:thetop)/(2*pi),...
    log10(abs(uncorrha(ndx3d([aset_size aset_size length(owref)],...
        which_coord,which_coord,1:thetop)))), 'b-')
ylabel(['H(',int2str(aset(which_coord)),',',int2str(aset(which_coord)),') in/lbf (log10 of')])
hold on
v=axis;
xlabel('Omega (Hz)')
if titles
    if use_antires == 'on '
        title([int2str(odivisions),' PT INTEGRAL SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT])
    else
        title([int2str(odivisions),' PT INTEGRAL SOLTNS WEIGHTED LEFT TO RIGHT])
    end
end
plot(omegax(1:loopindex)/(2*pi),ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),'k+')
% if use_antires
% %plot(antiresfreq(which_coord,1:loopindex)/(2*pi),ones(1,loopindex)*(v(%3)+abs(v(4)-v(3)).*.98),'k*')
% end
grid on
%hh=legend('Experimental FRF','Corrected INT Anal %RF','Uncorrected Anal FRF','Included Modes');
%axes(hh)

if loopindex == 2
    print -cdjcolor
    print -dmfile fig6_3
    h=figure(h+1);
    hold off
load od2
plot(owref(1:thetop)/(2*pi),...

```

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```

log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),r',...
owref(1:thetop)/(2*pi),...
log10(abs(odcorrha(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),g',...
owref(1:thetop)/(2*pi),...
log10(abs(odcorrha2(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),b',...
owref(1:thetop)/(2*pi),...
log10(abs(uncorrha(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),m')
ylabel(['H','int2str(aset(which_coord))','int2str(aset(which_coord)),' in/lbf (log10 of)'])
hold on
v=axis;
xlabel('Omega (Hz)')
if titles
    if use_antires == 'on'
        title(['int2str(odivisions),' PT INTEGRAL SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT'])
    else
        title(['int2str(odivisions),' PT INTEGRAL SOLTNS WEIGHTED LEFT TO RIGHT'])
    end
end
plot(omegax(1:loopindex)/(2*pi),ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),k+')
%
if use_antires
%
%plot(antiresfreq(which_coord,1:loopindex)/(2*pi),ones(1,loopindex)*(v(%3)+abs(v(4)-v(3)).*.98),k*)
%
grid on
hh=legend('Experimental FRF','Corrected INT Anal FRF','Corrected MAT Anal FRF','Uncorrected Anal FRF','Included
75 Modes');
axes(hh)
hold off
print -dcdjcolor
print -dmfile fig6_5
80 elseif loopindex == 3
if odivisions == 1
    print -dcdjcolor
    print -dmfile fig6_7
    h=figure(h+1);
    hold off
    load od3
    plot(owref(1:thetop)/(2*pi),...
90 log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),r',...
owref(1:thetop)/(2*pi),...
log10(abs(odcorrha(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),g',...
owref(1:thetop)/(2*pi),...
log10(abs(odcorrha3(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),b',...
owref(1:thetop)/(2*pi),...
log10(abs(uncorrha(ndx3d([aset_size aset_size length(owref]),...
which_coord,which_coord,1:thetop))),m')
ylabel(['H','int2str(aset(which_coord))','int2str(aset(which_coord)),' in/lbf (log10 of)'])
hold on
v=axis;
xlabel('Omega (Hz)')
if titles
    if use_antires == 'on'
        title(['int2str(odivisions),' PT INTEGRAL SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT'])
    else

```

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```

        title([int2str(odivisions),' PT INTEGRAL SOLTNS WEIGHTED LEFT TO RIGHT])
    end
end
810 plot(omegax(1:loopindex)/(2*pi),ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),'k+')
% if use_antires
% %plot(antiresfreq(which_coord,1:loopindex)/(2*pi),ones(1,loopindex)*(v(%3)+abs(v(4)-v(3)).*.98),'k*')
% end
grid on
815 hh=legend('Experimental FRF','Corrected INT Anal FRF','Uncorrected Anal FRF','Included Modes');
axes(hh)
hold off
print -dcdjcolor
print -dmfile fig6_8
820
end
elseif loopindex == 4
    print -dcdjcolor
    print -dmfile fig6_4
825 end
% if pswitch=='y'
% prtf1(fignum)
% delete(h)
% else
830 % delete(h)
% end
end
delete(h)
clear H_T_EXP
835 clear odcorrrha
clear uncorrha
end

load H_T_EXP
840 load od1
load od2
load od3
load od4
load uncorrha
845 %load lod5
    h=figure(h+1);
    plot(owref/(2*pi),...
        log10(abs(H_T_EXP(ndx3d([aset_size aset_size length(owref]),...
850 which_coord,which_coord,'!')))),r',...
        owref/(2*pi),...
        log10(abs(odcorrrha1(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!')))),g',...
        owref/(2*pi),...
        log10(abs(odcorrrha2(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!')))),b',...
        owref/(2*pi),...
        log10(abs(odcorrrha3(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!')))),m',...
        owref/(2*pi),...
        log10(abs(odcorrrha4(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!')))),c',...
        owref/(2*pi),...
        log10(abs(uncorrha(ndx3d([aset_size aset_size length(owref]),...
        which_coord,which_coord,'!')))),k',...
865 ylabel(['H',int2str(aset(which_coord)),...
        ',int2str(aset(which_coord)),') in/lbf (log10 of)'])

```

CHAP6.M

```
hold on  
v=axis;  
% xlabel('Omega (Hz)')  
if titles  
    if use_antires == 'on'  
        title([int2str(odivisions),' PT INTEGRAL SOLTNS WITH ANTIRESONANCES WEIGHTED LEFT TO RIGHT'])  
    else  
        title([int2str(odivisions),' PT INTEGRAL SOLTNS WEIGHTED LEFT TO RIGHT'])  
    end  
end  
% plot(omegax(1:loopindex)/(2*pi),...  
% ones(1,loopindex)*(v(3)+abs(v(4)-v(3)).*.98),k+)  
% if use_antires  
% %plot(antiresfreq(which_coord,1:loopindex)/(2*pi),ones(1,loopindex)*(v(%3)+abs(v(4)-v(3)).*.98),k*)  
% end  
v=axis;  
axis([10 300 v(3) v(4)]);  
grid on  
385 hh=legend('Experimental FRF','1 mode MAT solution','2 mode MAT solution','3 mode MAT solution','4 mode MAT  
solution','Uncorrected Anal FRF');  
axes(hh)  
hold off  
%if pswitch=='y'  
% prtf1l(fignum)  
% delete(h)  
%else  
% delete(h)  
%end  
395  
  
print -dcnjcolor  
print -dmfile fig6_12  
00
```

SETUP.M

```

%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
% PROGRAM: SETUP.M %%%%%%
% INITIALIZES DATA FILE FOR A FINITE ELEMENT ANALYSIS %%%%%%
% VARIABLES SAVED TO FILE SETUP.MAT %%%%%%
%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%% %%%%%%
5      clc
10     clear
15     L=input('Enter the total length of the beam: ');
20     numel=input('Num of elements: ');
25     dof_node=input('num dof per node: ');
30     area=input('area of beam: ');
35     eei=inp
40     %ee=inp
45     pho=input('mass density for beam: ');
50     numdof=dof_node*(numel+1)
55     clc
60     while 1
       conforce=input('Enter # of concentrated loads: ');
       if ~isnan(conforce)
         break
       end
       end
       for i=1:conforce
         forcepos(i)=input(['Beam position of force ',num2str(i),': ']);
         forcesiz(i)=input(['Magnitude of force ',num2str(i),': ']);
       end
       force=zeros(numdof,1);
       for i=1:conforce
         temppos=ceil(forcepos(i)/(L/numel));
         force(dof_node*temppos+1)=force(dof_node*temppos+1)+.5*forcesiz(i);
         force(dof_node*temppos+3)=force(dof_node*temppos+3)+.5*forcesiz(i);
       end
       end
       while 1
         lmass=input('Enter # of Lumped masses: ');
         if ~isnan(lmass)
           break
         end
         end
         for i=1:lmass
           masspos(i)=input(['Beam position of Mass ',num2str(i),': ']);
           masssiz(i)=input(['Magnitude of Mass ',num2str(i),': ']);
         end
         lumpmass=zeros(numel,1);
         for i=1:lmass
           temppos=round(masspos(i)/(L/numel));
           lumpmass(temppos)=lumpmass(temppos)+masssiz(i);
         end
         lumpmass'
         while 1
           lspring=input('Enter # of Lumped Springs: ');
           if ~isnan(lspring)
             break
           end
           end
           for i=1:lspring
             springpos(i)=input(['Beam position of Spring ',num2str(i),': ']);
           end
         end
       end
     end
   end
 
```

SETUP.M

```

springsiz(i)=input(['Spring constant of Spring ',num2str(i),' : ']);
end
lumpspring=zeros(numel,1);
for i=1:lspring
65    temppos=round(springpos(i)/(L/numel));
    lumpspring(temppos)=lumpspring(temppos)+springsiz(i);
end
lumpspring'
conn=[1,2];
70    for i=2:numel
        conn=[conn;i,i+1];
    end
    conn
while 1
75    bc=['pinned-pinned
          'clamp-clamp
          'left guided clamp'
          'right guided clamp'
          'cantilevered
          'free-free      '];
    clc
    help bctext
    n=input('Select a boundary condition: ');
    if ((n > 0) & (n < 7))
        break
    end
    end
if n == 1
    bc='pp';
90 elseif n == 2
    bc='cc';
elseif n == 3
    bc='lc';
elseif n == 4
    bc='rc';
elseif n==5
    bc='cl';
else
    bc='ff';
00 end

clear i
clear temppos
clear conforce
05 clear forcepos
clear forcesiz
clear lmass
clear masspos
clear masssiz
0 clear lspring
clear springpos
clear springsiz
save setup.mat
x=0:L/numel:L;
0 for i=1:length(x)
5    h(i)=5;
    end
    clg
    hold off
    %axis('off')
0

```

SETUP.M

```
%plot([],[])
%hold on
plot(x,h,x,h,'x')
```

125

BEAMMDL.M

```

%%%%%FORM FE BEAM MODEL WITH ERRORS AT PRESCRIBED LOCATIONS%%%%%
%%%%%MASS ERROR LOCATION SPECIFIED VIA POSOFMASSER. VALUE OF ERROR IN%%%%%
5 %PERCENT GIVEN BY VALOFLMASSERR, ETC.%%%%%
%%%%%clear;clc;clf
casename='Spatially Incomplete A set Stiffness error';
10 if exist('setup.mat')
    load setup      %load existing input data file
else
    setup          %create new input data file
end
15 struc_damping=0.00000000001;
lumpdamp=lumpspring*0      ;
posofMasserr=[3 4]      ;
valueofMasserr=[0.25 0.25]  ;
posofStifferr=[3 4];
20 valueofStifferr=[0.25 0.25]  ;
posoFDamperr=[3 4];
valueofDamperr=[0.25 0.25]  ;
for i=1:length(posofMasserr)
    lumpmass(posofMasserr(i))=valueofMasserr(i);
25 end
for i=1:length(posofStifferr)
    lumpspring(posofStifferr(i))=valueofStifferr(i);
end
for i=1:length(posoFDamperr)
    lumpdamp(posoFDamperr(i))=valueofDamperr(i);
30 end
elen=L/numel;
if bc=='pp'
    doftokill=2;
35 elseif bc=='cc'
    doftokill=4;
elseif bc=='ff'
    doftokill=0;
elseif bc=='cl'
    doftokill=2;
40 end

%%%%%BUILD THE ELEMENTAL MASS AND STIFFNESS MATRICES%%%%%
45 %FOR A 2DOF/NODE "F.E." STRUCTURES.%%%%%
%%%%%ELEMENT STIFFNESS MATRIX%%%%%
50 ke=[12 6*elen -12 6*elen;
60     6*elen 4*(elen^2) -6*elen 2*(elen^2);
     -12 -6*elen 12 -6*elen;
     6*elen 2*(elen^2) -6*elen 4*(elen^2)];
55 ke=(eeii/(elen^3)).*ke;
g=386;
elemke=eeii/(elen^3);

%ELEMENT MASS MATRIX%%%%%
60 me=[156 22*elen 54 -13*elen;
     22*elen 4*(elen^2) 13*elen -3*elen^2;
     54 -13*elen 156 22*elen];
0

```

BEAMMDL.M

```

54    13*elen    156    -22*elen;
-13*elen -3*(elen^2)   -22*elen 4*(elen^2)];
```

65 me = (((pho*area)*elen/g)/(420)).*me;

elemme=(((pho*area)*elen/g)/(420));

70 %%%%%% GOBAL STIFFNESS AND MASS MATRIX FOR A 2 DOF F.E. STRUCTURE%%%%%
%BASED ON THE ELEMENTAL MATRIXES KE AND ME. THE STRUCTURE CONSISTS OF%%%%%
%NUMEL ELEMENTS WITH ELEMENT CONNECTIVITY GIVEN BY MATRIX CONN%%%%%
%%%
%%%
%%%
goblk=zeros(numdof);
75 goblm=zeros(numdof);
goblc=zeros(numdof);
for i=1:numel
 v=conn(i,1);
 w=conn(i,2);
80 goblk(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)=...
 goblk(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)+...
 ke(1:dof_node,1:dof_node);
 goblk(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)=...
 goblk(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)+...
85 ke(1:dof_node,dof_node+1:2*dof_node);

 goblk(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)=...
 goblk(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)+...
 ke(dof_node+1:2*dof_node,1:dof_node);
90 goblk(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)=...
 goblk(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)+...
 ke(dof_node+1:2*dof_node,dof_node+1:2*dof_node);

95 goblm(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)=...
 goblm(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)+...
 me(1:dof_node,1:dof_node);
 goblm(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)=...
 goblm(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)+...
 me(1:dof_node,dof_node+1:2*dof_node);
100 goblm(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)=...
 goblm(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)+...
 me(dof_node+1:2*dof_node,1:dof_node);
 goblm(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)=...
 goblm(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)+...
 me(dof_node+1:2*dof_node,dof_node+1:2*dof_node);
105 end
 goblc=sqrt(-1)*struc_damping.*goblk;
 goblkx=goblk;
 goblcx=goblc;
 goblmx=goblm;

110

115 for i=1:numel
 v=conn(i,1);
 w=conn(i,2);
 goblkx(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)=...
 goblkx(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)+...
 lumpspring(i).*ke(1:dof_node,1:dof_node);

120 goblkx(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)=...

BEAMMDL.M

```

goblkx(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)+...
lumpspring(i).*ke(1:dof_node,dof_node+1:2*dof_node);

125 goblkx(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)=...
goblkx(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)+...
lumpspring(i).*ke(dof_node+1:2*dof_node,1:dof_node);

130 goblkx(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)=...
goblkx(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)+...
lumpspring(i).*ke(dof_node+1:2*dof_node,dof_node+1:2*dof_node);

135 goblmx(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)=...
goblmx(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)+...
lumpmass(i).*me(1:dof_node,1:dof_node);

140 goblmx(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)=...
goblmx(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)+...
lumpmass(i).*me(dof_node+1:2*dof_node,1:dof_node);

145 goblmx(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)=...
goblmx(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)+...
lumpmass(i).*me(dof_node+1:2*dof_node,dof_node+1:2*dof_node);

50 goblcx(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)=...
goblcx(dof_node*v-1:dof_node*v,dof_node*v-1:dof_node*v)+...
lumpdamp(i)*sqrt(-1)*struc_damping.*ke(1:dof_node,1:dof_node);

55 goblcx(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)=...
goblcx(dof_node*v-1:dof_node*v,dof_node*w-1:dof_node*w)+...
lumpdamp(i)*sqrt(-1)*struc_damping.*ke(1:dof_node,dof_node+1:2*dof_node);

60 goblcx(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)=...
goblcx(dof_node*w-1:dof_node*w,dof_node*v-1:dof_node*v)+...
lumpdamp(i)*sqrt(-1)*struc_damping.*ke(dof_node+1:2*dof_node,1:dof_node);

65 goblcx(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)=...
goblcx(dof_node*w-1:dof_node*w,dof_node*w-1:dof_node*w)+...
lumpdamp(i)*sqrt(-1)*struc_damping.*ke(dof_node+1:2*dof_node,dof_node+1:2*dof_node);
end

numdof=numdof-doftokill;

70 [gk,gm,gc,k_anal,m_anal,c_anal]=fixbcs(goblk,goblm,goblc,bc);

[gkx,gmx,gcx,k_exp,m_exp,c_exp]=fixbcs(goblkx,goblmx,goblcx,bc);
save beamdata

```

FSTATIC.M

```
function [kstat,mstat]=fstatic(k,m,oset,aset)
    aset_size=length(aset);
    kaa=k(aset,aset);
    kao=k(aset,oset);
    5   koo=k(oset,oset);
    koa=koa';
    clear k;
    k=[kaa,kao;koz,koo];

10   maa=m(aset,aset);
    mao=m(aset,oset);
    moo=m(oset,oset);
    moa=moz';
    clear m;
15   m=[maa,mao;moa,moo];

20   t_static=-koz*koz;
    T_static=[eye(aset_size); t_static];
    kstat=T_static*k*T_static;
    mstat=T_static*m*T_static;

25   end
```

FIRS_TAM.M

```

%
function [kirs,mirs]=firs_tam(k,m,oset,aset)
%
% this function returns the IRS reduced stiffness
5 % and mass matrices, given the unreduced counterparts.
% Care must be taken that the aset and oset vectors correspond
% with the existing arrangement of k and m.
% k and m are UNPARTITIONED matrices.
%
10 aset_size=length(aset);
%
kaa=k(aset,aset);
kao=k(aset,oset);
koo=k(oset,oset);
15 koa=kao';
clear k;
k=[koo,koa;kao,kaa];
%
maa=m(aset,aset);
20 mao=m(aset,oset);
moo=m(oset,oset);
moa=mao';
clear m;
m=[moo,moa;moa,maa];
25 %
t_static=-koo\koao;
T_static = [t_static; eye(aset_size)];
%
30 kstat=T_static*k*T_static;
mstat=T_static*m*T_static;
%
35 tirs=t_static+inv(koo)*(moa+moo*t_static)*inv(mstat)*kstat;
T_irs=[tirs;eye(aset_size)];
%
kirs=T_irs'*k*T_irs;
mirs=T_irs'*m*T_irs;
%
% end function firs_tam

```

FREQMODE.M

```

5      %%this function returns a vector U containing modal frequencies (rad/sec)^2 in ascending order along with associated mode shapes
10     function [u1,lambda,index]=freqmode(k,m)
11         [u,lambda]=eig(m\k);
12         [lambda,index]=sort(diag(lambda));
13         ll=zeros(length(k),length(k));
14         for i=1:length(k);
15             ll(i,i)=lambda(i);
16         end
17         lambda=diag(ll);
18         for j = 1:length(k)
19             u1(:,j) = u(:,index(j));
20         end

```

NDX3D.M

```
function [r,c] = ndx3d(siz,i,j,k)
%NDX3D      Index into 3-D matrix packed in a 2-D matrix.
%
%      ELEM = NDX3D([M N P],I,J,K) returns the element position ELEM
%      of the (i,j,k) elements of a M-by-N-by-P matrix which is
%      stored in a (M*N)-by-P matrix. For example, the three M-by-N
%      matrices A1,A2,A3 are packed into a 2-D matrix using
%      A = [A1(:) A2(:) A3(:)];
%
%      If length(I) is m, LENGTH(J) is n, and LENGTH(K) is p,
%      then ELEM will be an m-by-n-by-p matrix.
%
%
%      [R,C] = NDX3D([M N P],I,J,K) returns the row and column
%      position of the (i,j,k) element as stored in the normal
%      2-D matrix of size (M*N)-by-P.
%
%
%      To specify all the elements along one dimension use ':".
%      For instance, NDX3D([3 5 4],2:3,:,:,3:4) returns the
%      elements for the 2-by-5-by-2 matrix.
%
%
%      See also ELEM3D, MESHGRID, SLICE.
%
%
%      Clay M. Thompson 11-3-92
%      Copyright (c) 1992 by The MathWorks, Inc.
%      $Revision: 1.7 $ $Date: 1993/09/03 14:36:52 $
```

if isstr(i), i = 1:siz(1); end
if isstr(j), j = 1:siz(2); end
if isstr(k), k = 1:siz(3); end

if isempty(i) | isempty(j) | isempty(k), r = []; c = []; return, end

if any((i<0) | (i>siz(1))), error('Index I out of range.'); end
if any((j<0) | (j>siz(2))), error('Index J out of range.'); end
if any((k<0) | (k>siz(3))), error('Index K out of range.'); end

if nargout==2
 [jj,ii] = meshgrid(j,i);
 r = ii(:) + (jj(:)-1)*siz(1);
 c = k(:);

else
 [jj,ii,kk] = meshgrid(j,i,k);
 r = ii + (jj-1)*siz(1) + (kk-1)*prod(siz(1:2));

end

INTSUB.M

```

function [K,M,C]=intsub(Z_I, Z_R, omega, omega1, W, set_size)
waitbar_handle=waitbar(0,'Computing Integrals');
j=sqrt(-1);

5   integ=omega.^2.*abs(W);
rint=real(integ);
iint=imag(integ);
aa=mytrapz(omega1,rint)+mytrapz(omega1,iint)*j;
integ=(1./omega.^2).*abs(W);
10  rint=real(integ);
iint=imag(integ);
bb=mytrapz(omega1,rint)+mytrapz(omega1,iint)*j;
integ=abs(W);
rint=real(integ);
iint=imag(integ);
15  cc=mytrapz(omega1,rint)+mytrapz(omega1,iint)*j;
for i=1:set_size
    waitbar(i/set_size);
    for k=i:set_size
20    M(i,k)=(1/(aa*bb-cc^2))*(bb*mytrapz(omega1,...
        omega.*Z_I(ndx3d([set_size set_size length(omega)],...
        i,k,1:length(omega))).*abs(W))-cc*mytrapz(omega1,...
        1./omega.*Z_I(ndx3d([set_size set_size length(omega)],...
        i,k,1:length(omega))).*abs(W))) ;
25    M(k,i)=M(i,k);
    C(i,k)=(1/cc)*mytrapz(omega1,Z_R(...
        ndx3d([set_size set_size length(omega)],...
        i,k,1:length(omega))).*abs(W));
    C(k,i)=C(i,k);
30    K(i,k)=(1/(aa*bb-cc^2))*(cc*mytrapz(omega1,...
        omega.*Z_I(ndx3d([set_size set_size length(omega)],...
        i,k,1:length(omega))).*abs(W))-aa*mytrapz(omega1,...
        1./omega.*Z_I(ndx3d([set_size set_size length(omega)],...
        i,k,1:length(omega))).*abs(W))) ;
35    K(k,i)=K(i,k);
end
end
close(waitbar_handle)

```

40

MYTRAPZ.M

```
function z = trapz(x,y)
%TRAPZ Trapezoidal numerical integration.
% Z = TRAPZ(X,Y) computes the integral of Y with respect to X using
% trapezoidal integration. X and Y must be vectors of the same length,
% or X must be a column vector and Y a matrix with as many rows as X.
% TRAPZ computes the integral of each column of Y separately.
% The resulting Z is a scalar or a row vector.
%
% Z = TRAPZ(Y) computes the trapezoidal integral of Y assuming unit
% spacing between the data points. To compute the integral for
% spacing different from one, multiply Z by the spacing increment.
%
% See also SUM, CUMSUM.

% Clay M. Thompson, 10/16/90; Cleve Moler, 1/19/92.
% Copyright (c) 1984-94 by The MathWorks, Inc.

% Make sure x and y are column vectors, or y is a matrix.

% Trapezoid sum computed with vector-matrix multiply.

[m,n]=size(x);
z=0;
for index=1:m
    yy=y(1+(index-1)*n:n+(index-1)*n);
    yy=yy(:);
    xx=x(index,:);
    xx = xx(:);
    z=z+diff(xx)'*(yy(1:n-1) + yy(2:n))/2;
end
```

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